

Compressors

Standards and Codes

API STD 617

Centrifugal Compressors for Petroleum, Chemical, and Gas Industry Services

API STD 618

Reciprocating Compressors for Petroleum, Chemical, and Gas Industry Services

API STD 619

Rotary Compressors for Petroleum, Petrochemical, and Gas Industry Services

API STD 681

Liquid Ring Vacuum Compressors for Petroleum, Chemical

Compressors Classification

Kinetic

A – Centrifugal

B – Axial

Positive Displacement

C -Reciprocating

D -Rotary

***Screw**

*** Lobe**

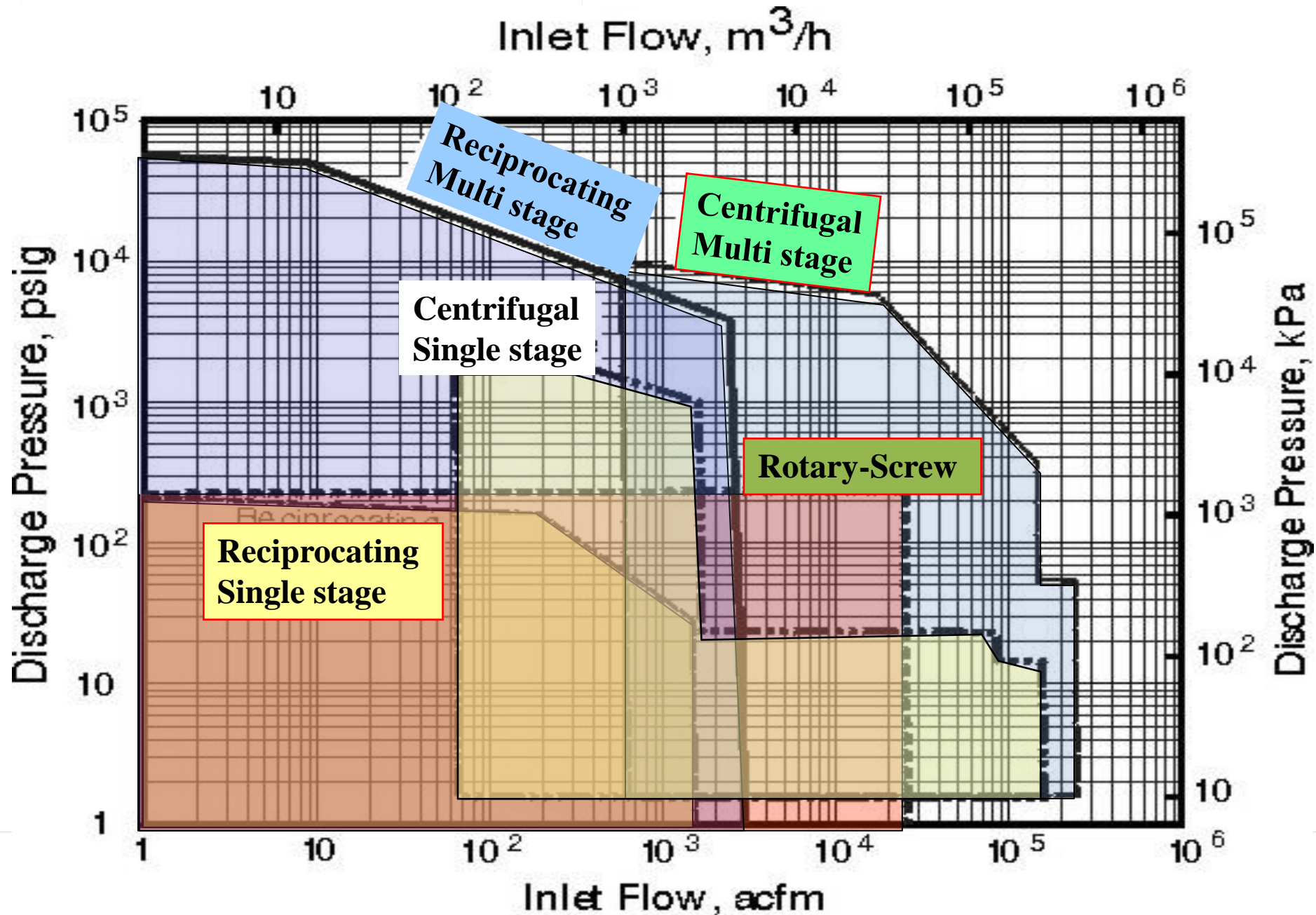
***Sliding Vane**

***Gear**

*** Others**

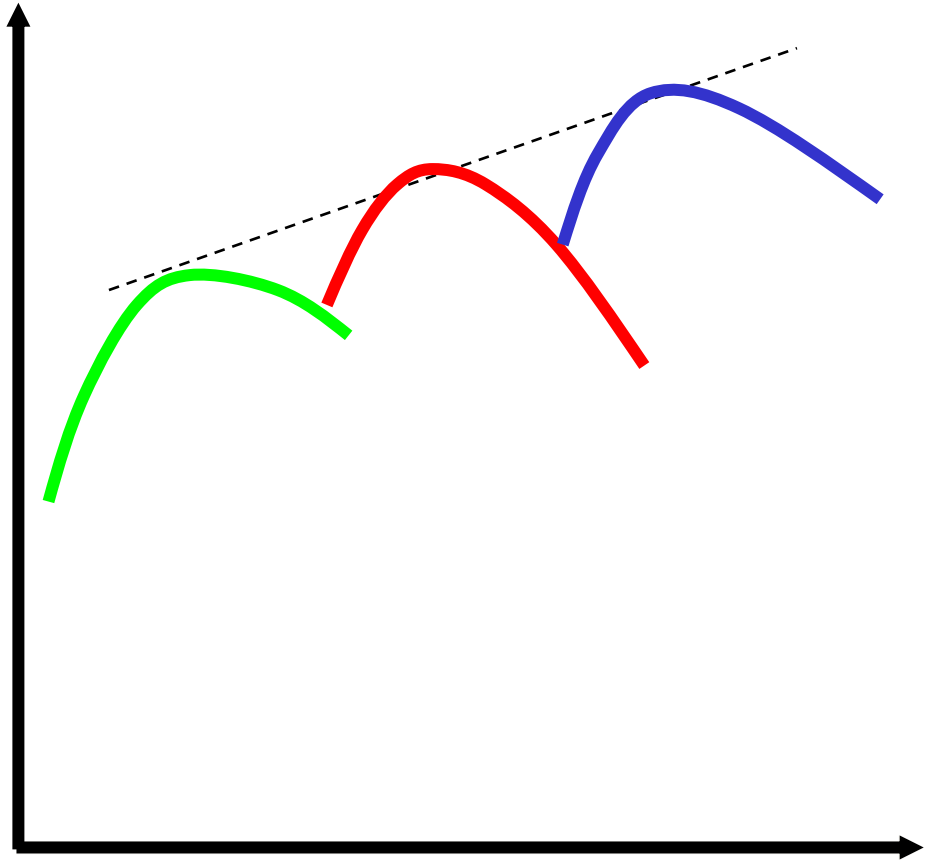
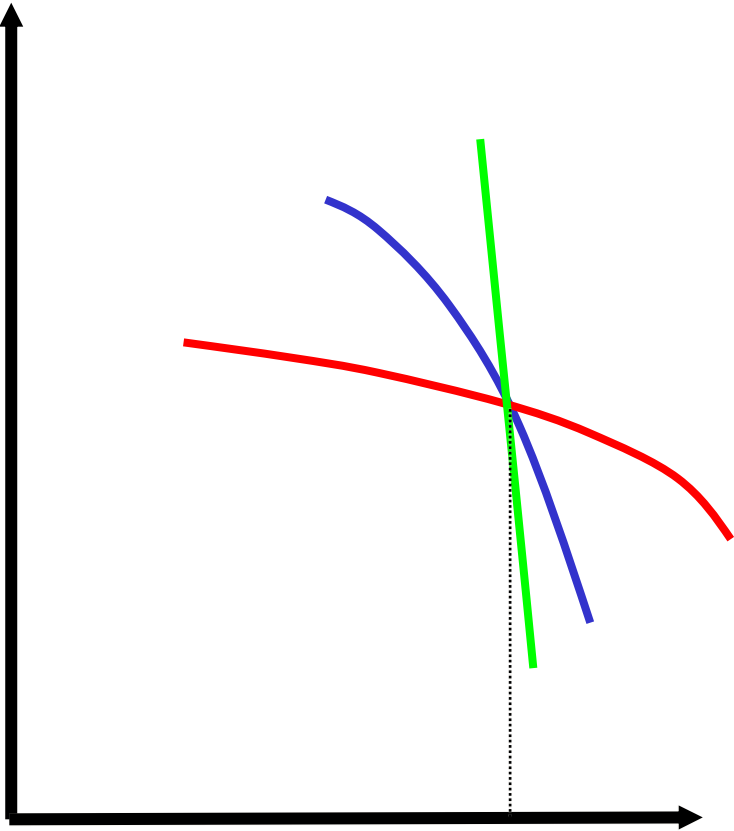
| | Red | Red | Red |
|------|--------------|-------------|-------------|
| Grey | Black | Cyan | Light Green |
| Grey | Light Green | Cyan | Black |
| Grey | Yellow | Red | Red |
| Grey | Bright Green | Cyan | Black |
| Grey | Black | Light Green | Light Green |
| Grey | Yellow | Red | Red |
| Grey | Red | Yellow | Yellow |

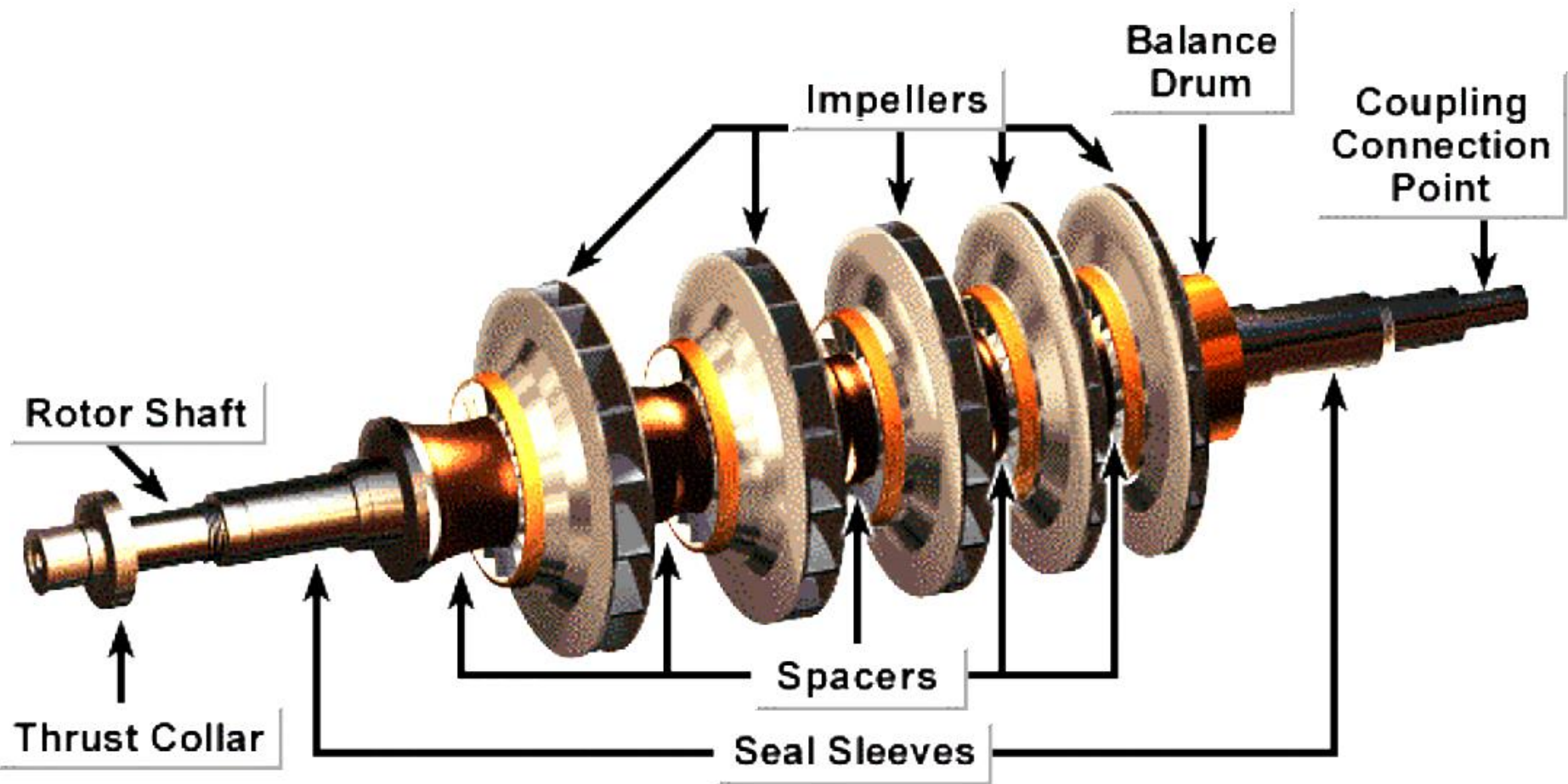
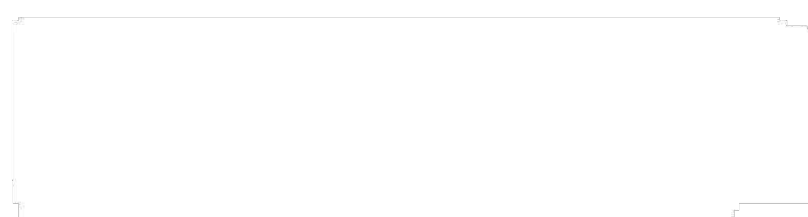
Compressor Selection



Different Types of Compressors

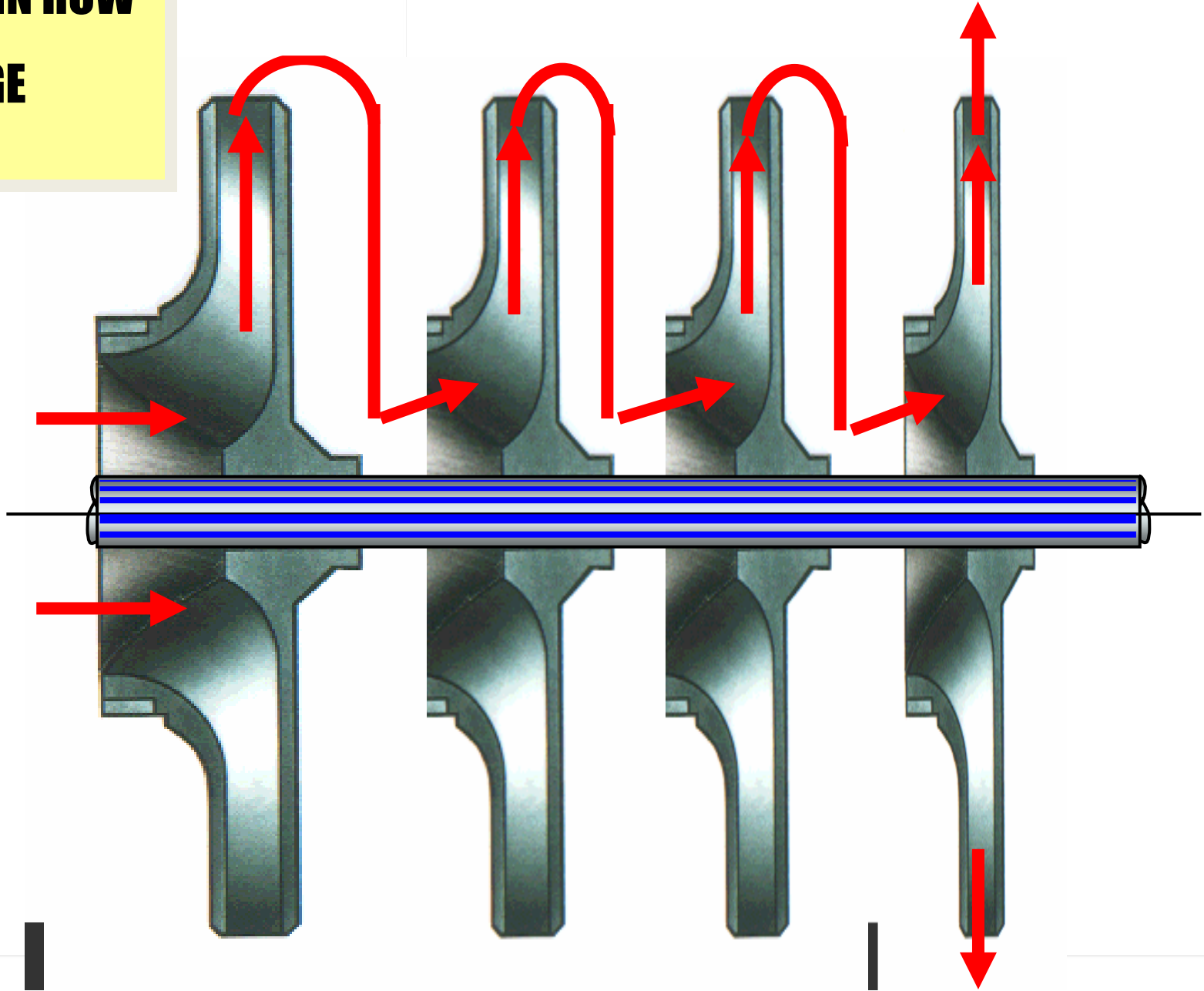
| <i>Type</i> | <i>Advantages</i> | <i>Disadvantages</i> |
|------------------------------|---|---|
| Centrifugal | <ul style="list-style-type: none"> - Wide operating range - Low maintenance - High reliability | <ul style="list-style-type: none"> - Unstable at low flow - Moderate efficiency |
| Axial | <ul style="list-style-type: none"> - High efficiency - High speed capability - Higher flow for a given size | <ul style="list-style-type: none"> - Low pressure ratio per stage - Narrow flow range - Fragile and expensive blading |
| Positive displacement | <ul style="list-style-type: none"> - Pressure ratio capability not affected by gas properties - Good efficiencies at low specific speed | <ul style="list-style-type: none"> - Limited capacity - High weight to capacity ratio - Higher maintenance requirements - Introduces vibrations into the system - Bigger foundation requirements |



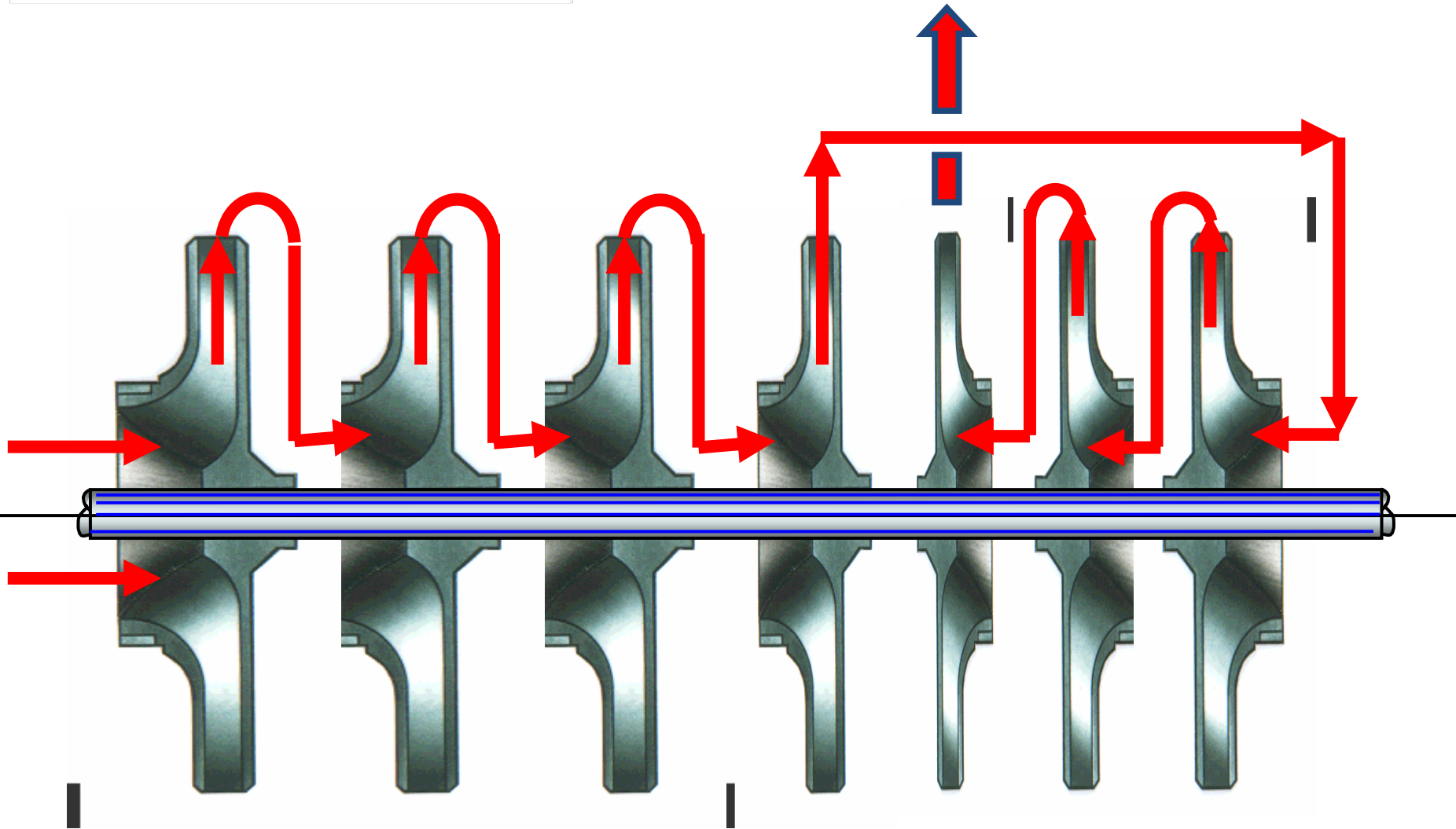


IMPELLERS IN ROW

MULTI-STAGE COMP.

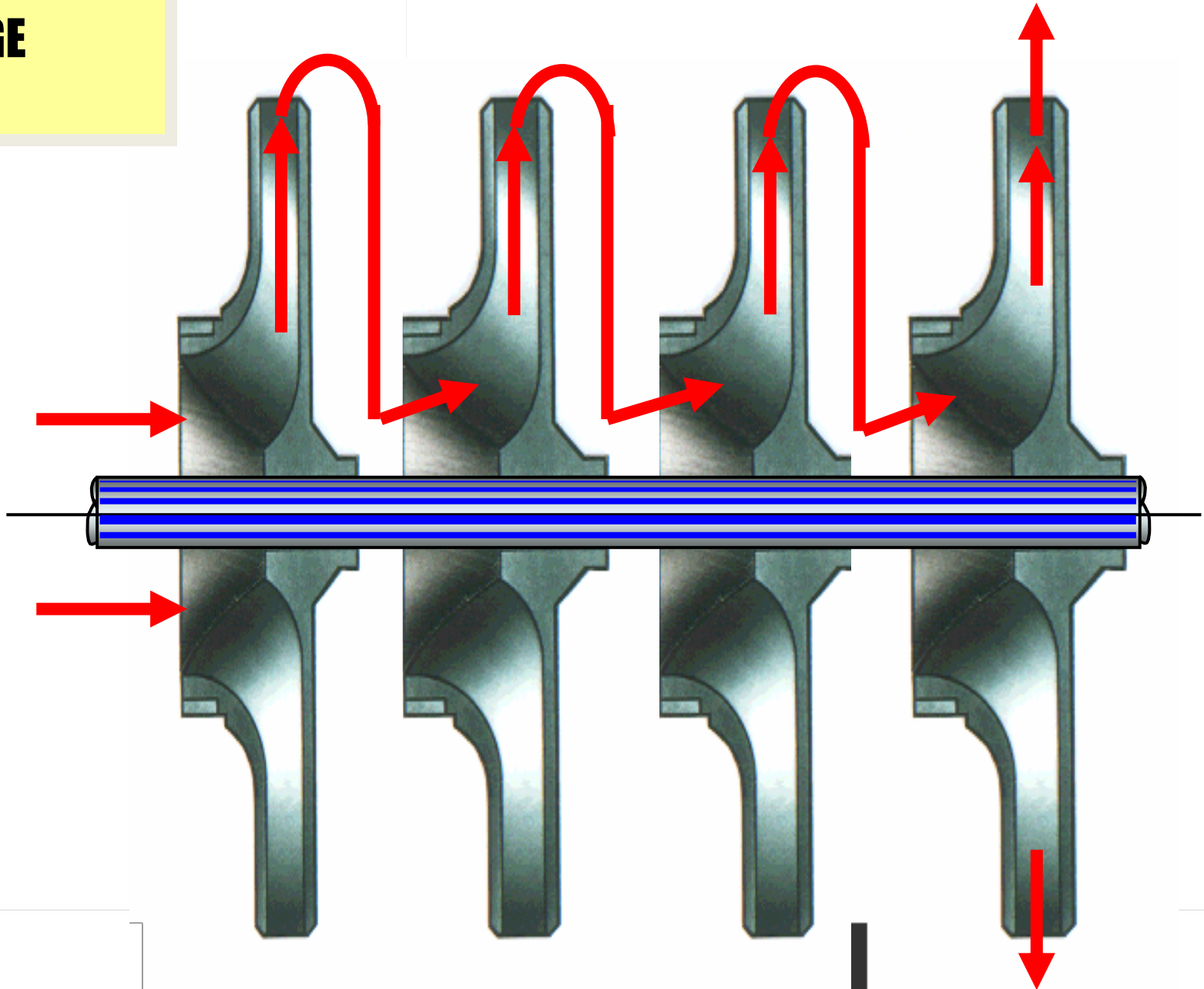


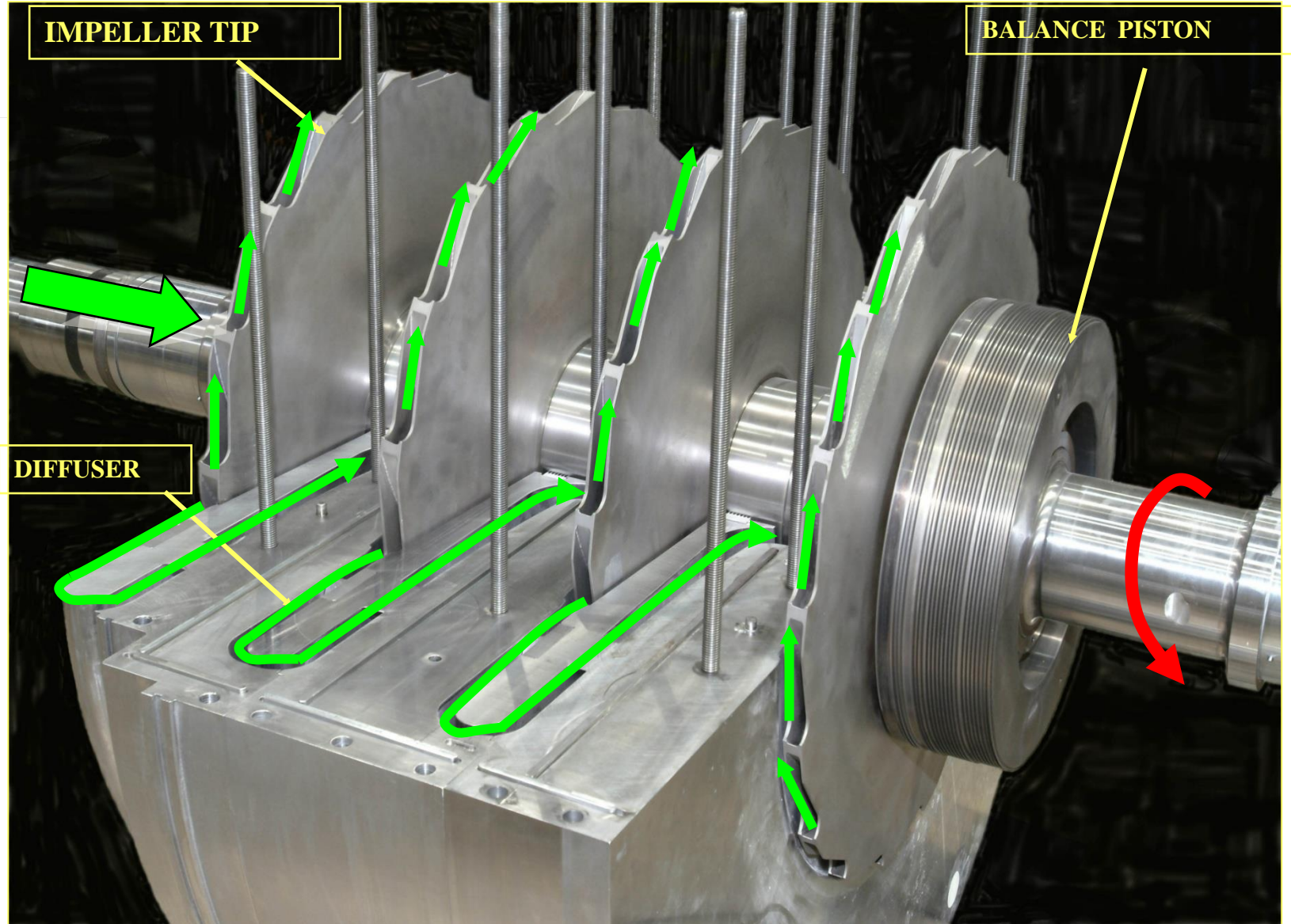
**OPPOSITE IMPELLERS
MULTI-STAGE COMP.**



IMPELLERS IN ROW

MULTI-STAGE PUMP





IMPELLER TIP

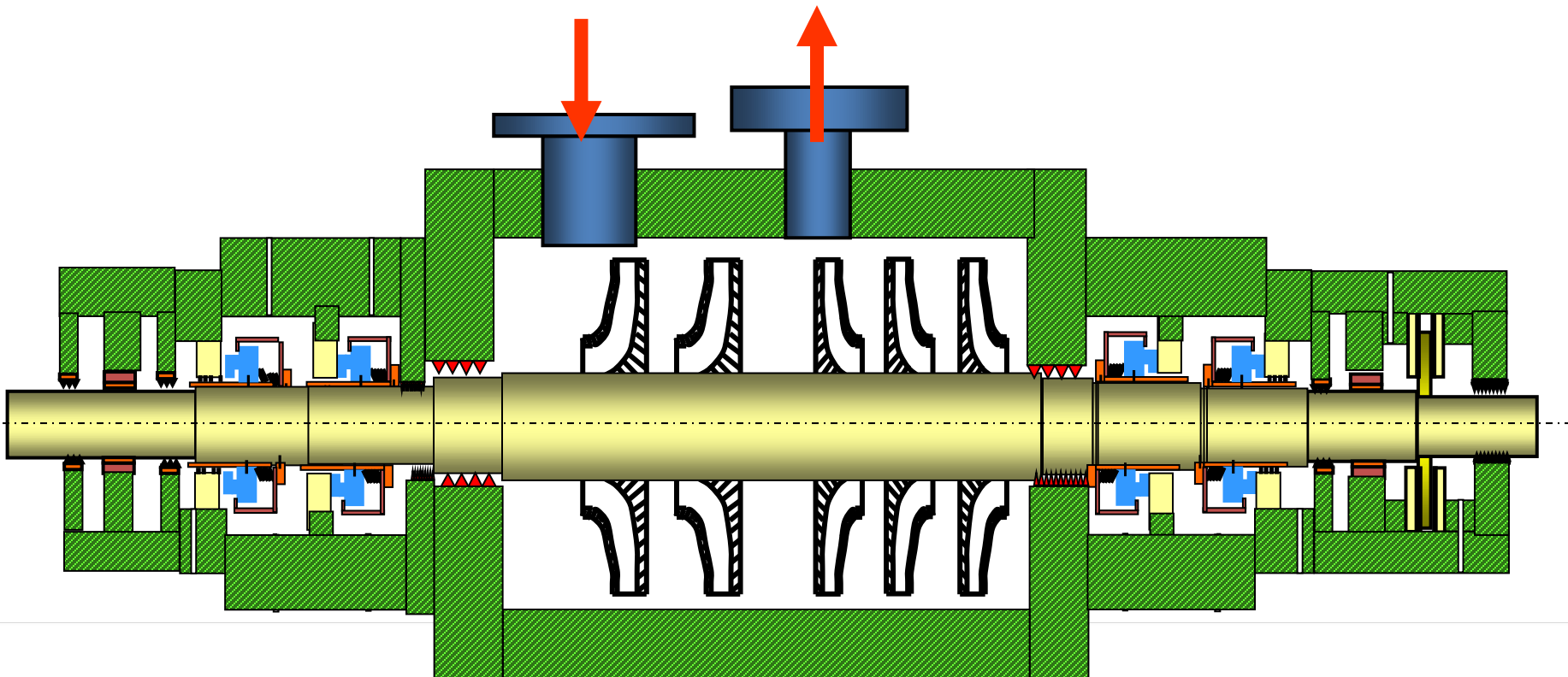
BALANCE PISTON

DIFFUSER

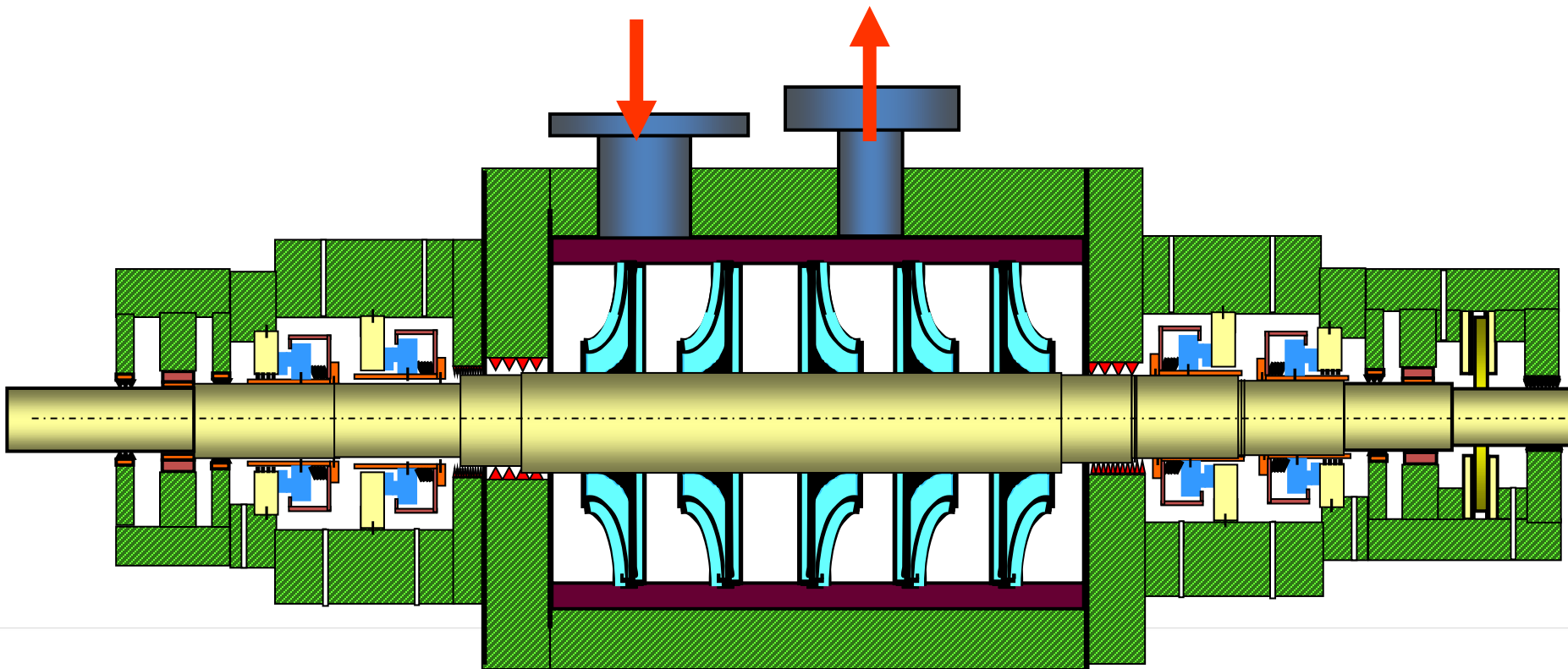
SPLIT TYPE COMPRESSORS

1- Horizontally Split

High Flow Medium pressure

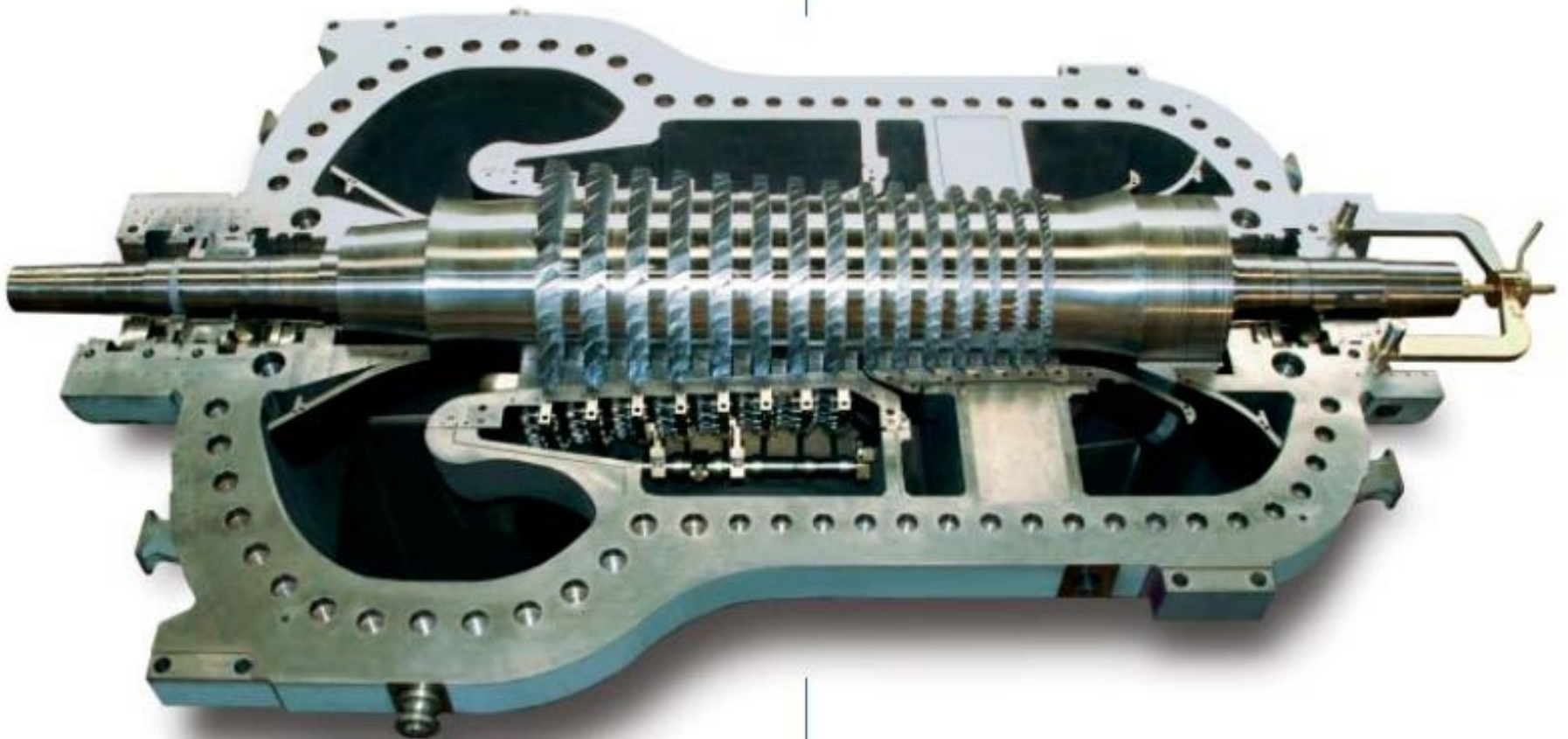


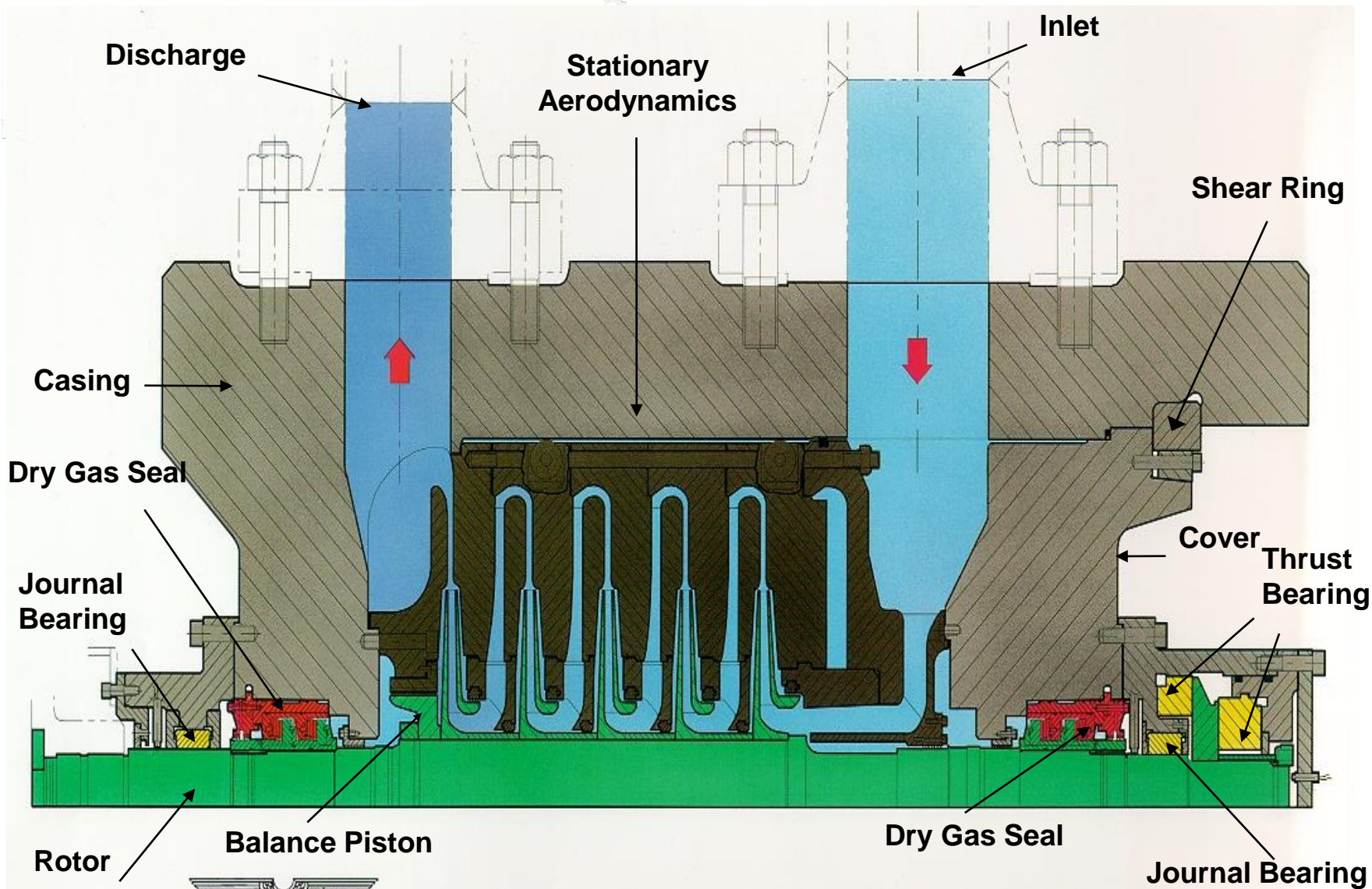
2- Vertically Split (Double Barrel) high pressure and medium Flow



Axial Flow Compressors

- * API 617
- * Very High Flow rate
- * Low pressure
- * Very High efficiency

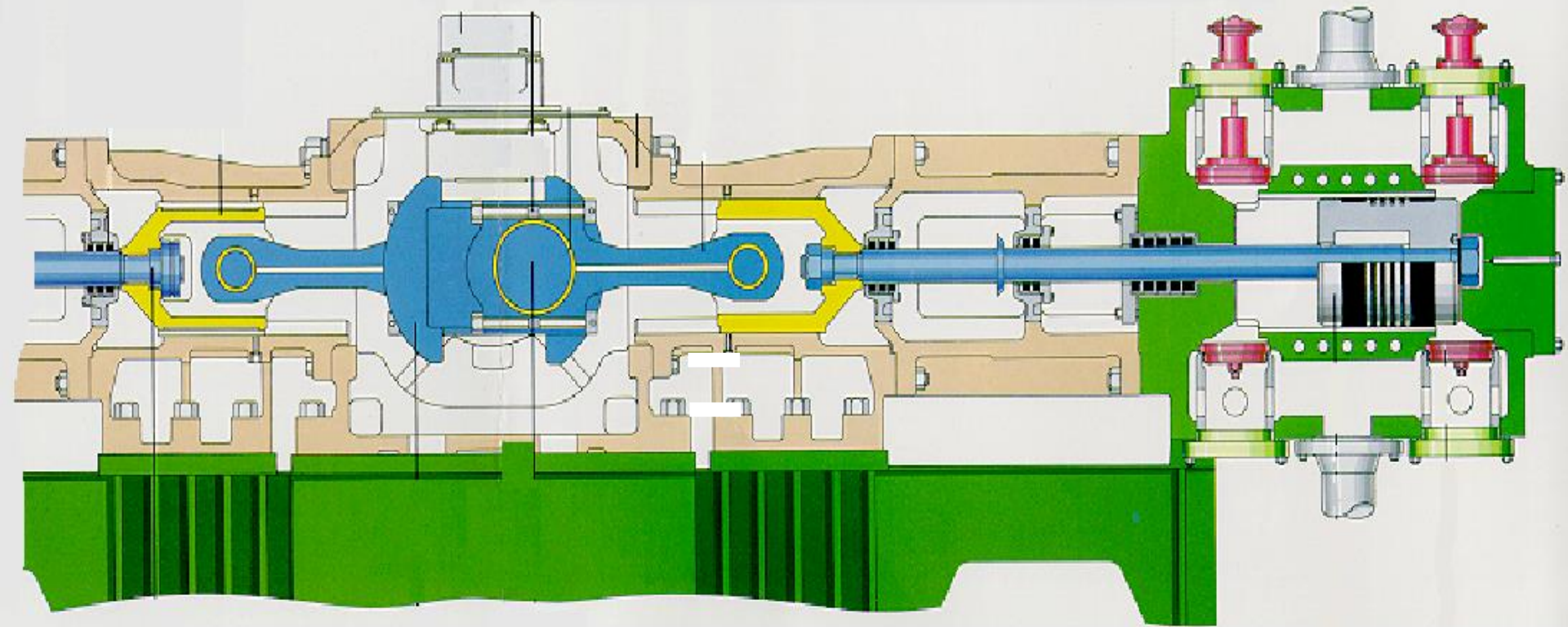




*API 617
 * High Flow rate

* Medium pressure
 * High efficiency

Reciprocating Compressor



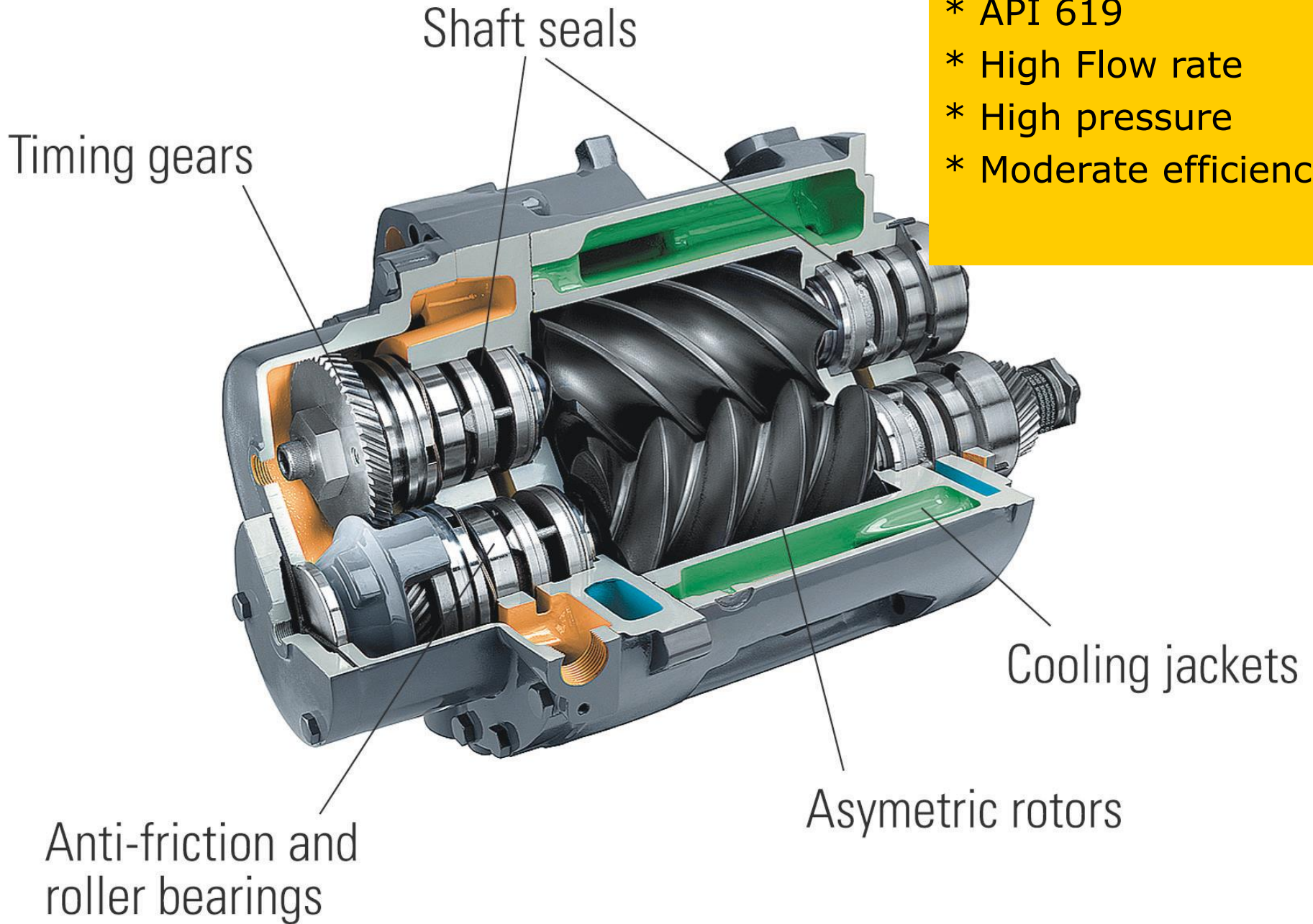
* API 618

* Low Flow rate

* High pressure

* Moderate efficiency

Rotary-Screw Compressor



**** *Compressor Selection***

Application

Operating parameter

footprint/weight

Availability

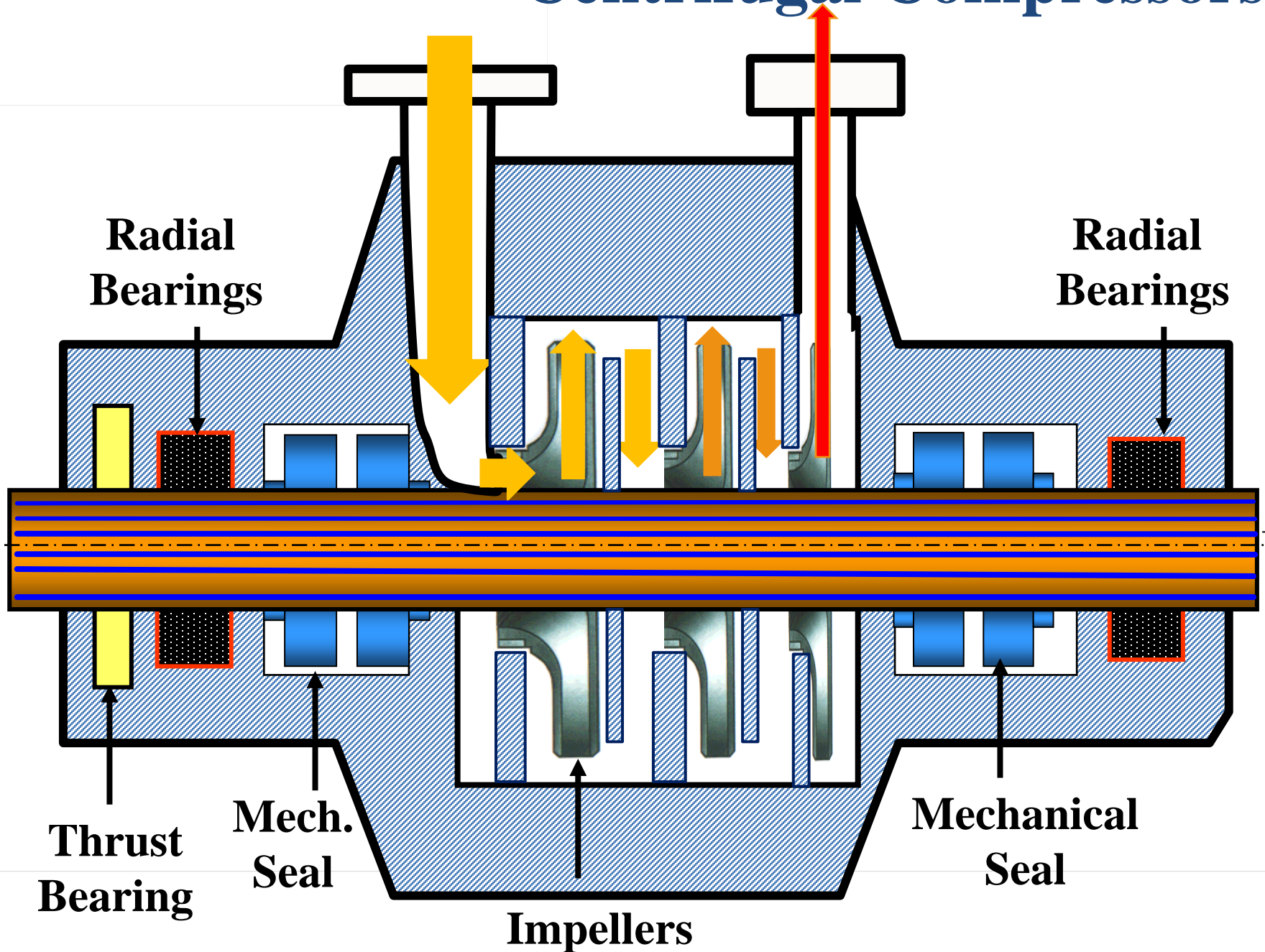
Reliability

maintenance

drivers

Centrifugal Compressor

Centrifugal Compressors



Centrifugal Compressor

** *Compressors components*

Rotor Shaft Impellers Thrust collars Spacers

Diffusers

Casing

Labyrinths

Mechanical Seals

Couplings

HSC

LSC

Gear Box

Anti- Surge System

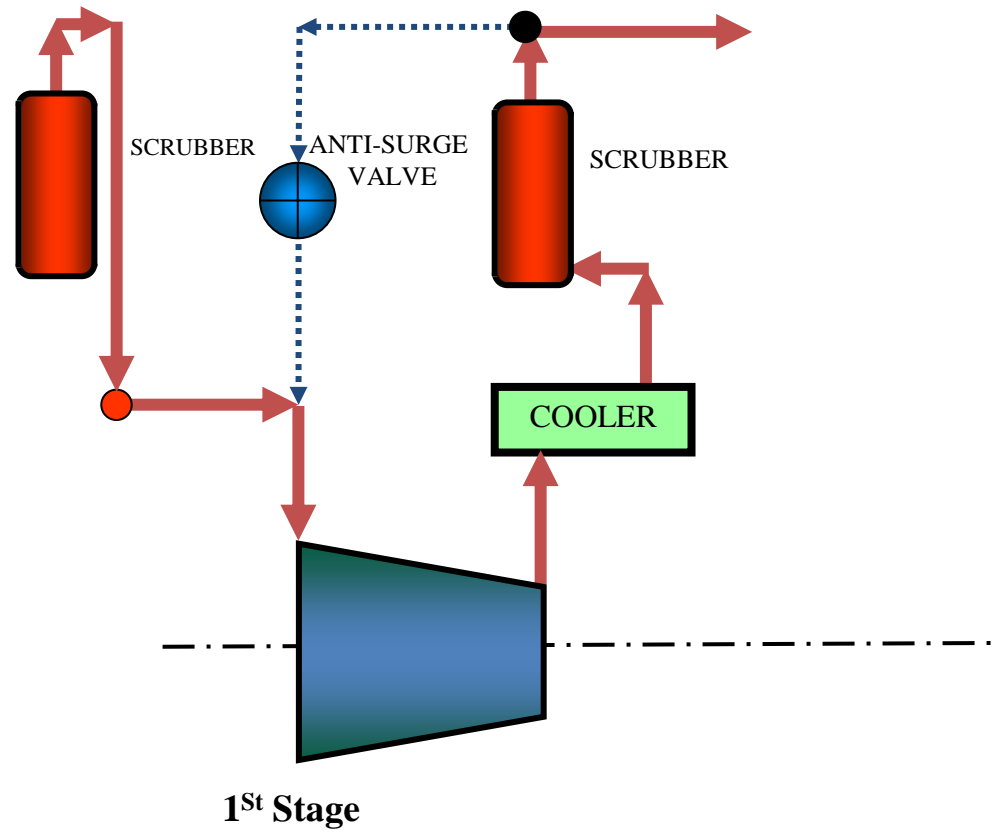
Lube Oil System

Driver

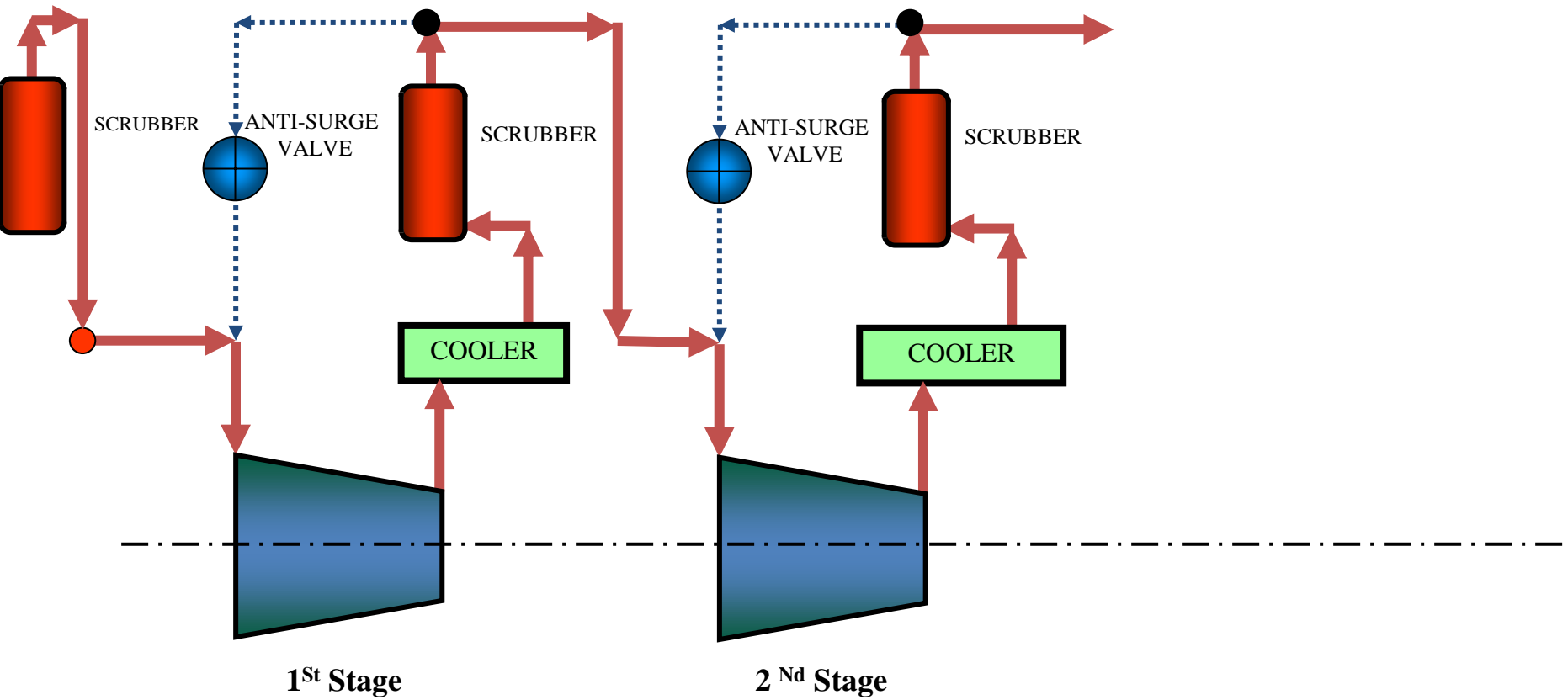
General aspects

- 1- A Complete spare balanced rotor to be ideally stored in ware house**
- 2- Compressor will be equipped with a complete surge control system.**
- 3- It is advisable to use dry gas mechanical seal system instead of wet mechanical seal one.**

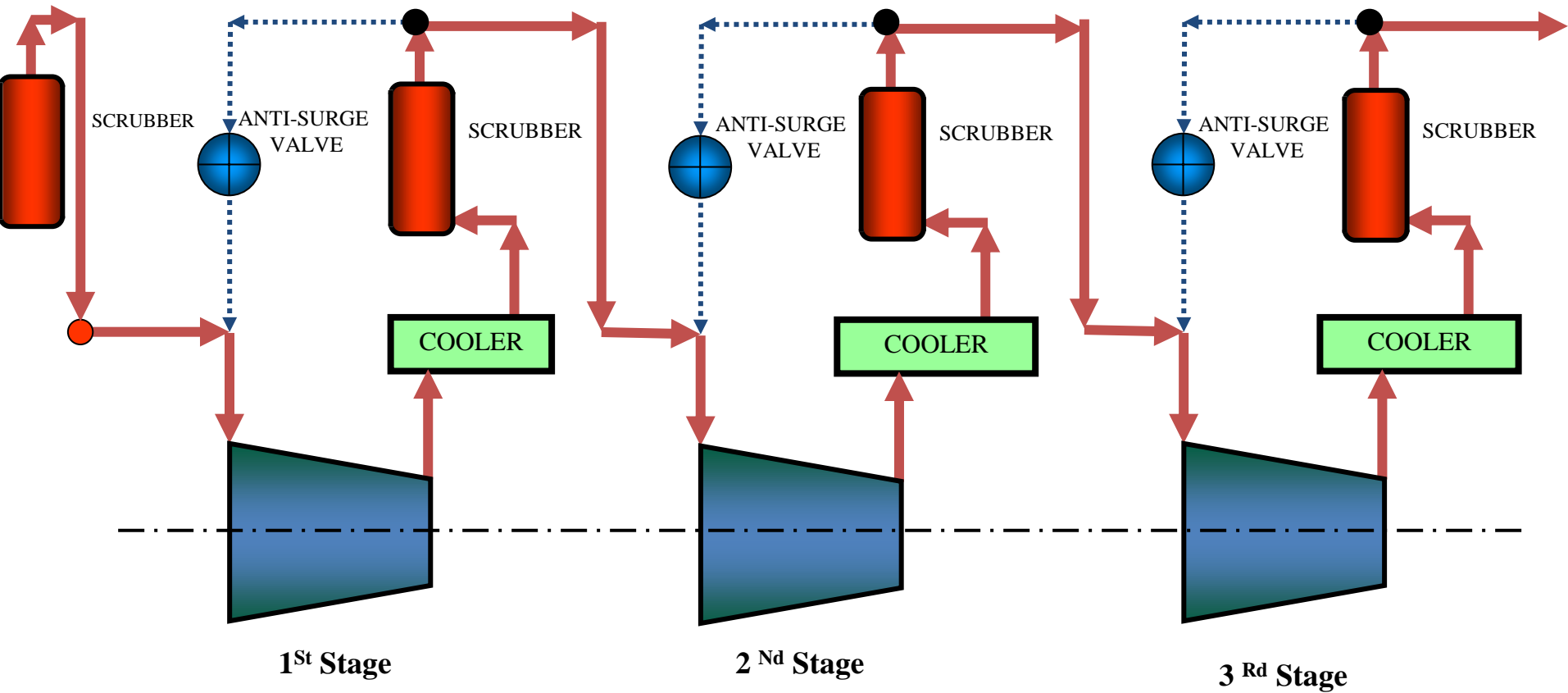
*** Centrifugal Compressor Operation*



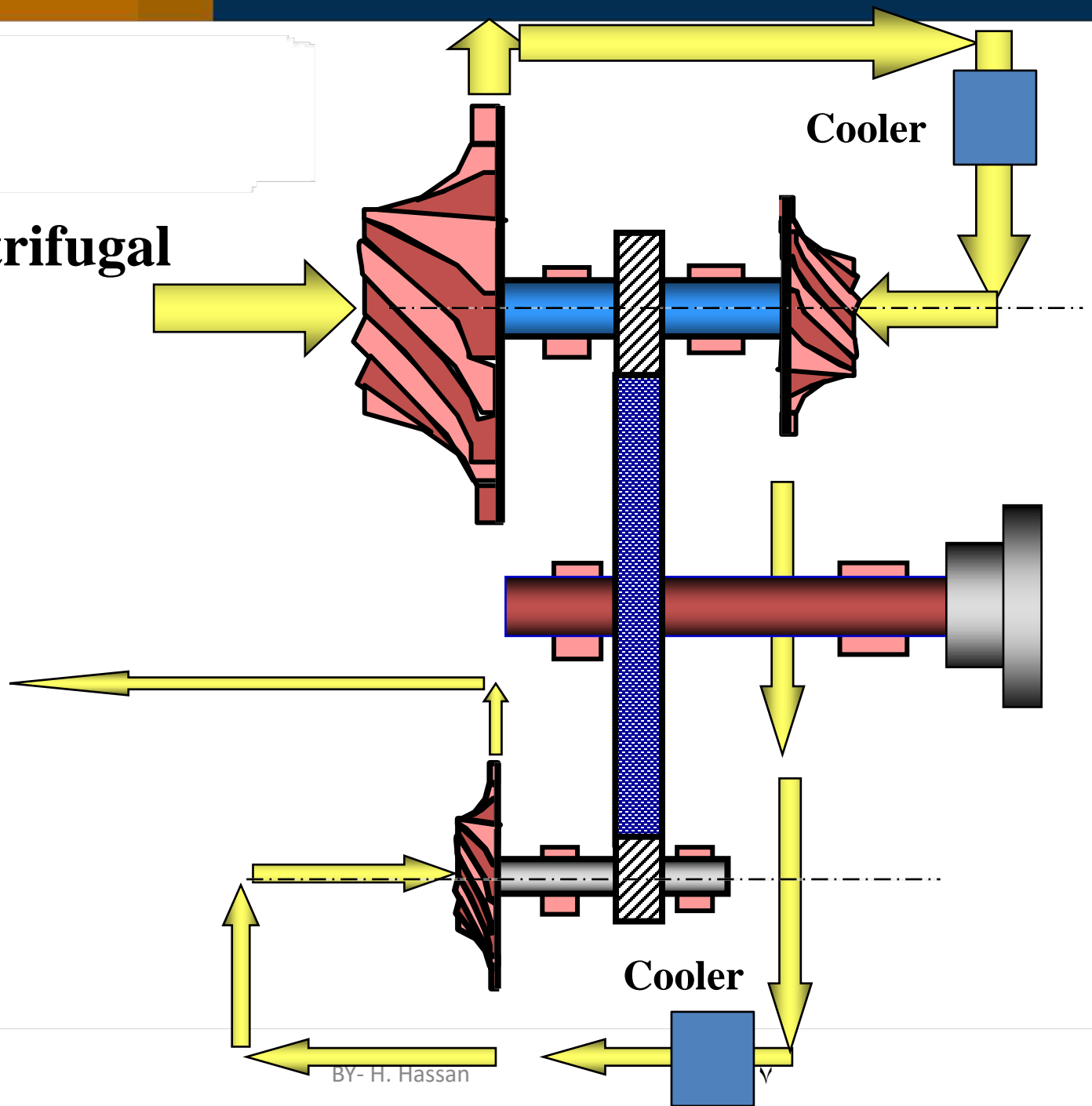
*** Centrifugal Compressor Operation*



*** Centrifugal Compressor Operation*



3 Stages Geared Centrifugal Compressor





Surge Phenomenon



***CENTRIFUGAL
COMPRESSORS***

***HAPPENS
ONLY TO :***

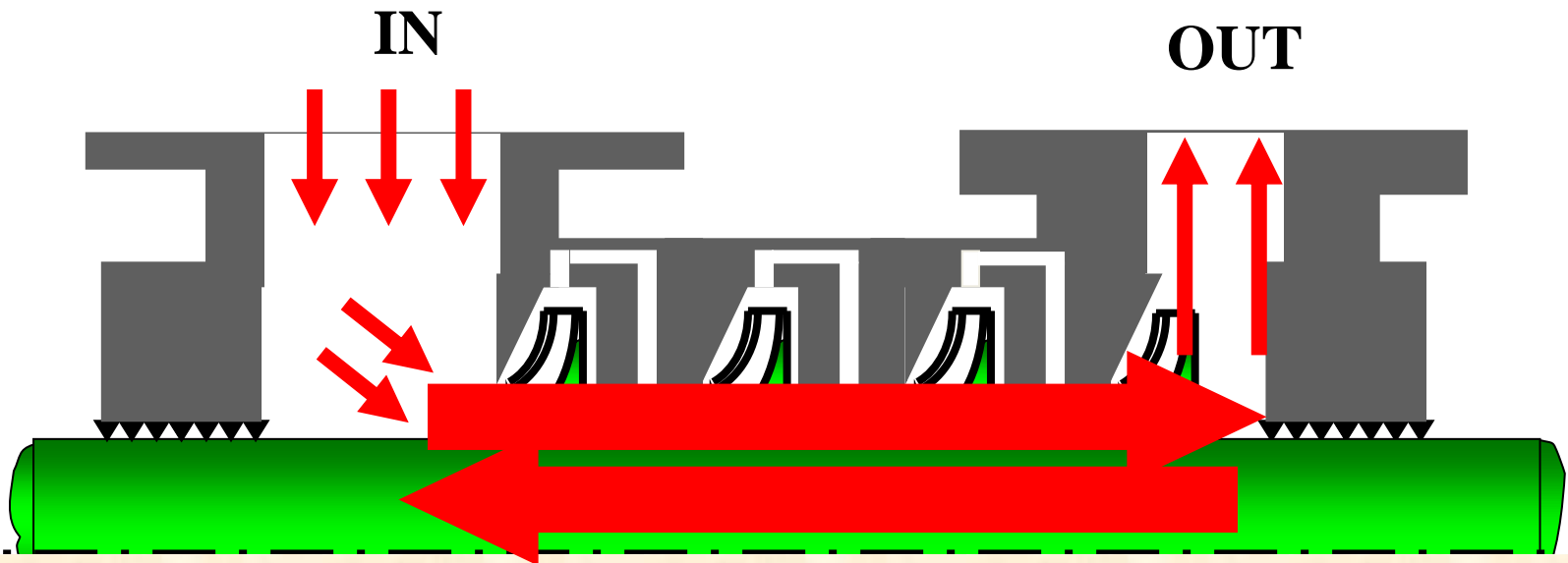
AND

***AXIAL FLOW
COMPRESSORS***

***But Does not Happen to
Reciprocating
Compressors***

SURGE PHENOMEN

It is the flow back of gases from the outlet of the **Last stage** of the compressor towards the suction and return again to discharge



SURGE PHENOMEN

Why

- **FLOW-RATE IS NOT ENOUGH**
- **GAS PROPERTY**
- **COMPRESSOR PERFORMANCE**
- **FLATENESS OF P.C. AT LOW Q**

- **FLOW-RATE
IS NOT ENOUGH**

This will happen at
starting and shutdown,
also at abnormal conditions.

- **GAS PROPERTY**

Gas is compressible but liquid is not.

• **COMPRESSOR PERFORMANCE**

Centrifugal and Axial compressors are pumping gas continuously but reciprocating is not.

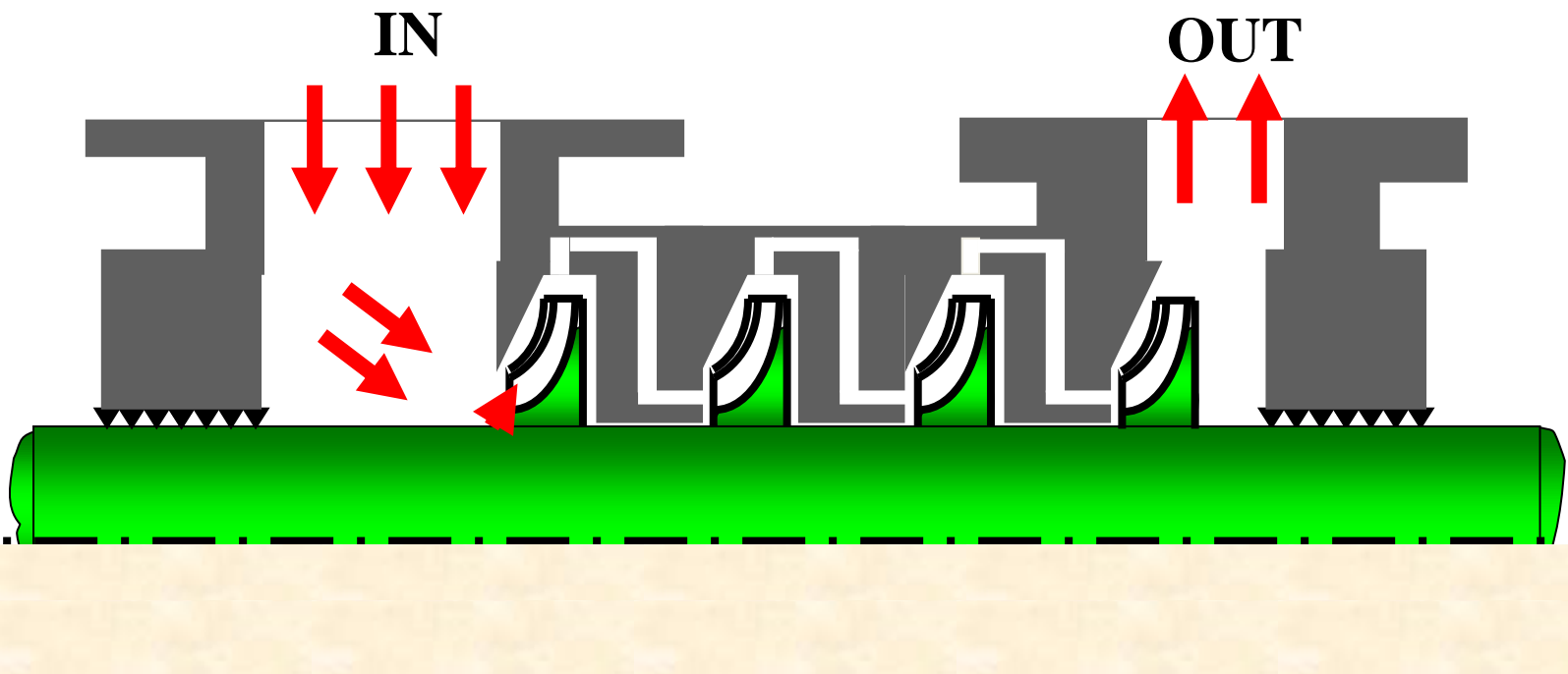
• **FLATENESS OF P.C. AT LOW Q**

Gas pressure has the same energy at horizontal portions of the performance curve

NO SURGE

IF

INLET FLOW RATE Q IS ENOUGH



**IF INLET FLOW RATE Q
IS NOT ENOUGH**

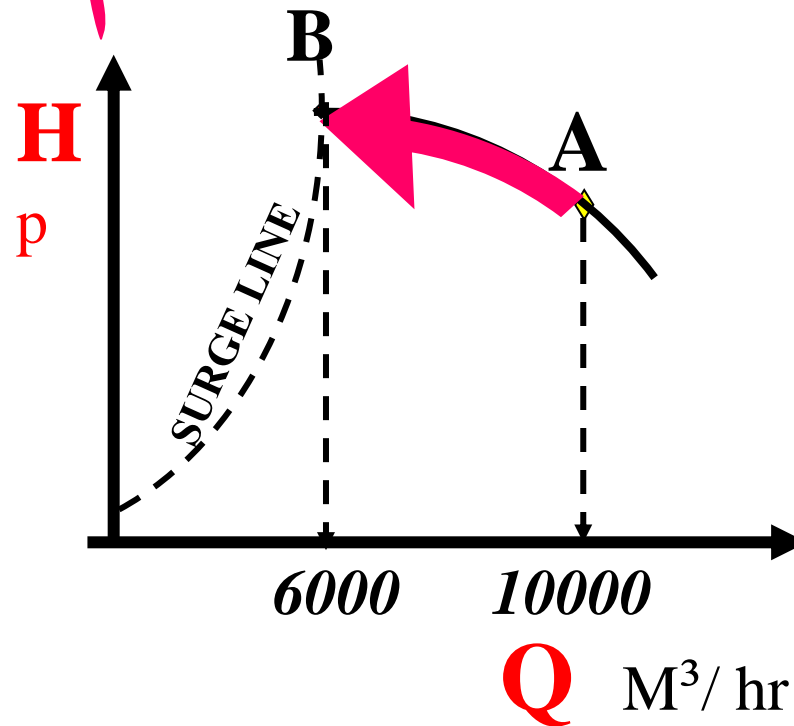
COMPRESSOR

is surging

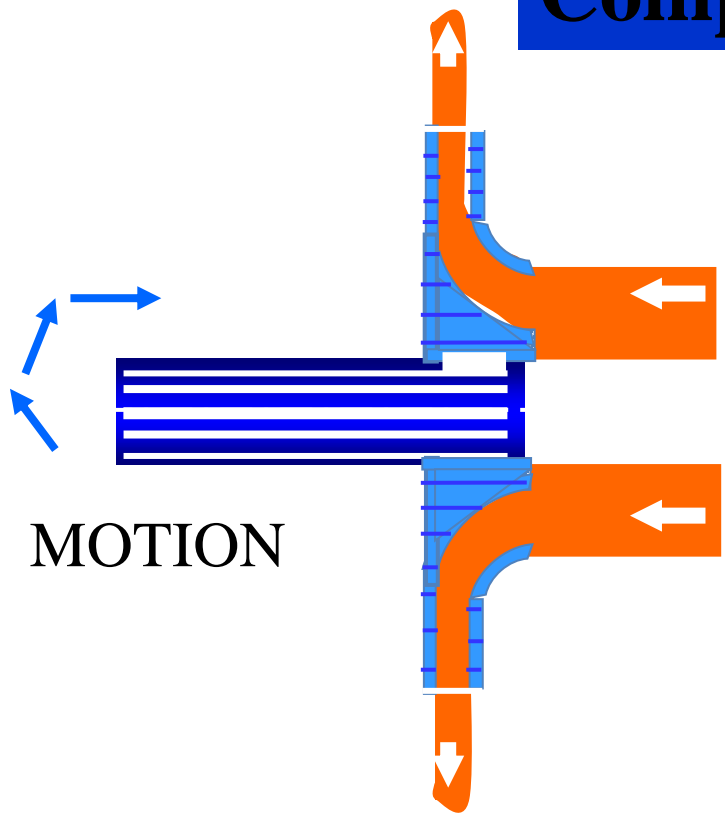


5 to 20 cycles per second

SURGE PHENOMENON

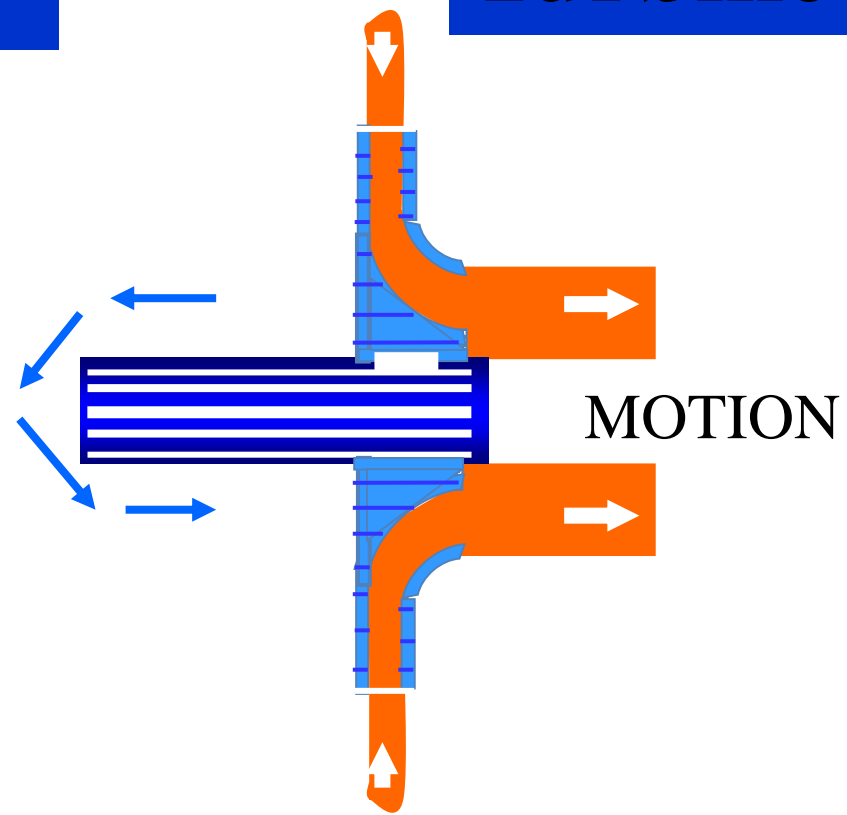


Compressor



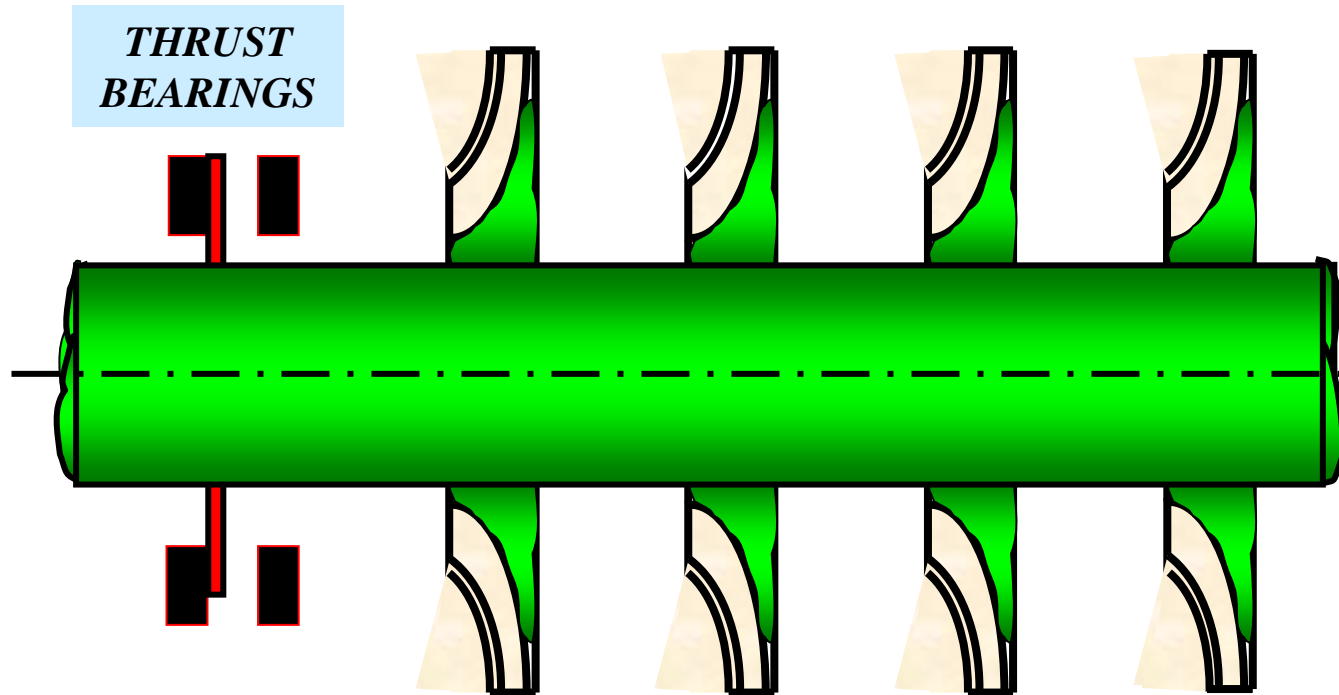
HYDRAULIC
ENERGY

Turbine



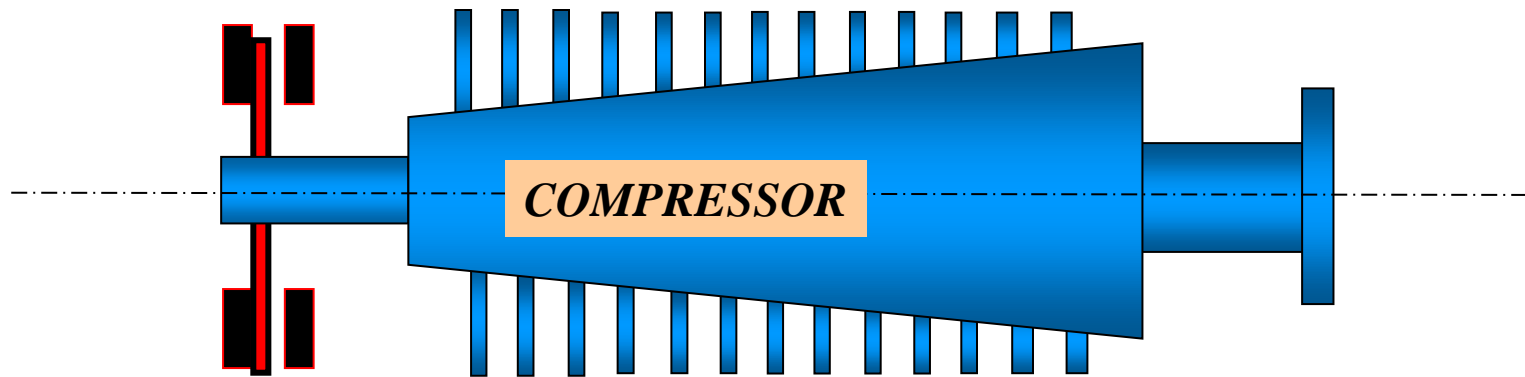
HYDRAULIC
ENERGY

EFFECT OF SURGE ON CENTRIFUGAL COMPRESSOR ROTOR



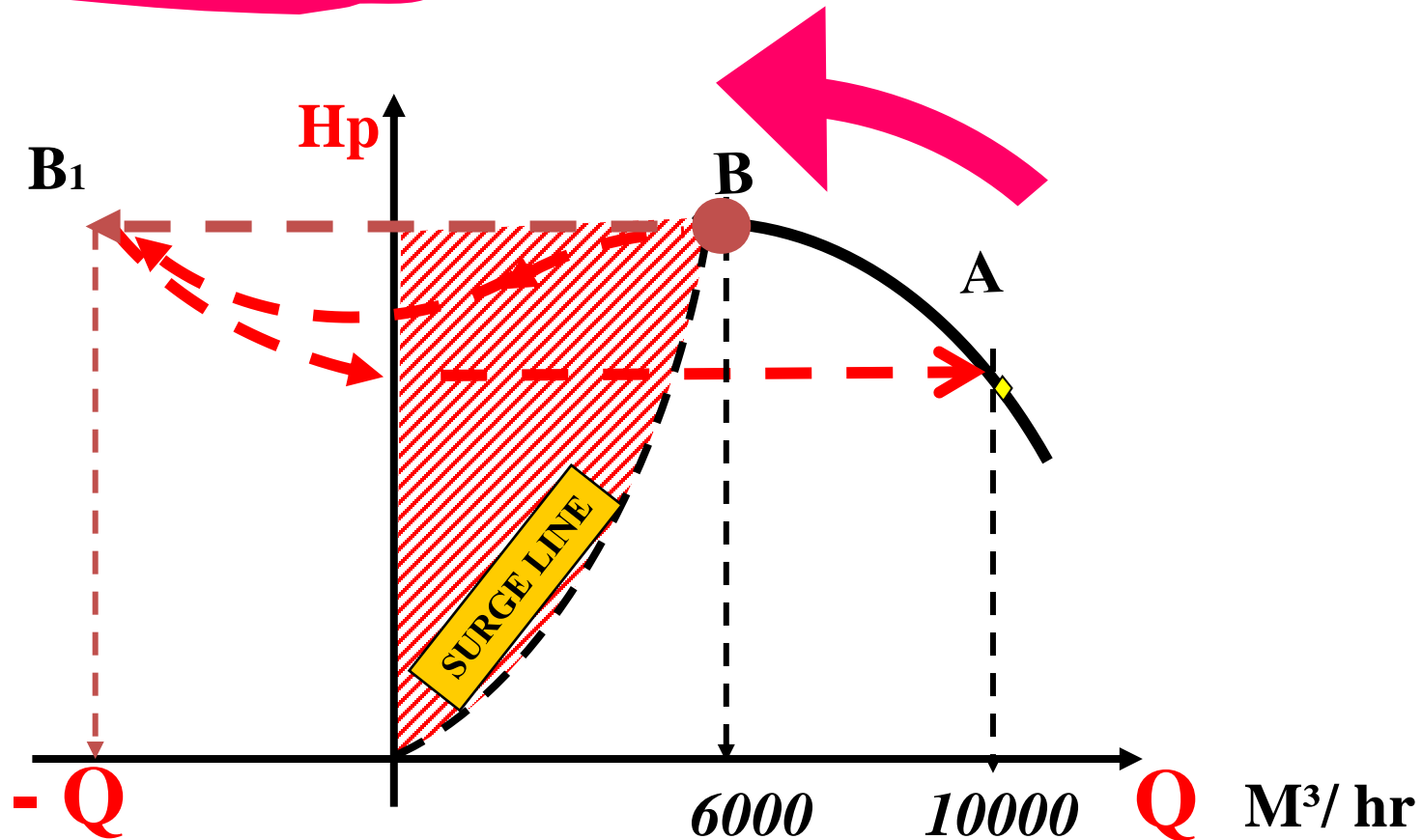
**SURGE WILL DAMAGE THE COMPRESSOR
THRUST BEARINGS**

*THRUST
BEARINGS*



*SURGE WILL DAMAGE THE
COMPRESSOR **THRUST BEARINGS***

SURGE CYCLE



5 to 20 cycles per second

ANTI-SURGE METHODS

1- *BY PASS WITH ANTI - SURGE VALVE*

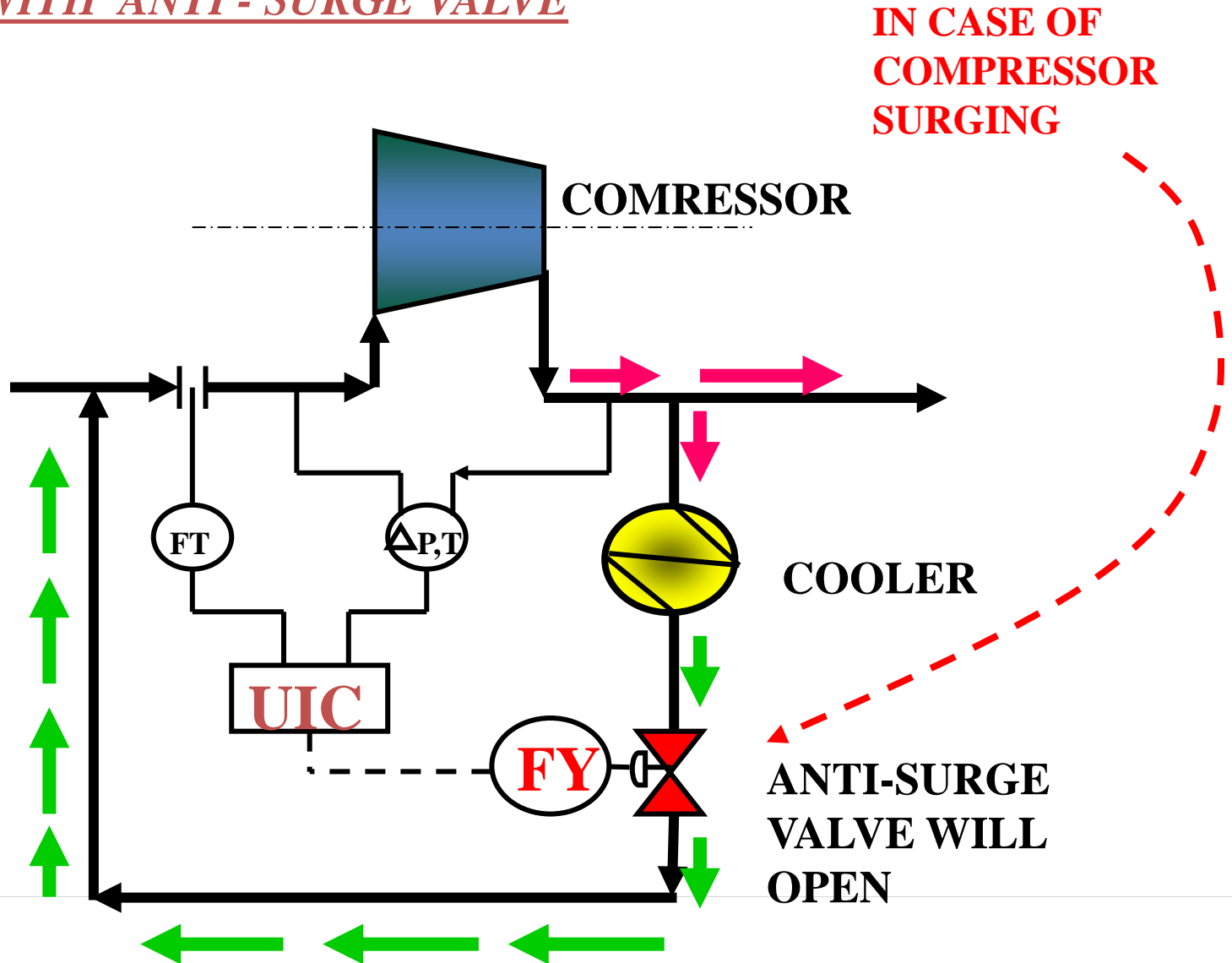
**THIS METHODS IS COMMONLY
USED FOR GAS COMPRESSORS**

2 -*BLOW OFF VALVE*

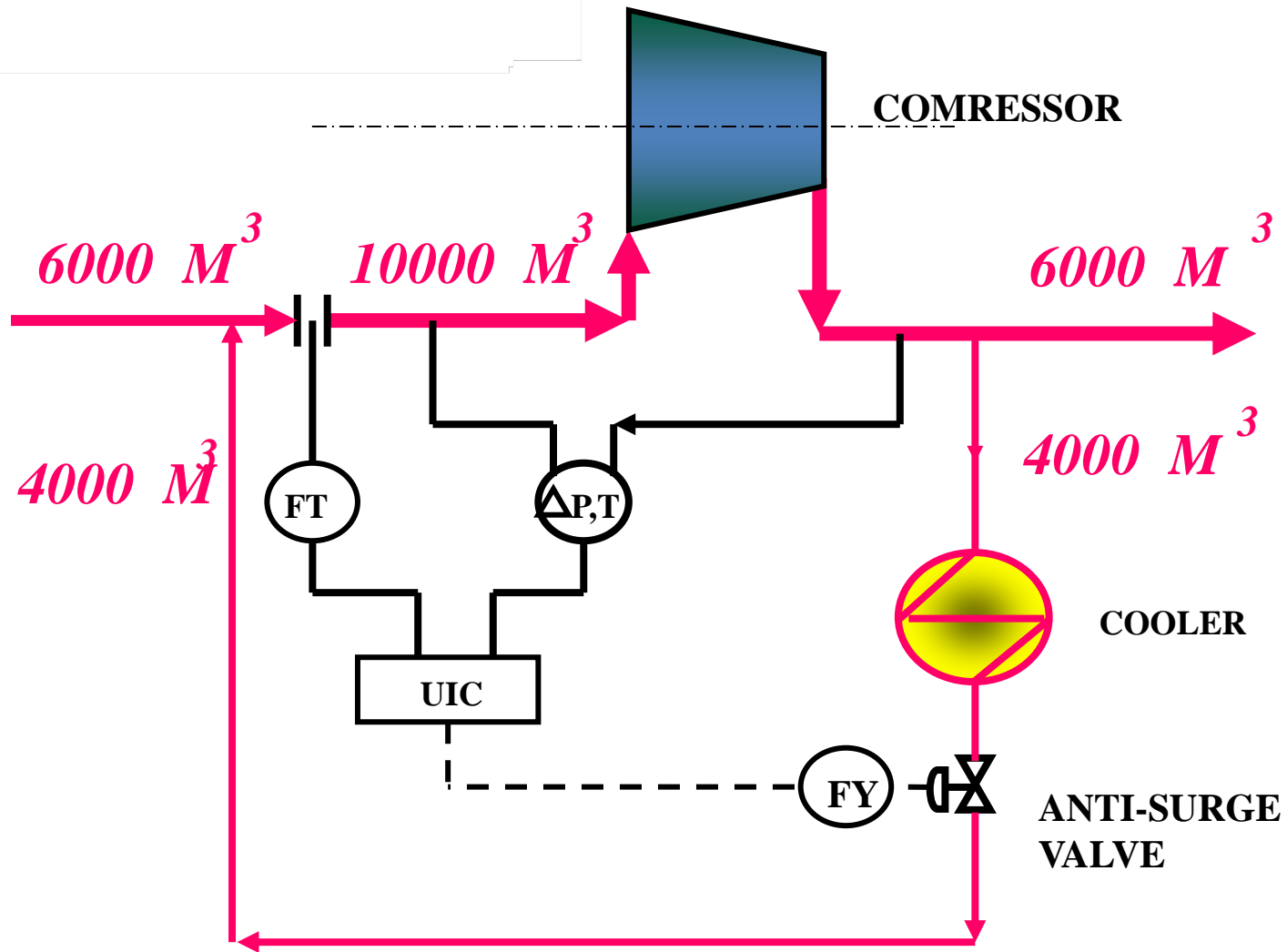
**THIS METHODS IS COMONLY
USED FOR AIR COMPRESSORS**

ANTI-SURGE METHODS

1- BY PASS WITH ANTI - SURGE VALVE



1. BY PASS WITH ANTI - SURGE VALVE

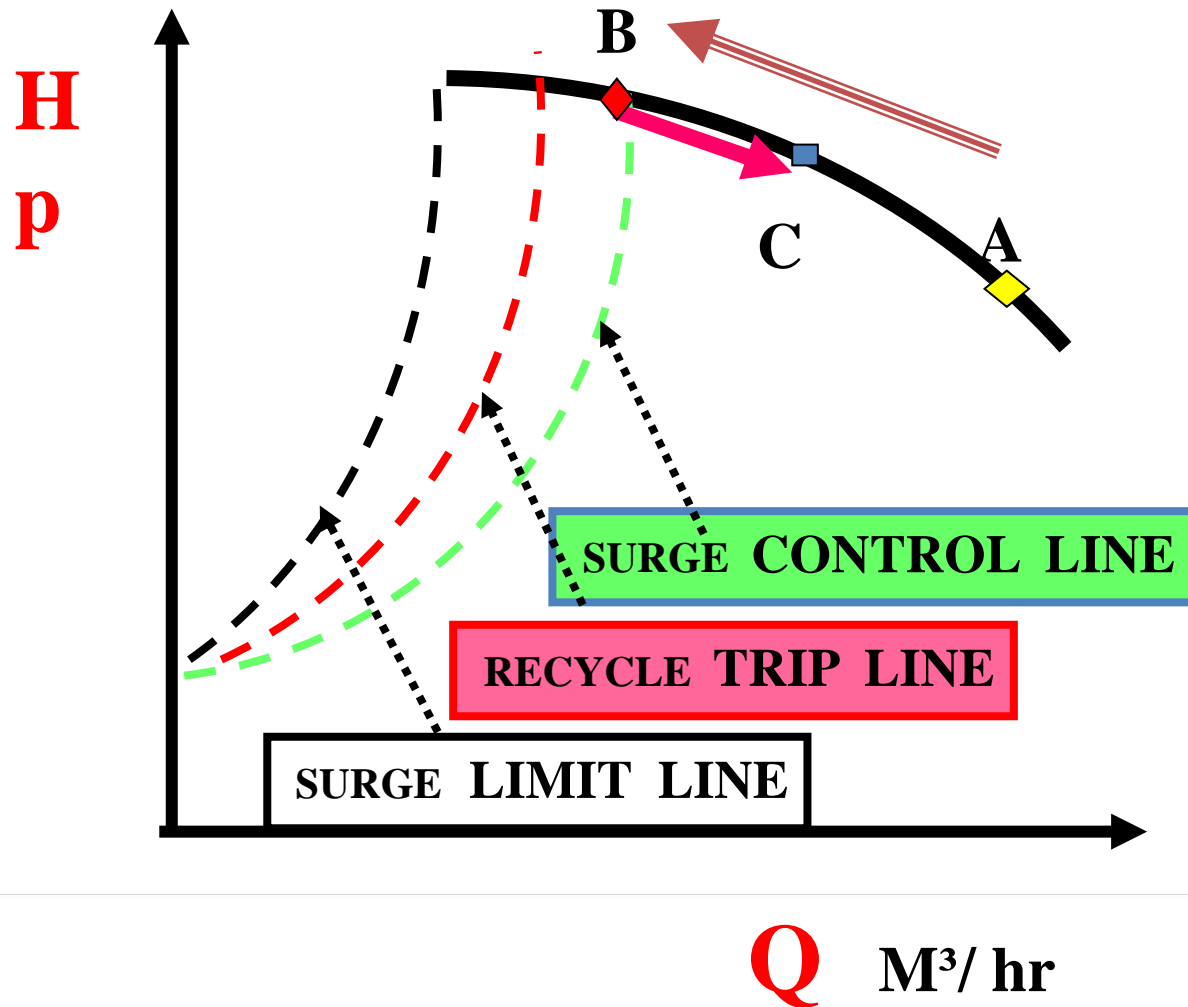


UIC = ANTI - SURGE INTEGRATED CONTROLER

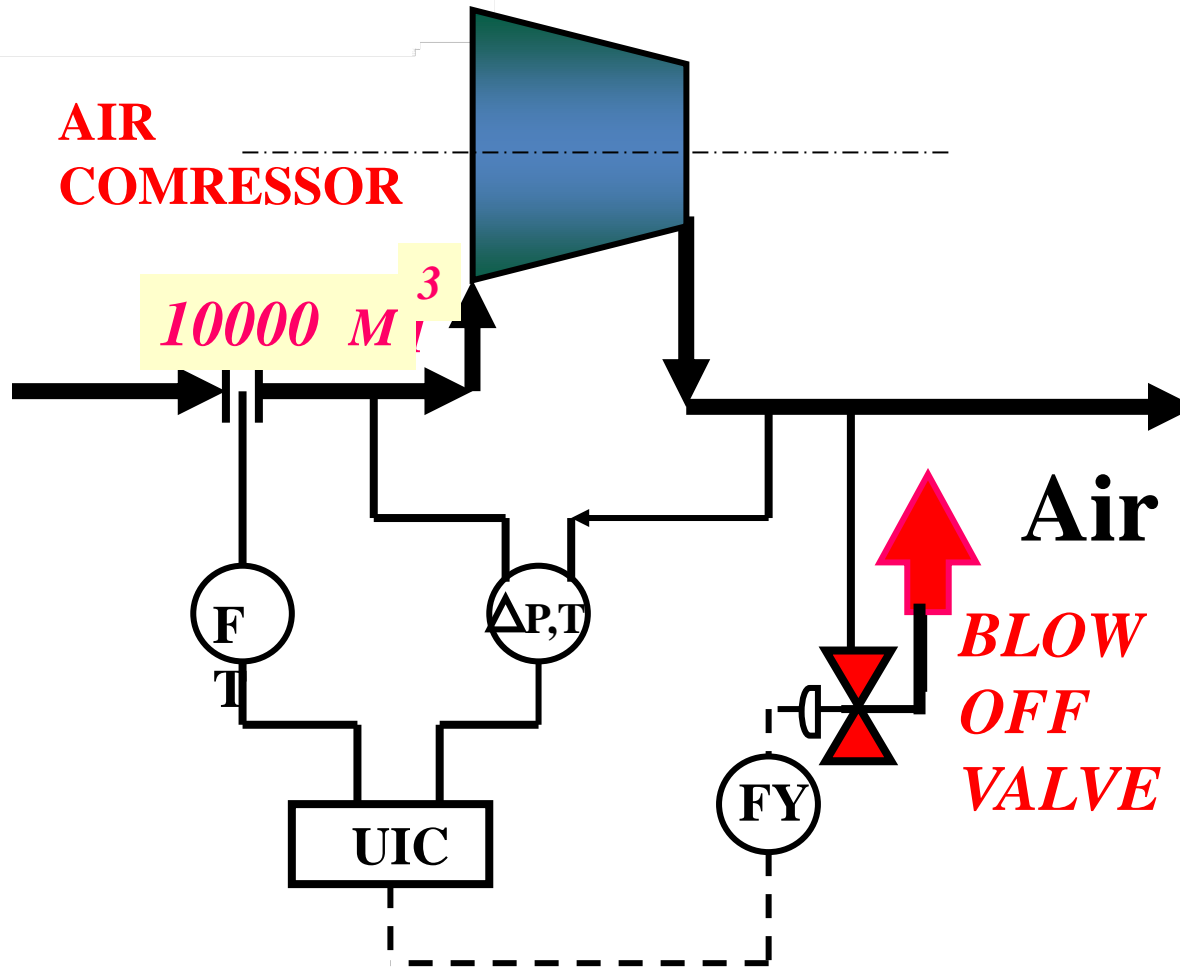
FY^{ετ} = TRANSDUCER

BY PASS WITH ANTI - SURGE VALVE

GRAPHICALLY



2-BLOW OFF VALVE

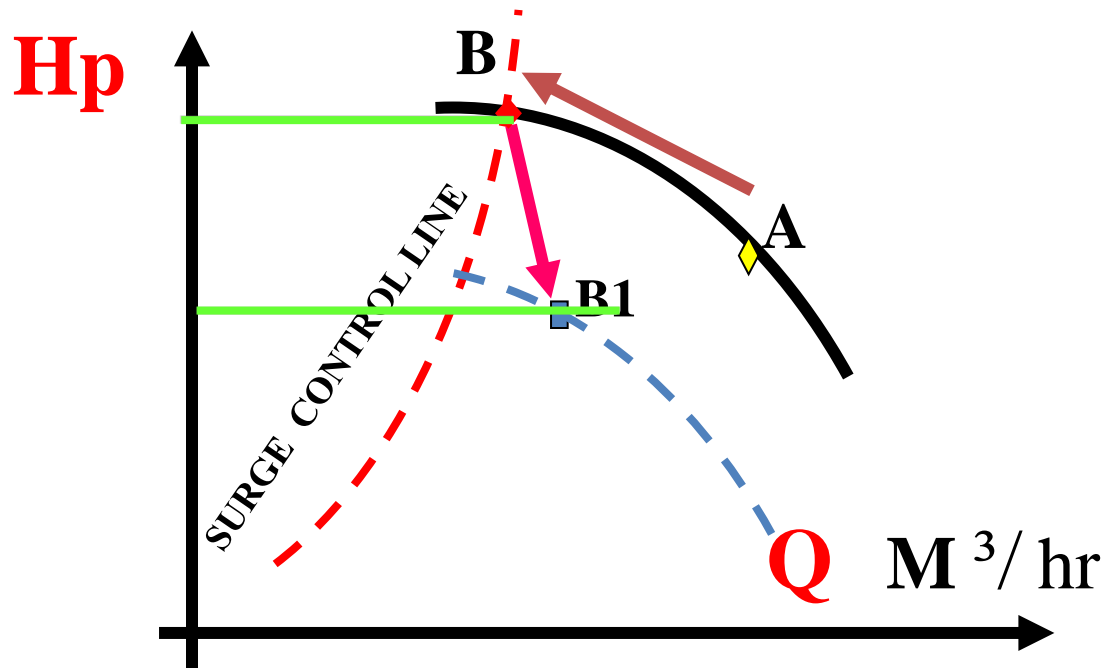


UIC = ANTI-SURGE INTEGRATED CONTROLLER

FY_{ξo} = TRANSDUCER

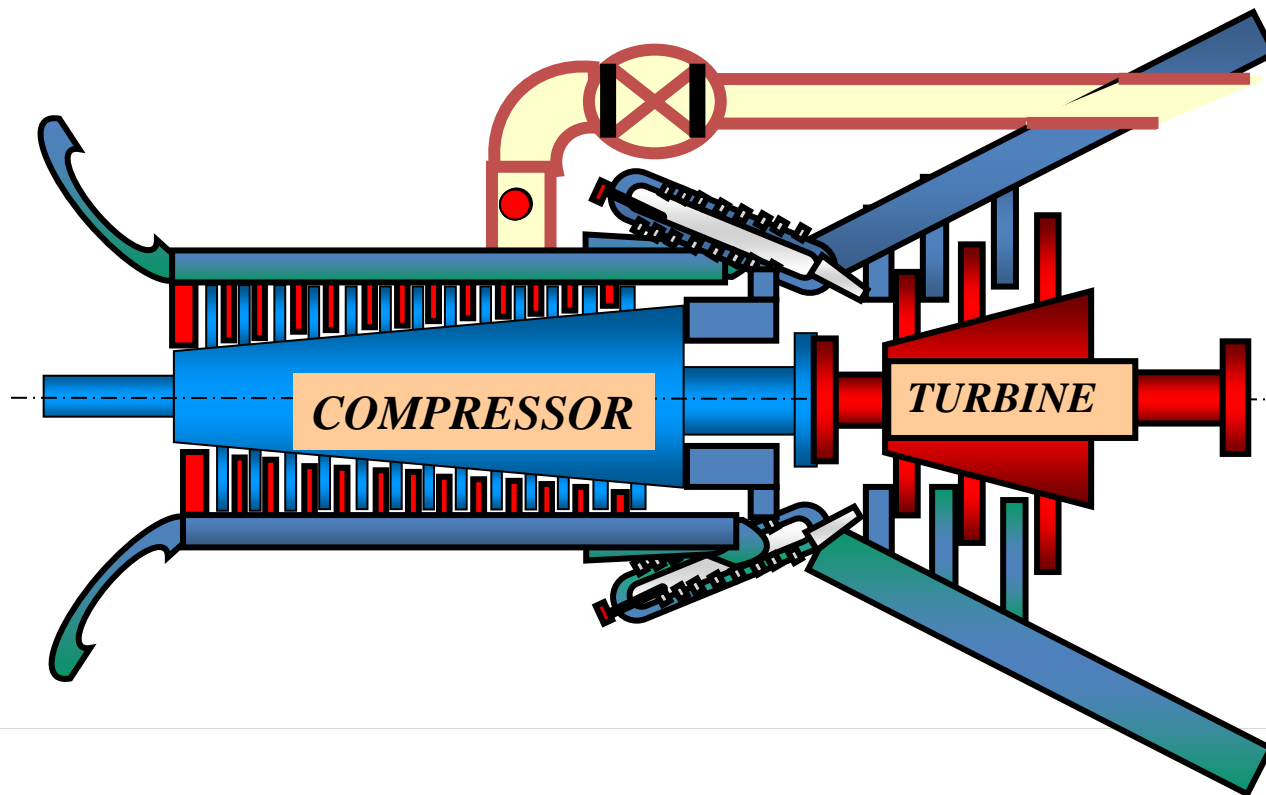
2 – BLOW OFF VALVE

GRAPHICALLY



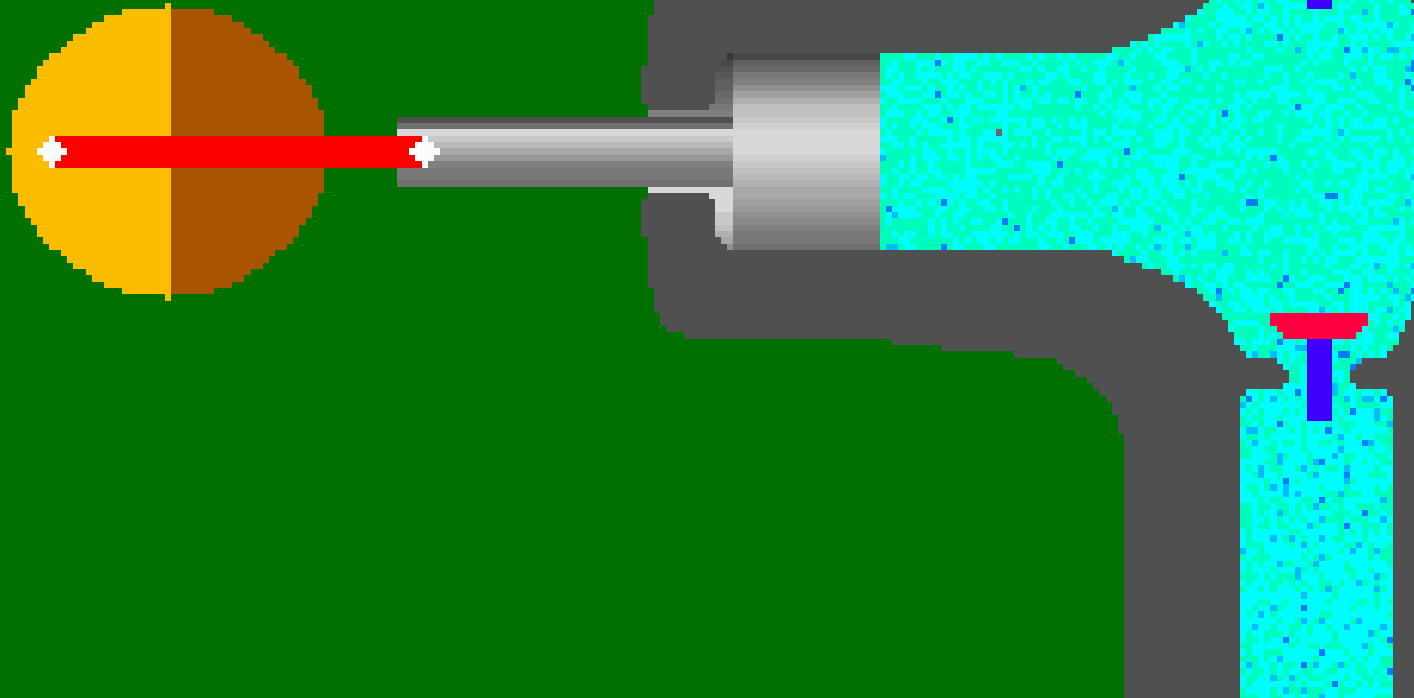
IN CASE OF GAS TURBINE
AIR COMPRESSOR SURGE
BLEED VALVE WILL OPEN

BLEED VALVE

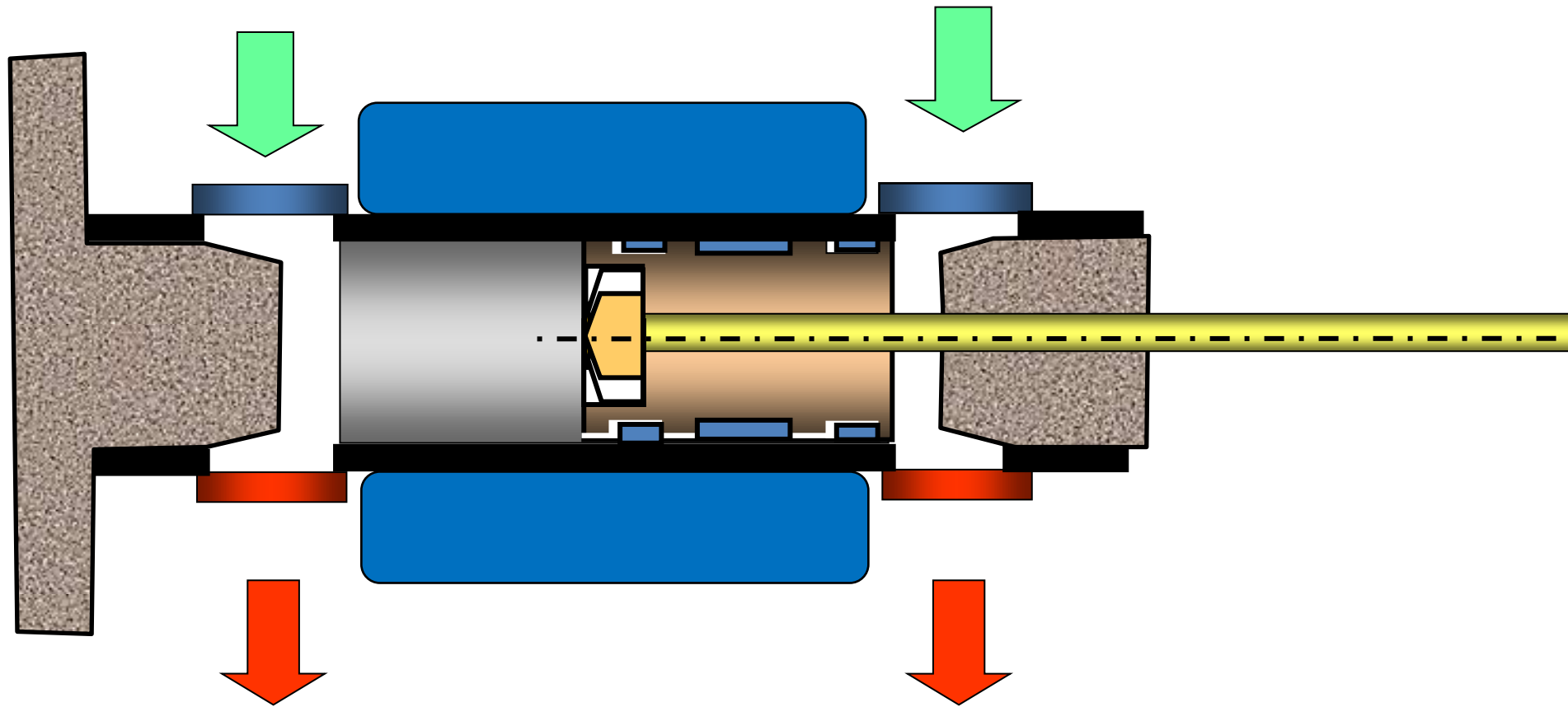


Positive Displacement Compressors

Reciprocating Compressors



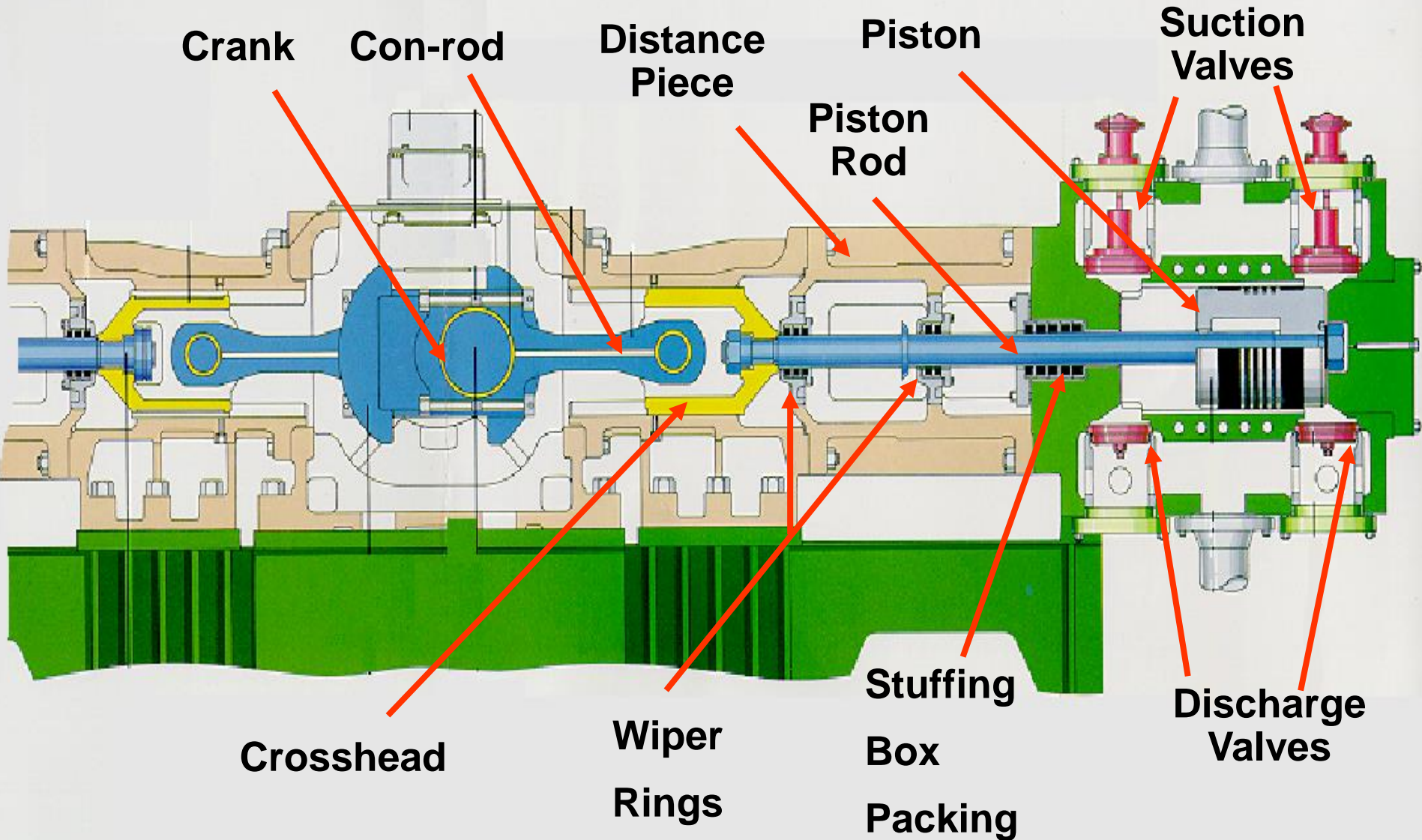
Reciprocating Compressors Construction



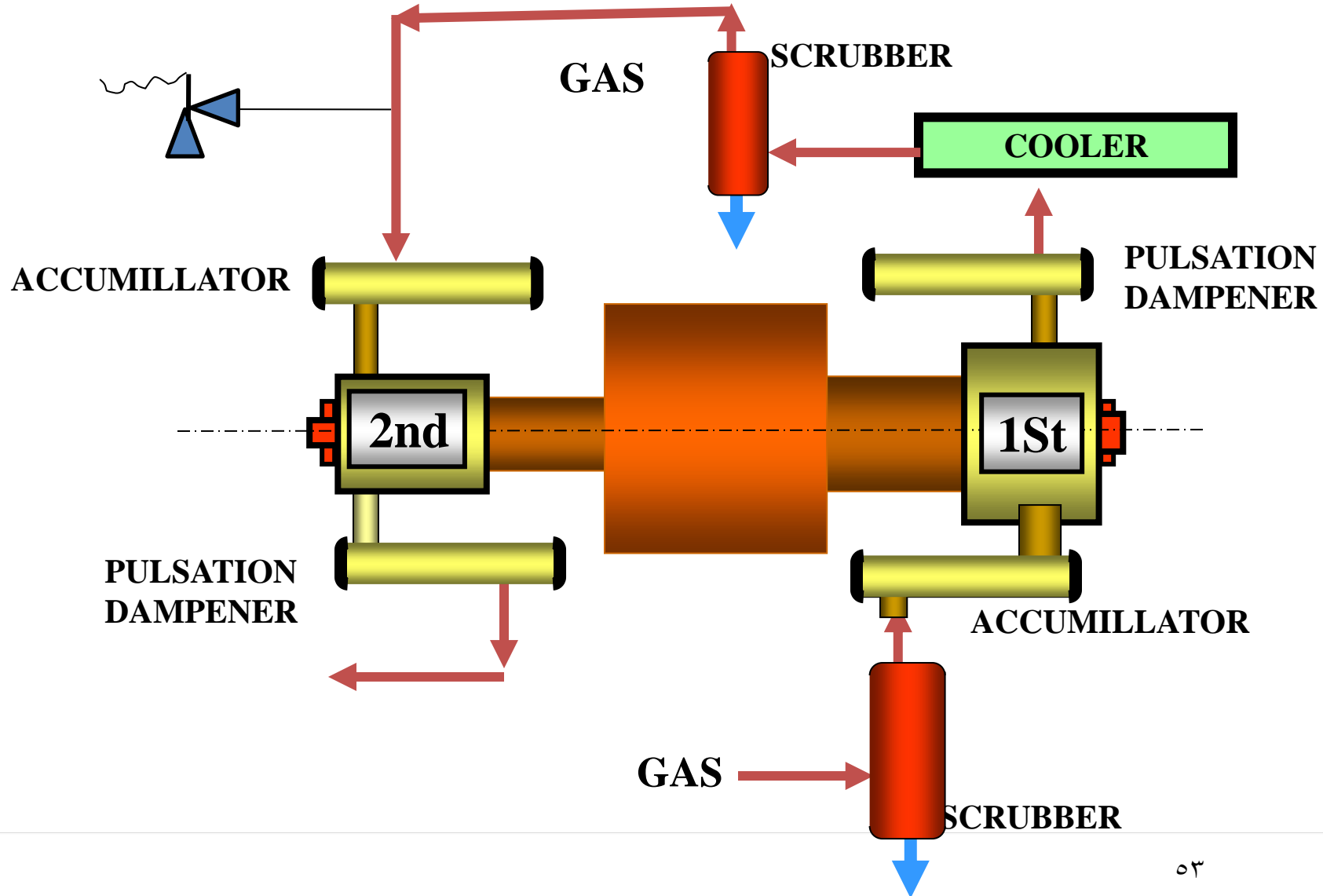
Reciprocating Compressors properties

- **Lower Power Density**
- **Low Speed (300–1800 rpm)**
- **Generally used in low volumetric rate applications (< 500 acfm [850 m³/h])**
- **Higher maintenance**
- **Unaffected by changes in gas properties (MW, T)**
- **Most common driver – gas engine, electric motor**
- **Pulsating flow/vibration**

Reciprocating Compressors construction

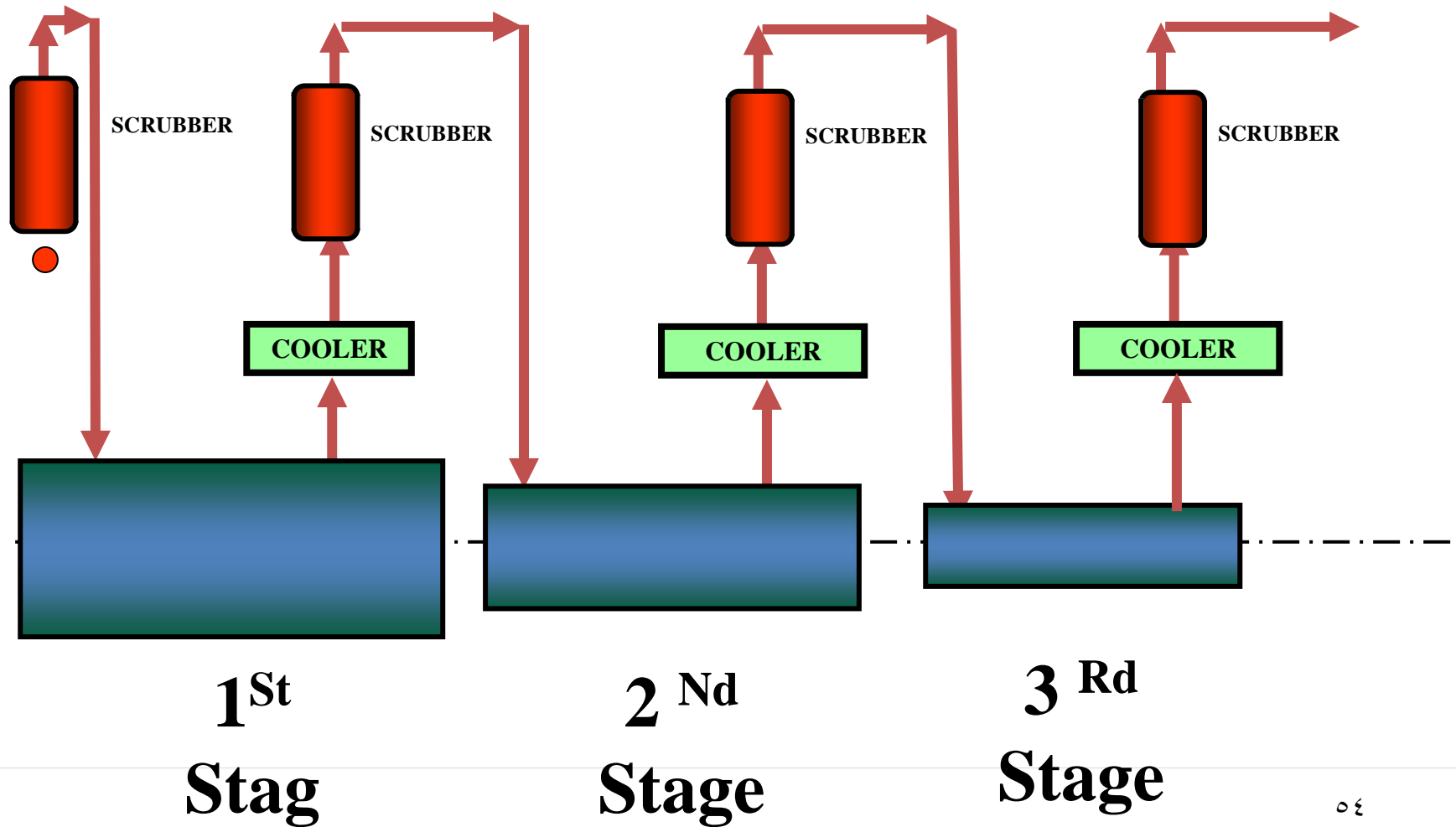


Multi stage reciprocating compressor

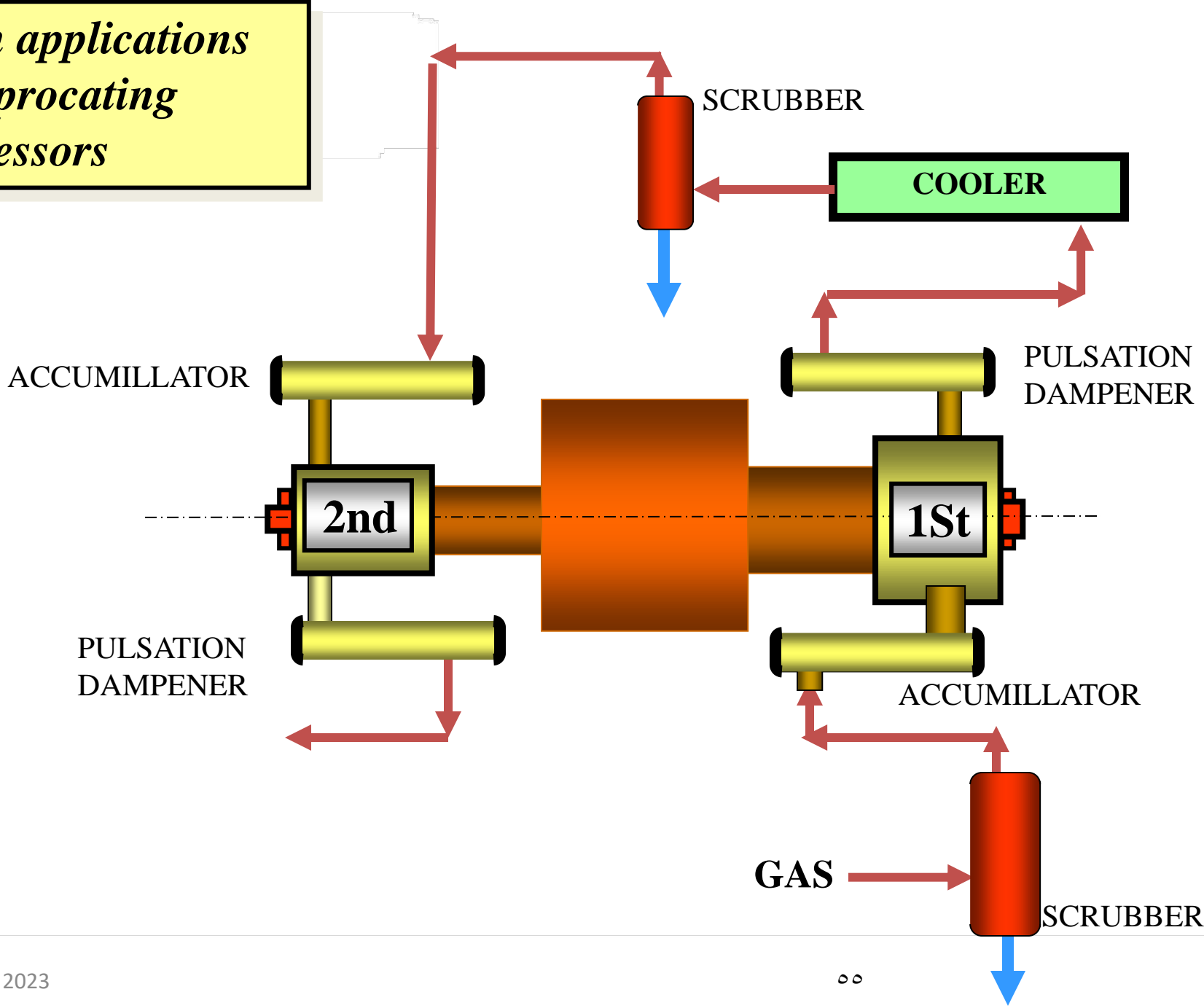


Multi stage reciprocating compressor

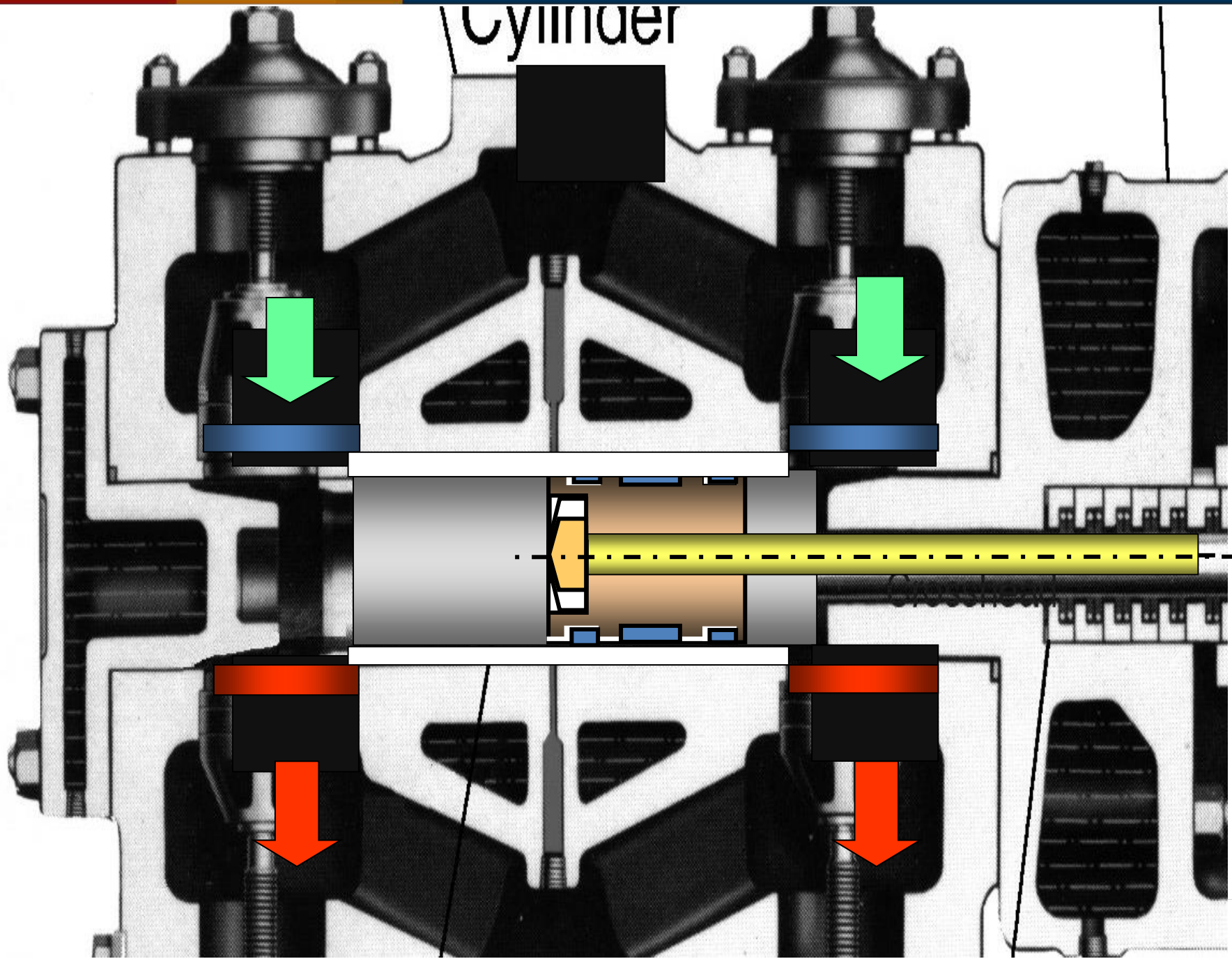
Compressors intermediate pressures



*Design applications
of reciprocating
compressors*



Cylinder



Reciprocating Compressors Flow Regulation

1- VARIABLE CLEARANCE

- Manual Operated Clearance
- Pneumatic Operated Clearance

2- SUCTION VALVE UNLOADERS

3- VARIABLE SPEED

4- BYPASS

5- Shut down the Compressor Periodically (Start-Stop Operation)

1- VARIABLE CLEARANCE

Manual Operated Clearance

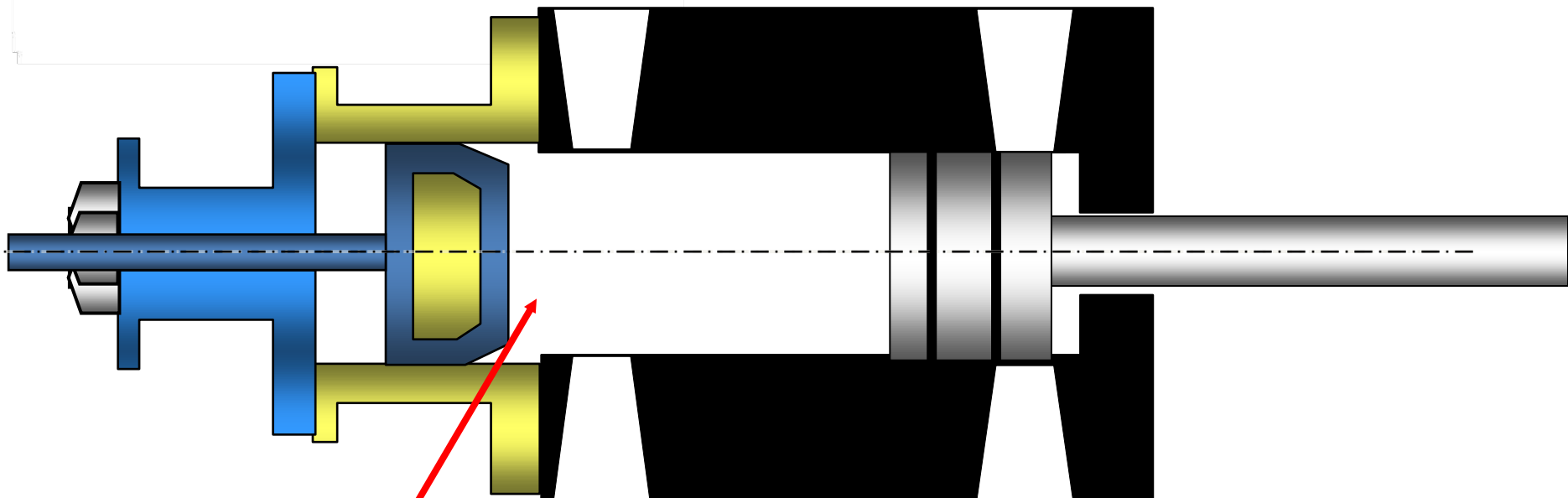
There is a clearance existed in the cylinder head.

A piston rod complete with an adjustable nut is connected to piston with packing to seal the clearance volume in front of the compressor piston.

The more the clearance volume in front of the comp. piston, The less the compressor flow rate.

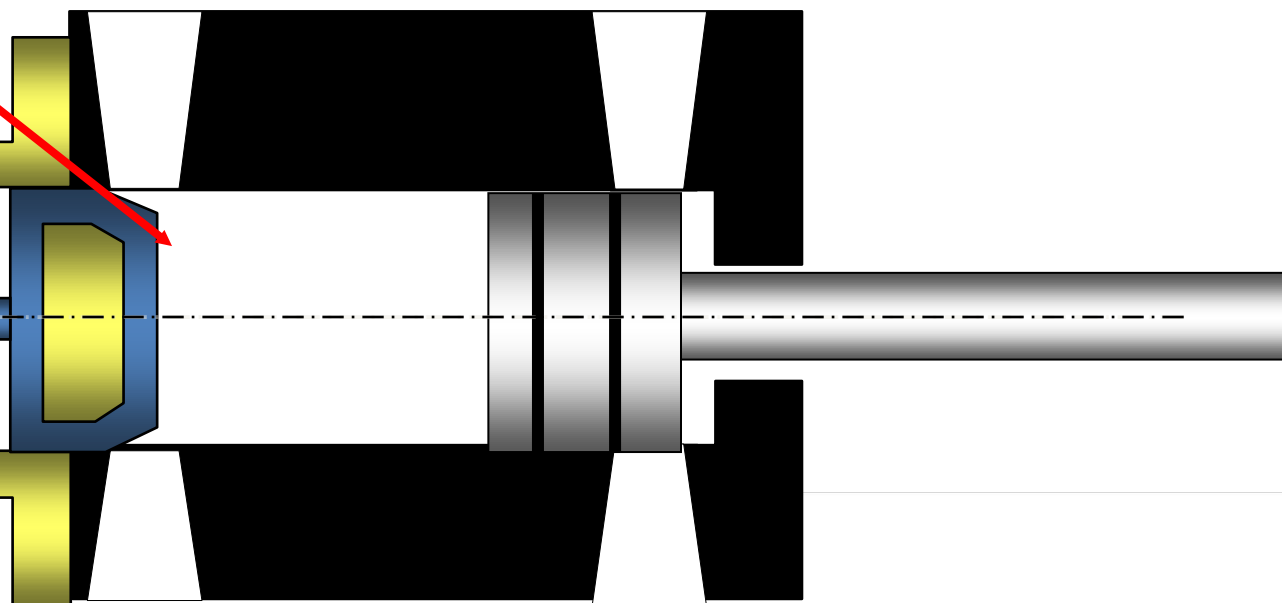
No effect of the clearance volume in front of the piston on the compressor discharge pressure.

Clearance Volume Effect

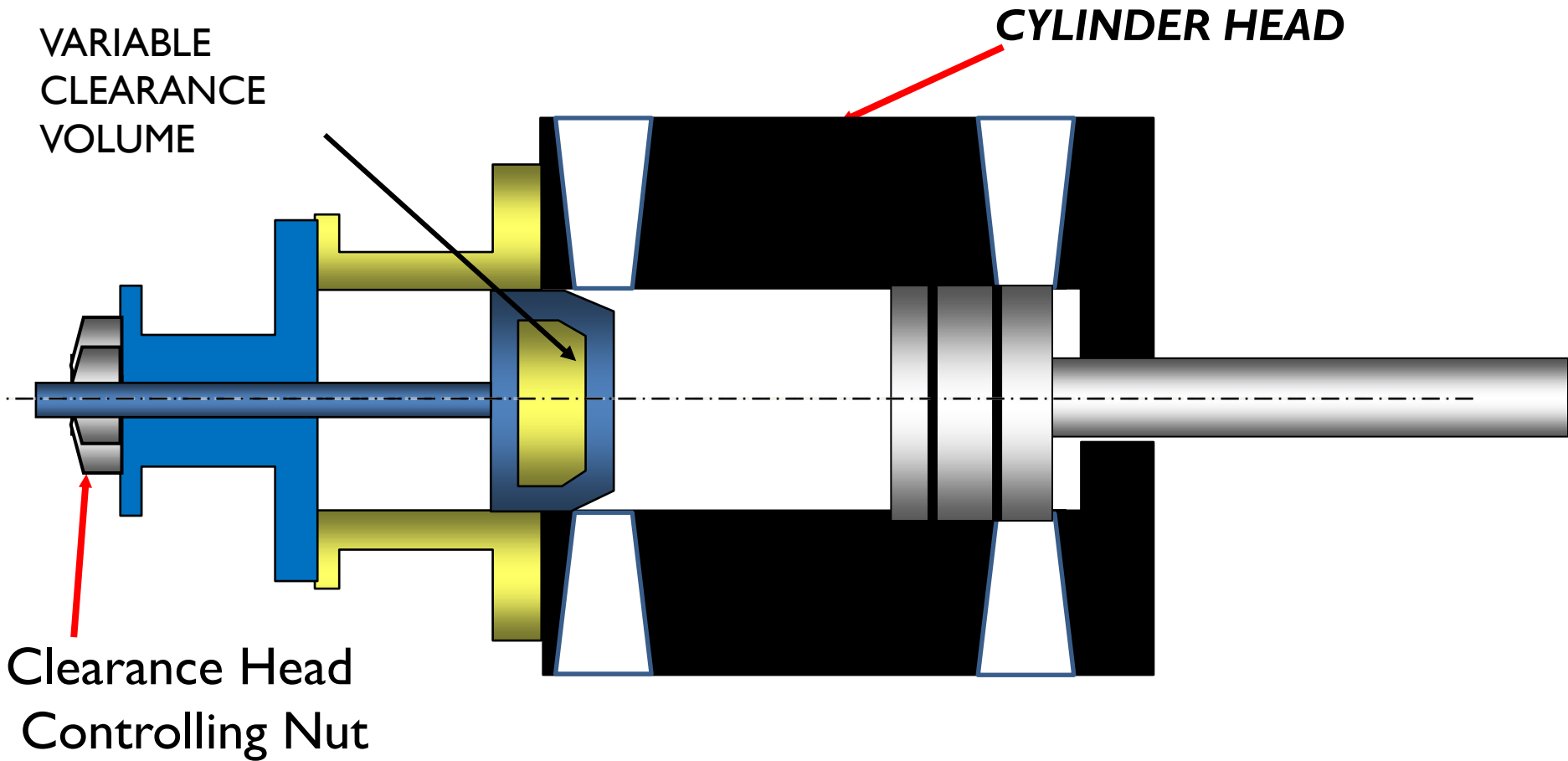


**Clearance
Volume**

**Clearance
Head**

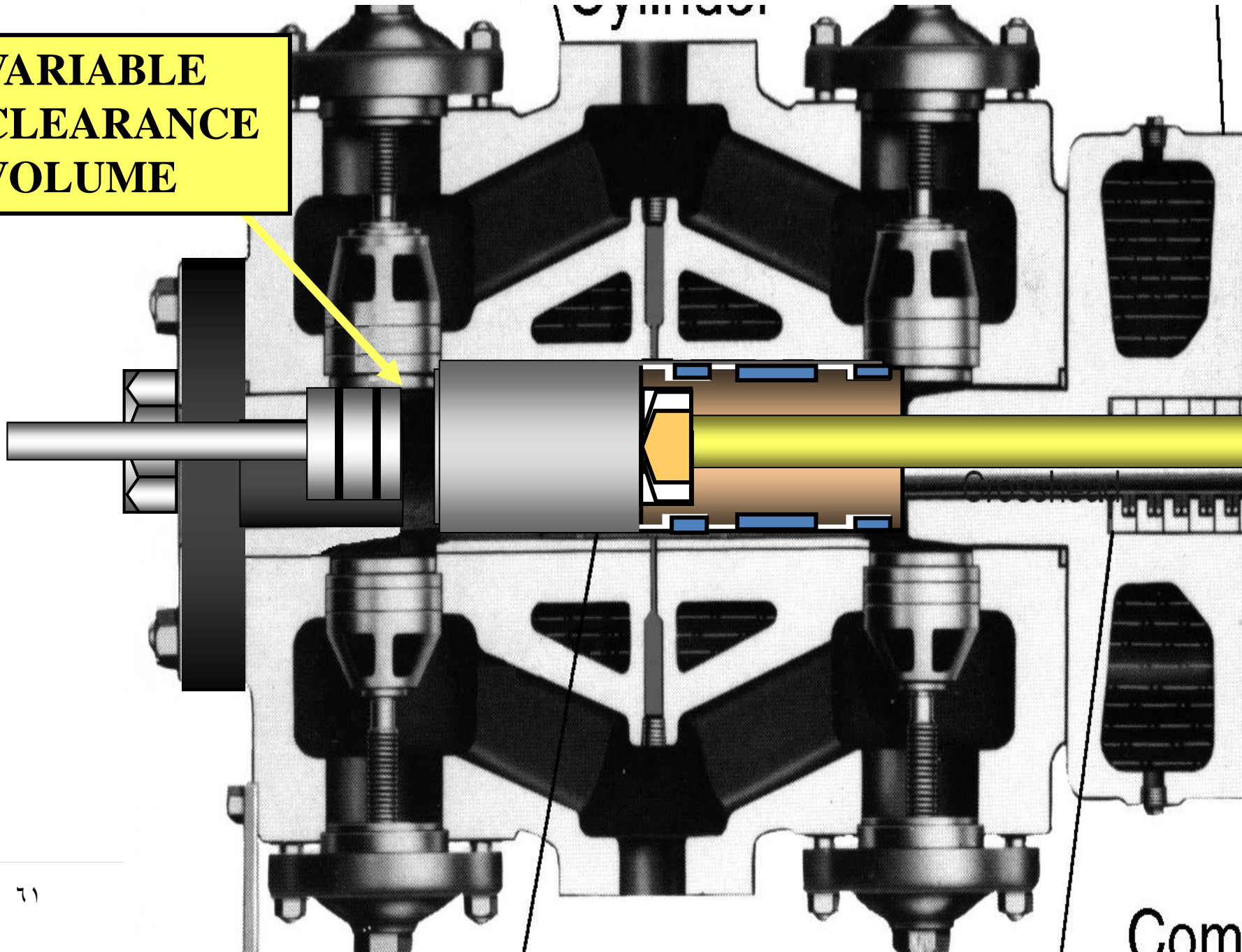


Clearance Volume Effect



Manual Operated Clearance

**VARIABLE
CLEARANCE
VOLUME**



EFFECT OF CLEARANCE VOLUME

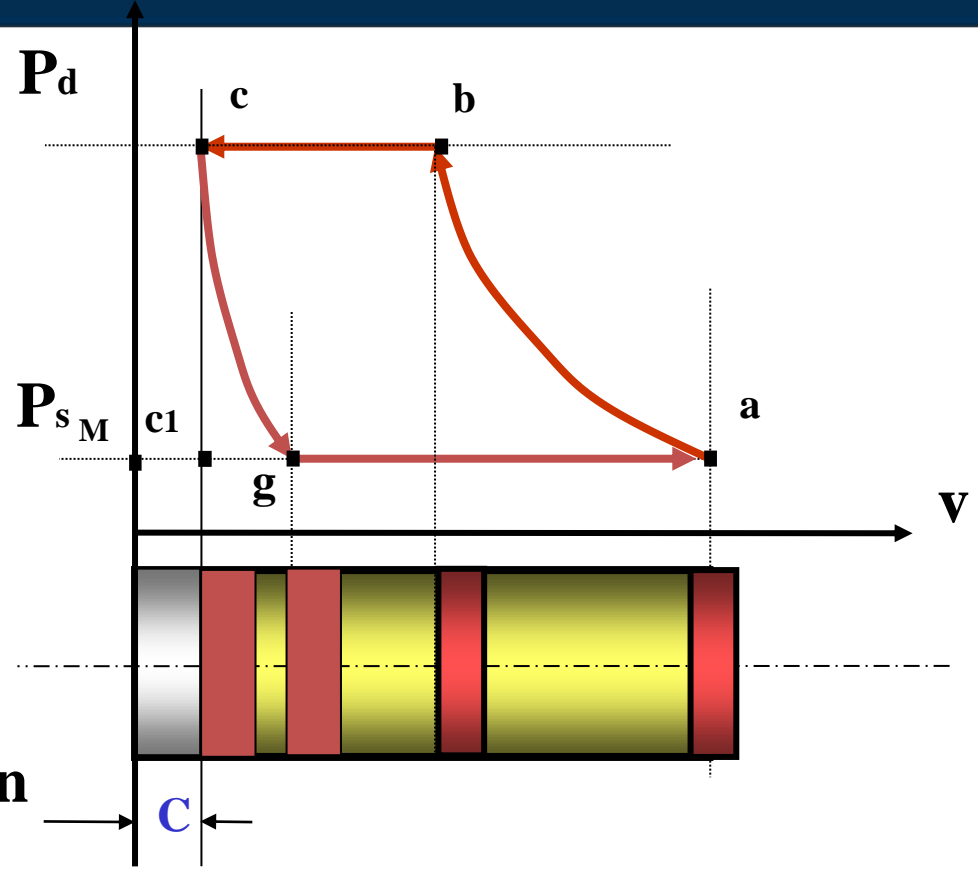
At

Point **a** suction valves closed

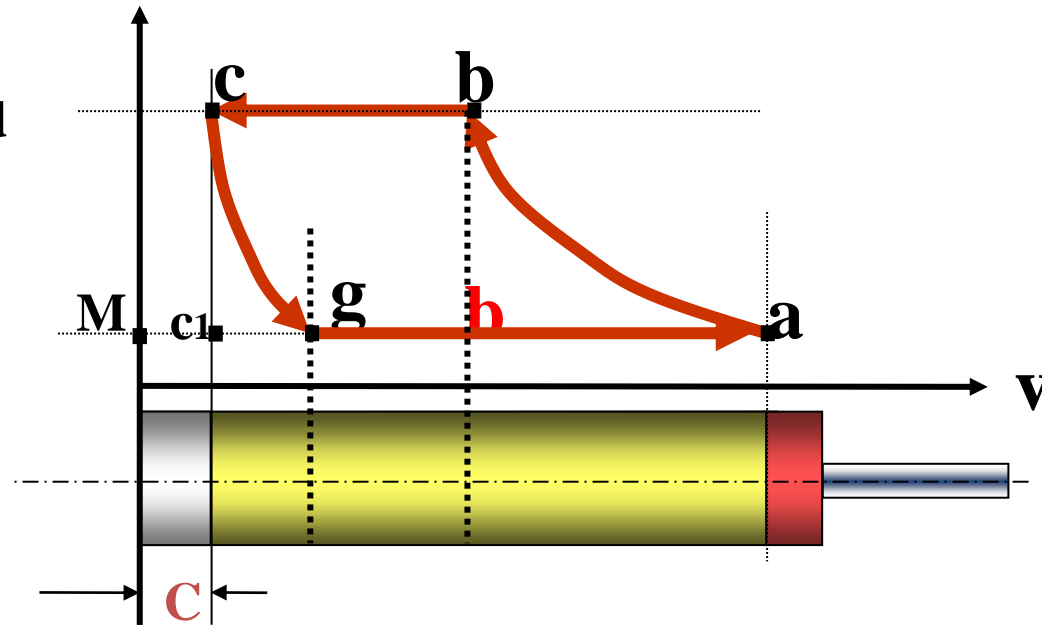
Point **b** discharge valves open

Point **c** discharge valves closed

Point **g** suction valves open



C = Clearance volume

P_d P_s 

Pressure ratio

$$= \frac{P_d}{P_s} \quad (\text{ABSOLUTE})$$

Volumetric efficiency

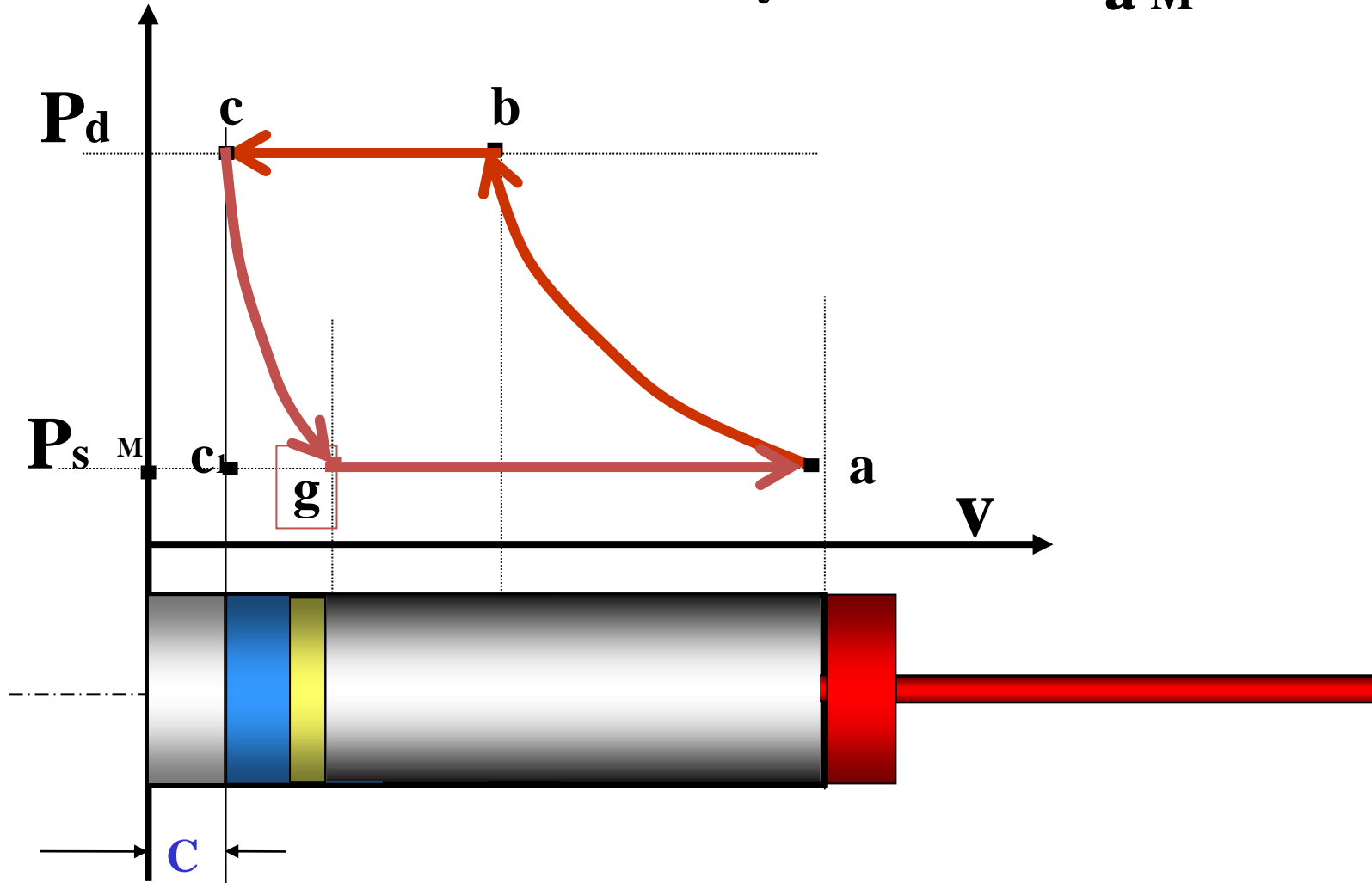
$$= \frac{V_{ag}}{V_{\text{cylinder}}} = \frac{ag}{aM}$$

Compression ratio

$$= \frac{V_{\text{cylinder}}}{V_{bM}} = \frac{aM}{bM}$$

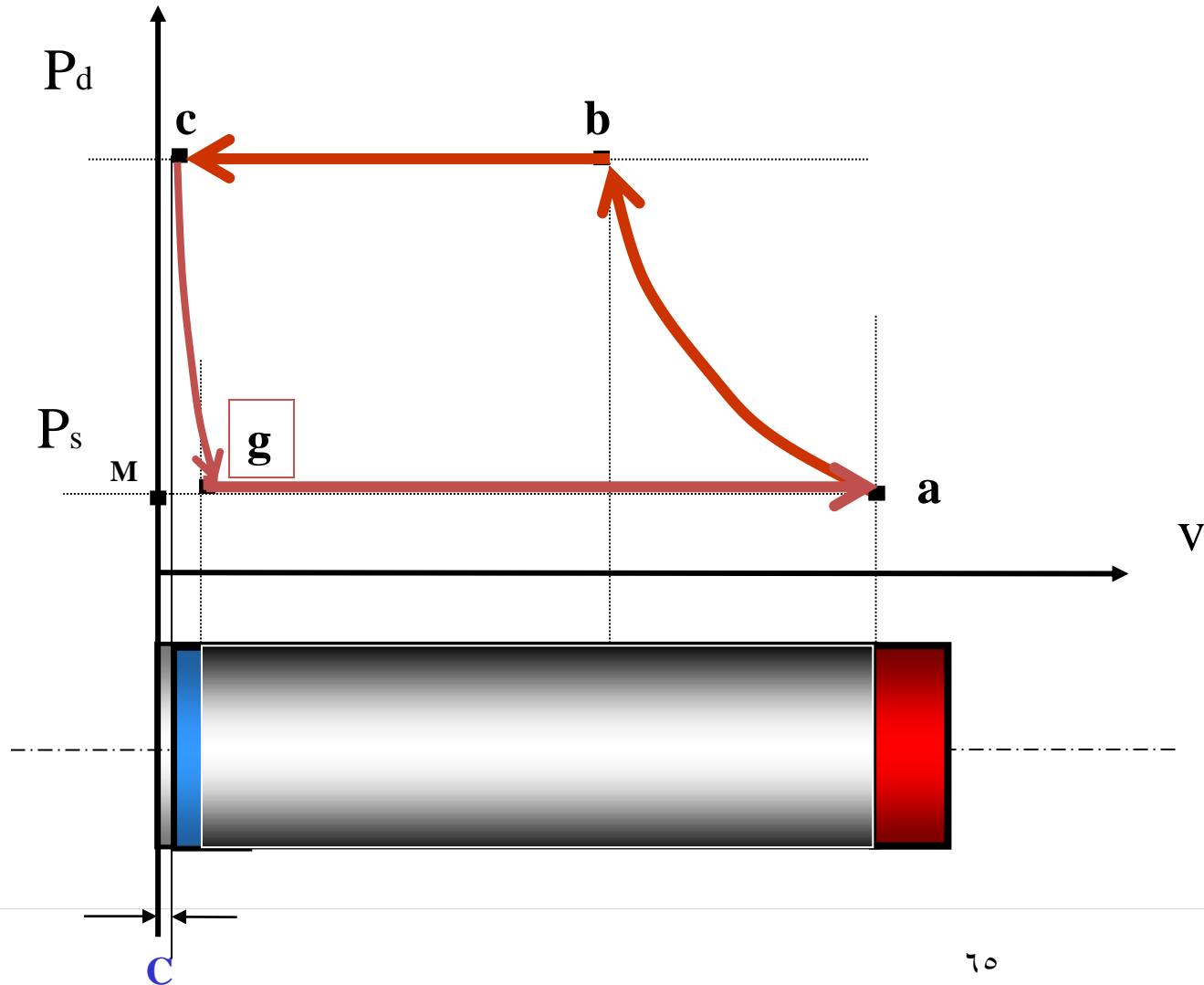
Volumetric efficiency

$$= \frac{V_{ag}}{V_{cylinder}} = \frac{ag}{aM}$$



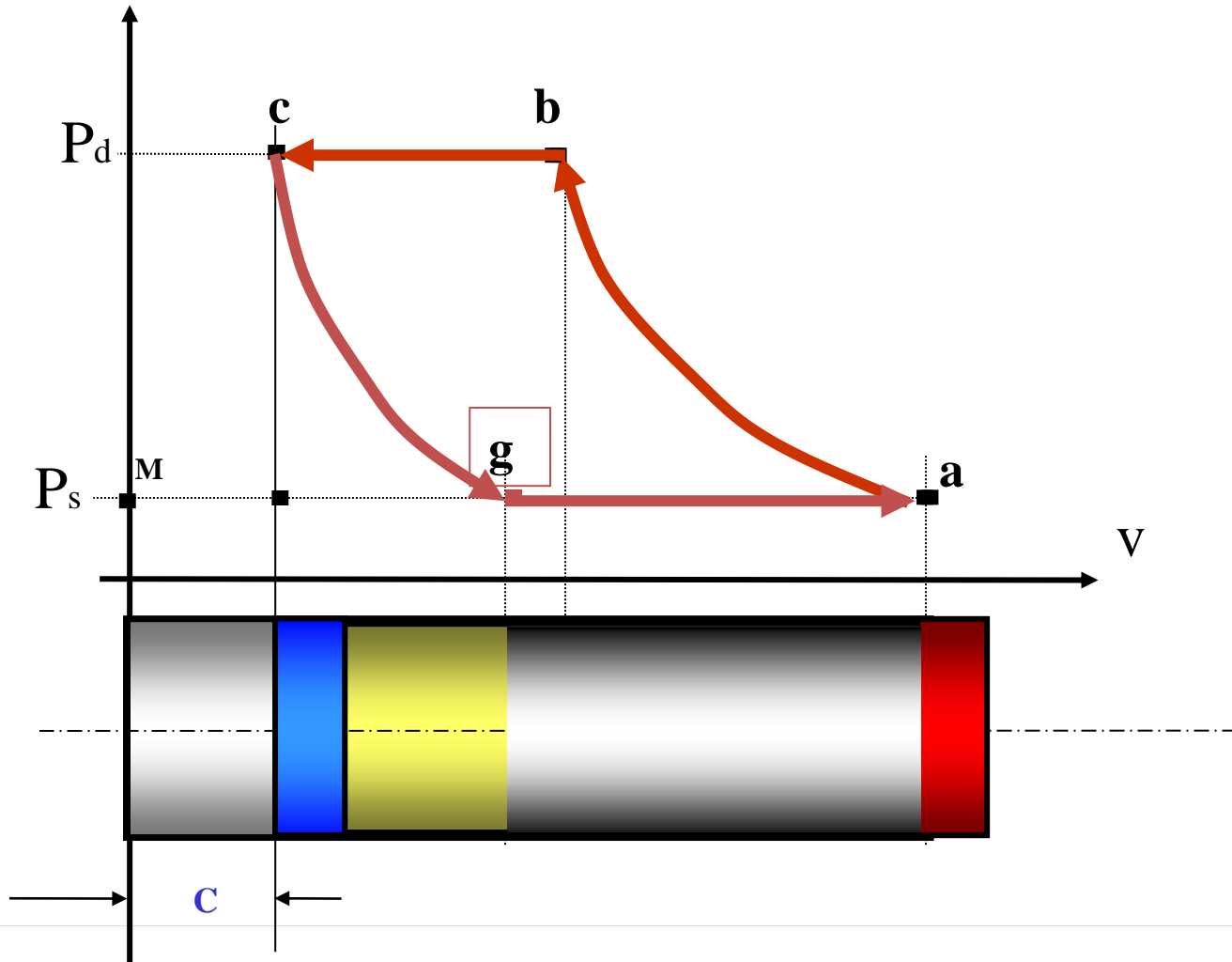
Volumetric efficiency

$$= \frac{V_{a g}}{V_{\text{cylinder}}} = \frac{a g}{a M}$$



Volumetric efficiency

$$= \frac{V_{a g}}{V_{\text{cylinder}}} = \frac{a g}{a M}$$



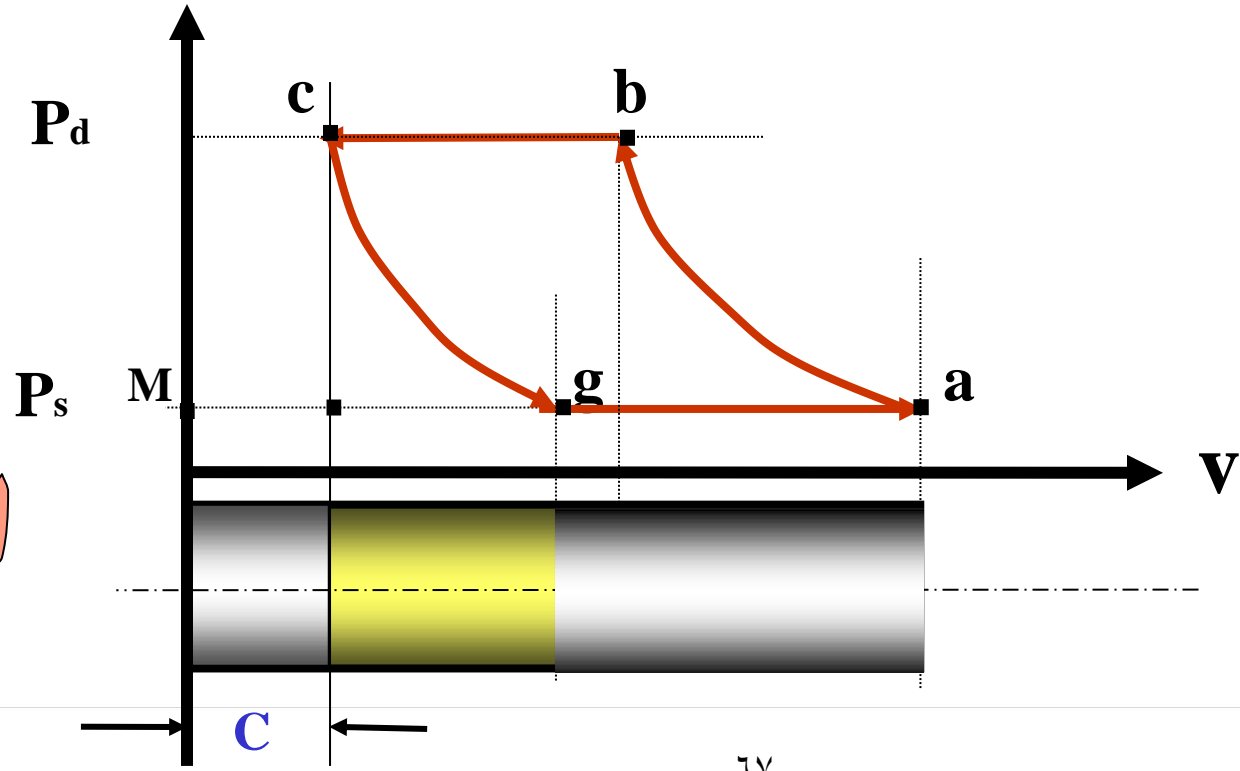
If C Increased suction valves will open at (g)

Volumetric efficiency

$$= \frac{V_{a g}}{V_{cylinder}} = \frac{a g}{a M}$$

EV DECREASES

Q DECREASES

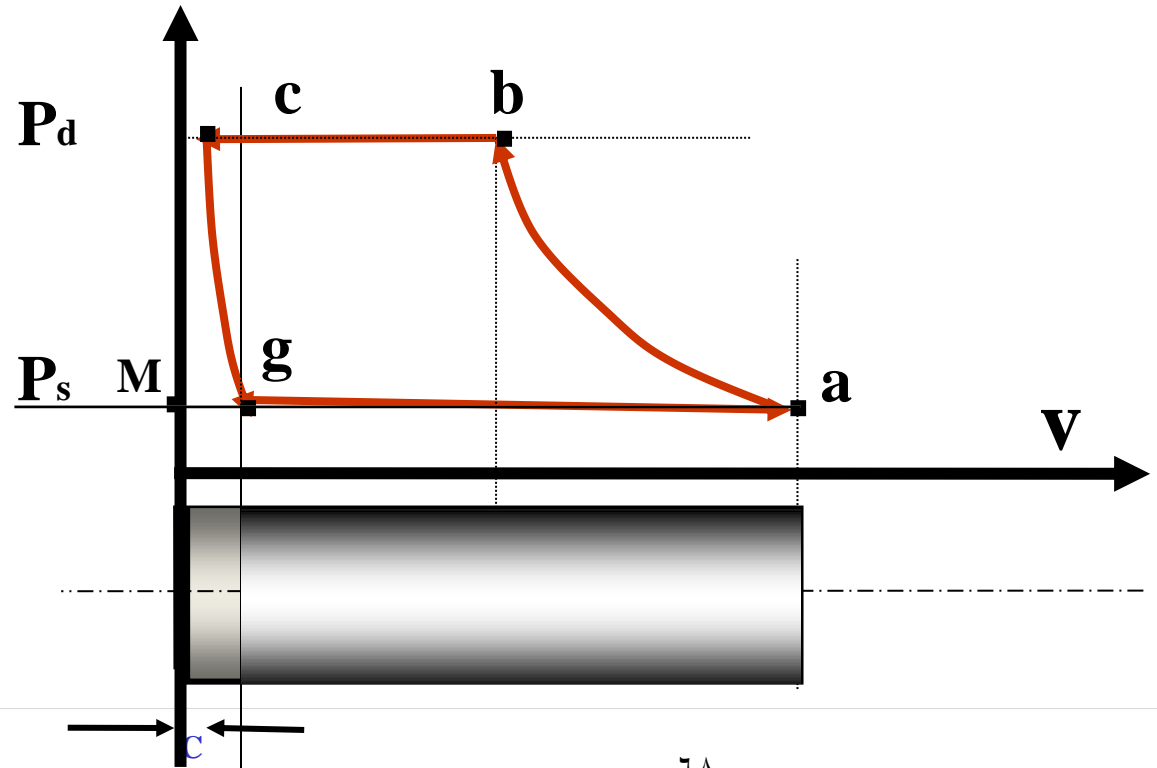


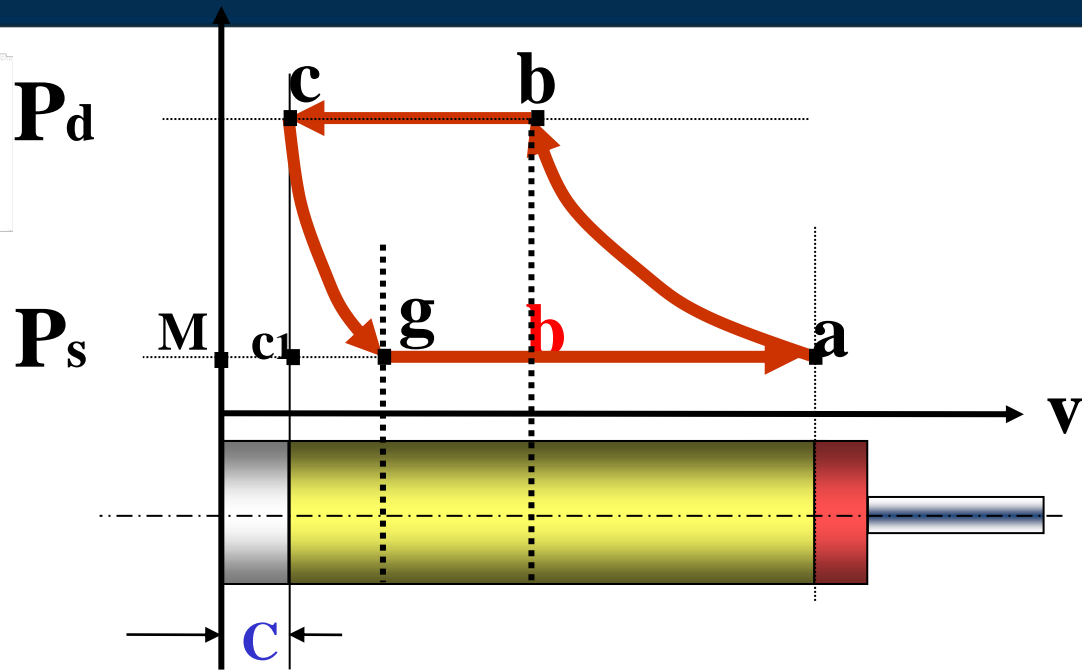
If **c** Decreased suction valves will open at
(g)

$$\text{Volumetric efficiency} = \frac{V_{a g}}{V_{\text{cylinder}}} = \frac{a g}{a M}$$

EV INCREASES

Q INCREASES





Pressure ratio

$$= \frac{P_d}{P_s} \quad (\text{ABSOLUTE})$$

Volumetric efficiency

$$= \frac{V_{a g}}{V_{\text{cylinder}}} = \frac{a g}{a M}$$

Compression ratio

$$= \frac{V_{\text{cylinder}}}{V_{b M}} = \frac{a M}{b M}$$

• SUCTION VALVE UNLOADERS

There are an unloaders existed on suction valves of the compressor and pneumatically operated

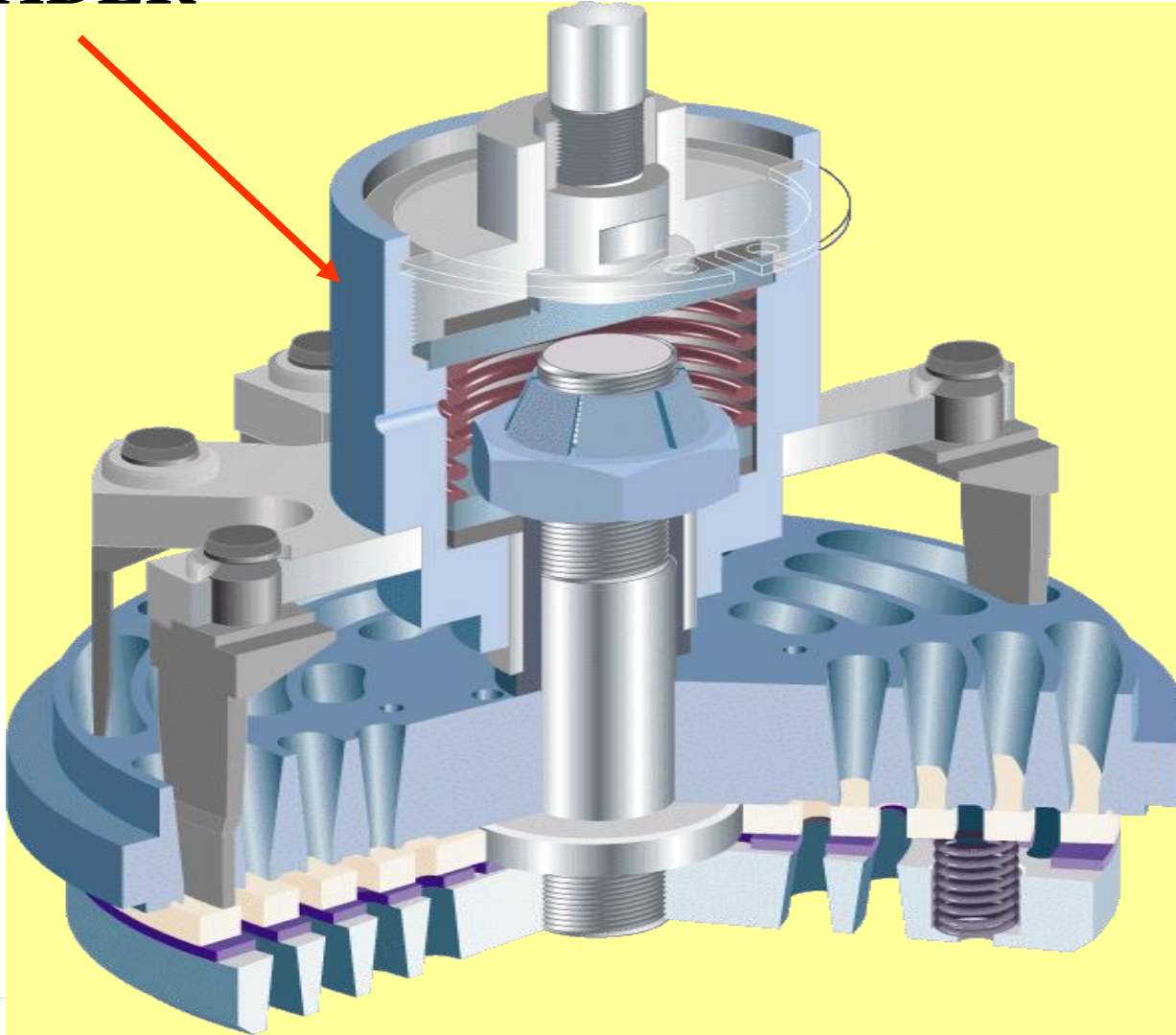
If the unloaders energized, the suction valves will be opened.

If there are 4 suction valves and the unloaders are energized for :

- 1- One suction valve (Capacity will be $\frac{3}{4}$ max. Q)
- 2- Two suction valve (Capacity will be $\frac{1}{2}$ max. Q)
- 3- Three suction valve (Capacity will be $\frac{1}{4}$ max. Q)
- 4- Four suction valve (Capacity will be = 0 Q)

SUCTION VALVE

UNLOADER



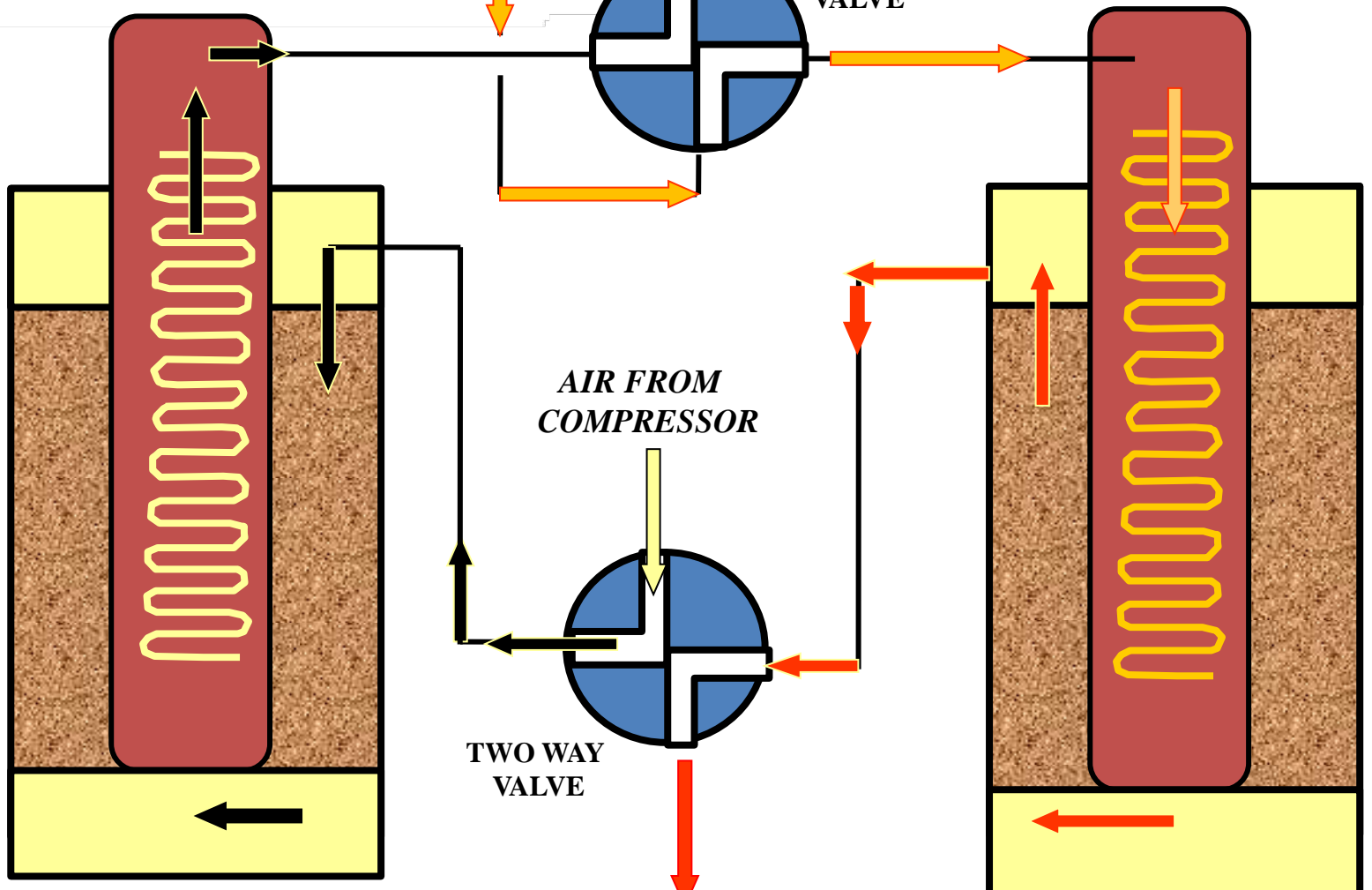


Air Compressor Dryer

<https://www.youtube.com/watch?v=6tVw9VYQDIg>

AIR COMPRESSORS DRYER

Dry Air to instrument Receiver



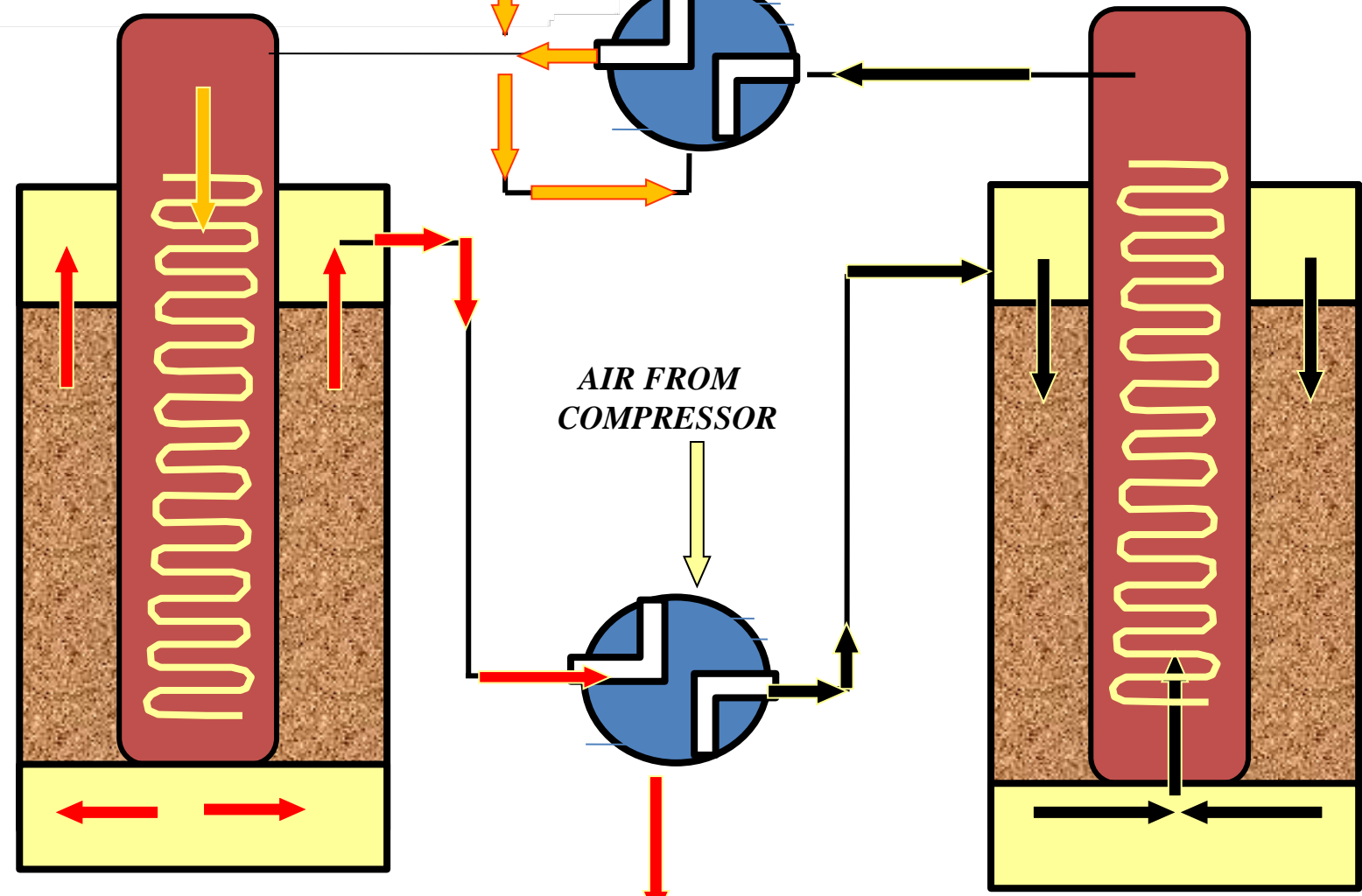
ACTIVATED

Wet Air to Atmosphere

REGENERATED

AIR COMPRESSORS DRYER

Dry Air to instrument Receiver



REGENERATED

Wet Air to Atmosphere

ACTIVATED



Reciprocating Compressor Operations

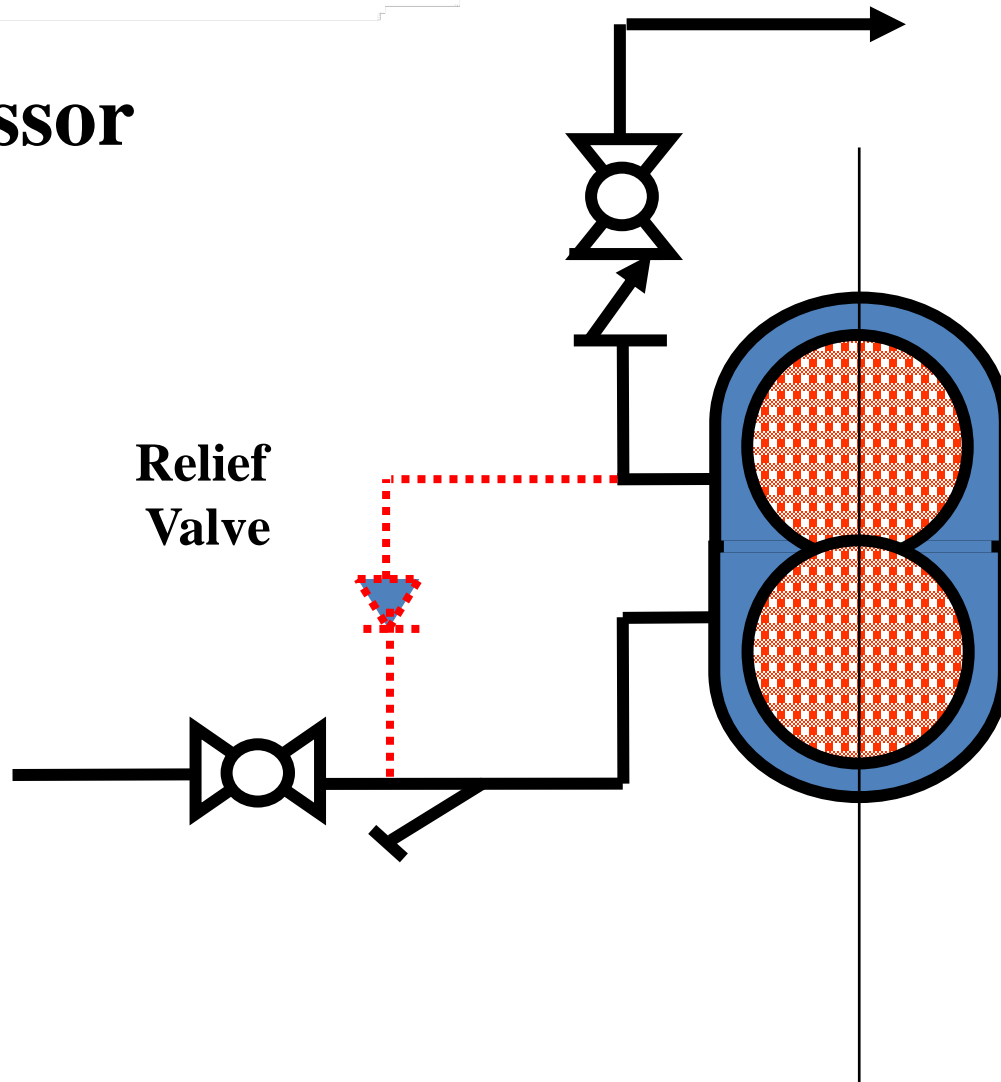
<https://www.youtube.com/watch?v=ooBJMI0H7PQ>



Discussion

***Rotary
Compressors***

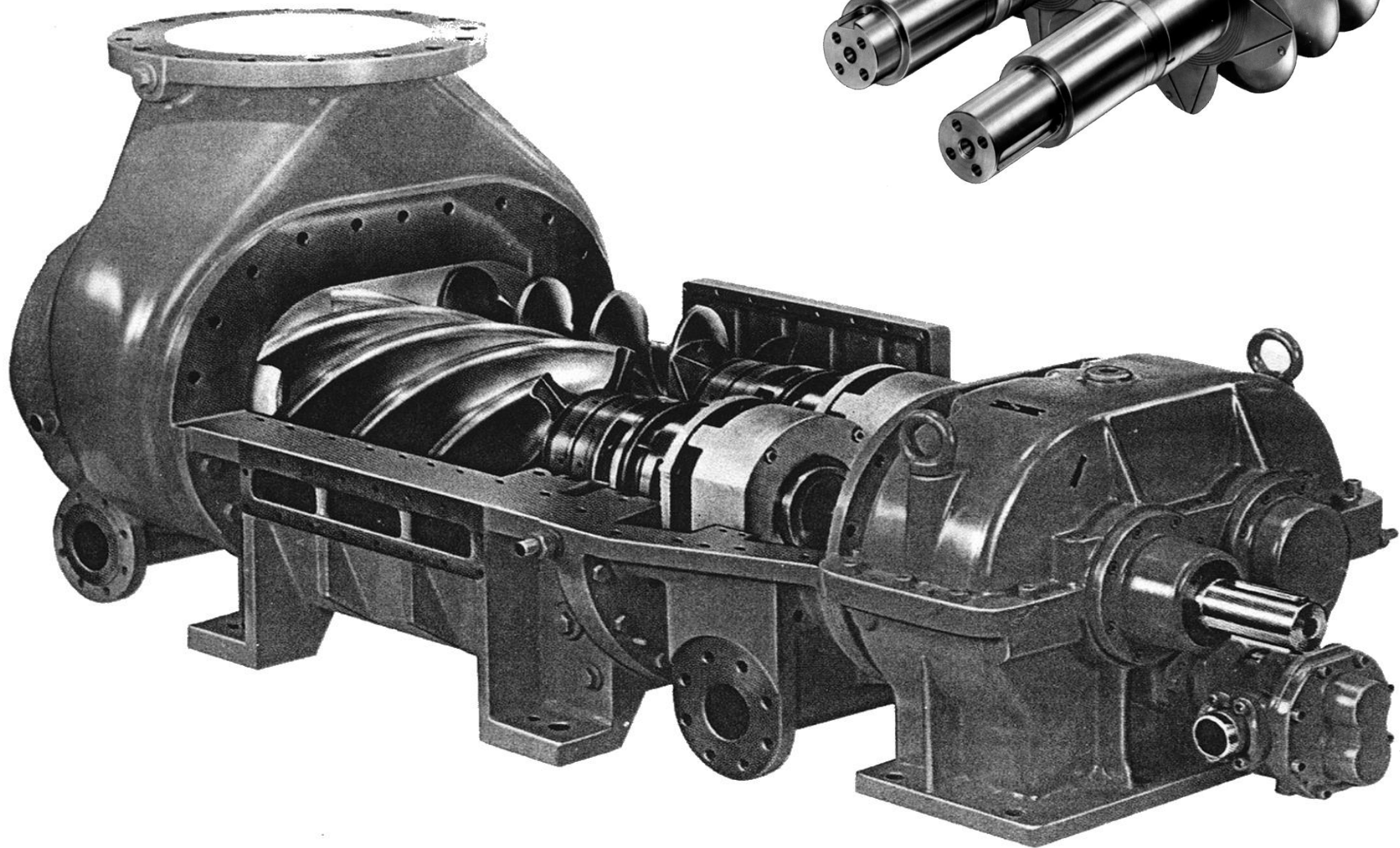
Rotary Compressor



Screw compressor

- **Dry Screw Compressor**
- **Oil Flooded Screw Compressor**

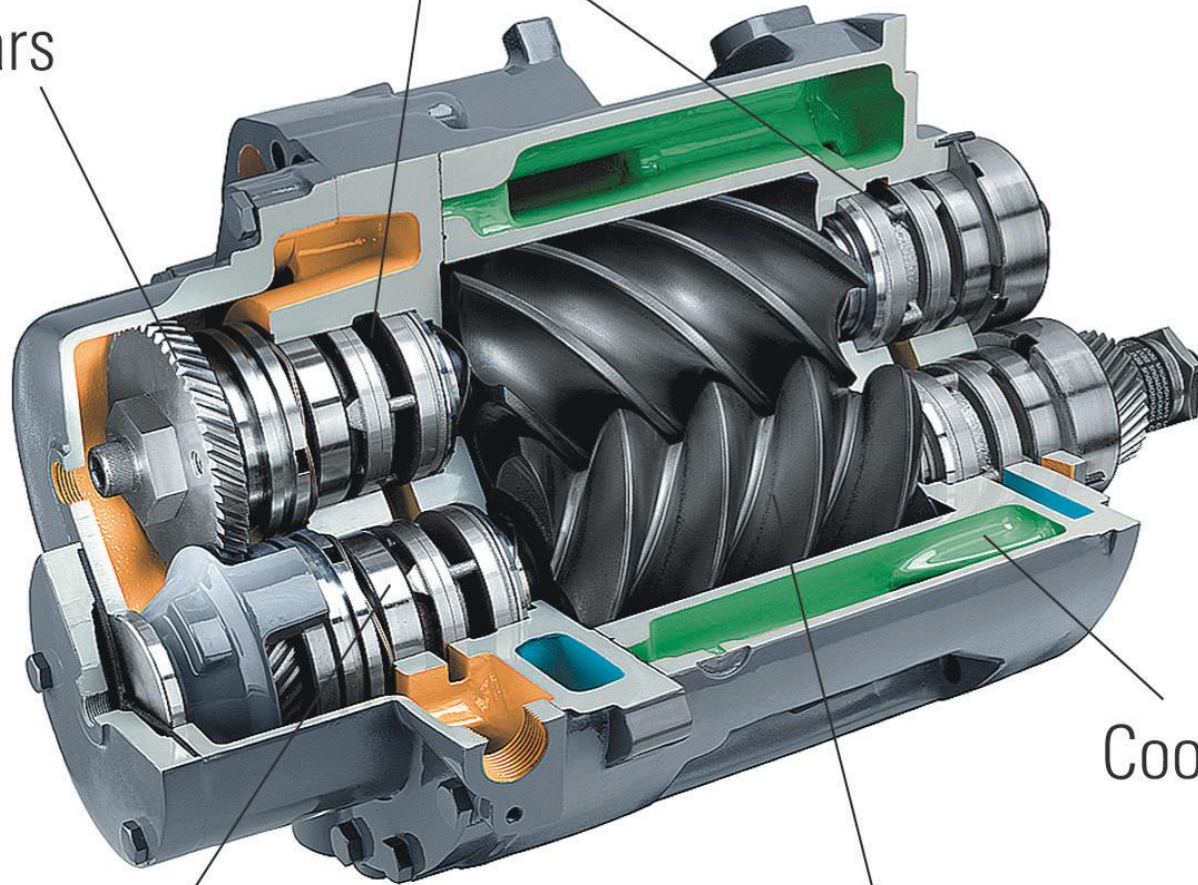
Rotary Screw Compressors



Rotary-Screw Compressor

Shaft seals

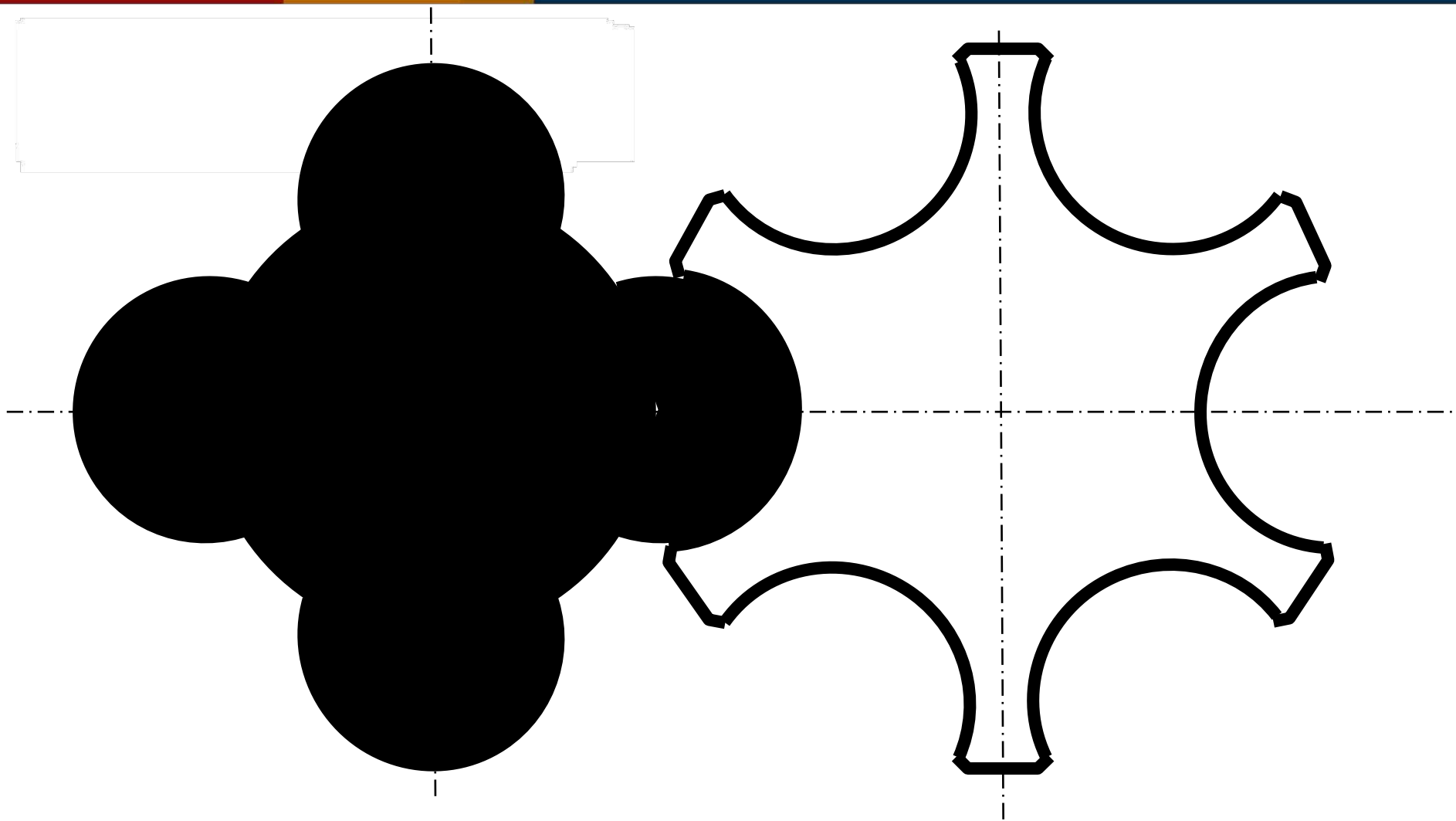
Timing gears



Cooling jackets

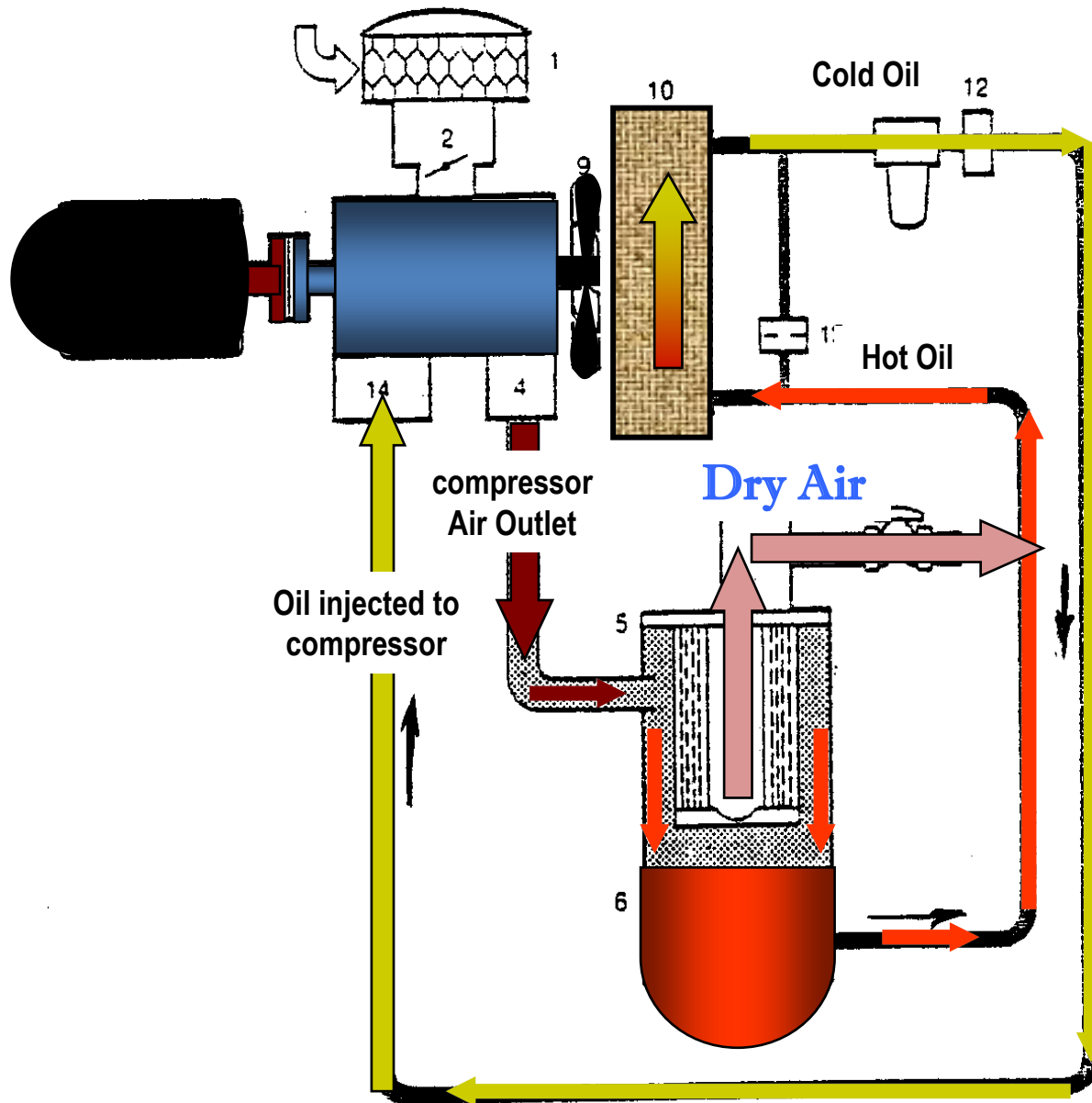
Asymmetric rotors

Anti-friction and
roller bearings

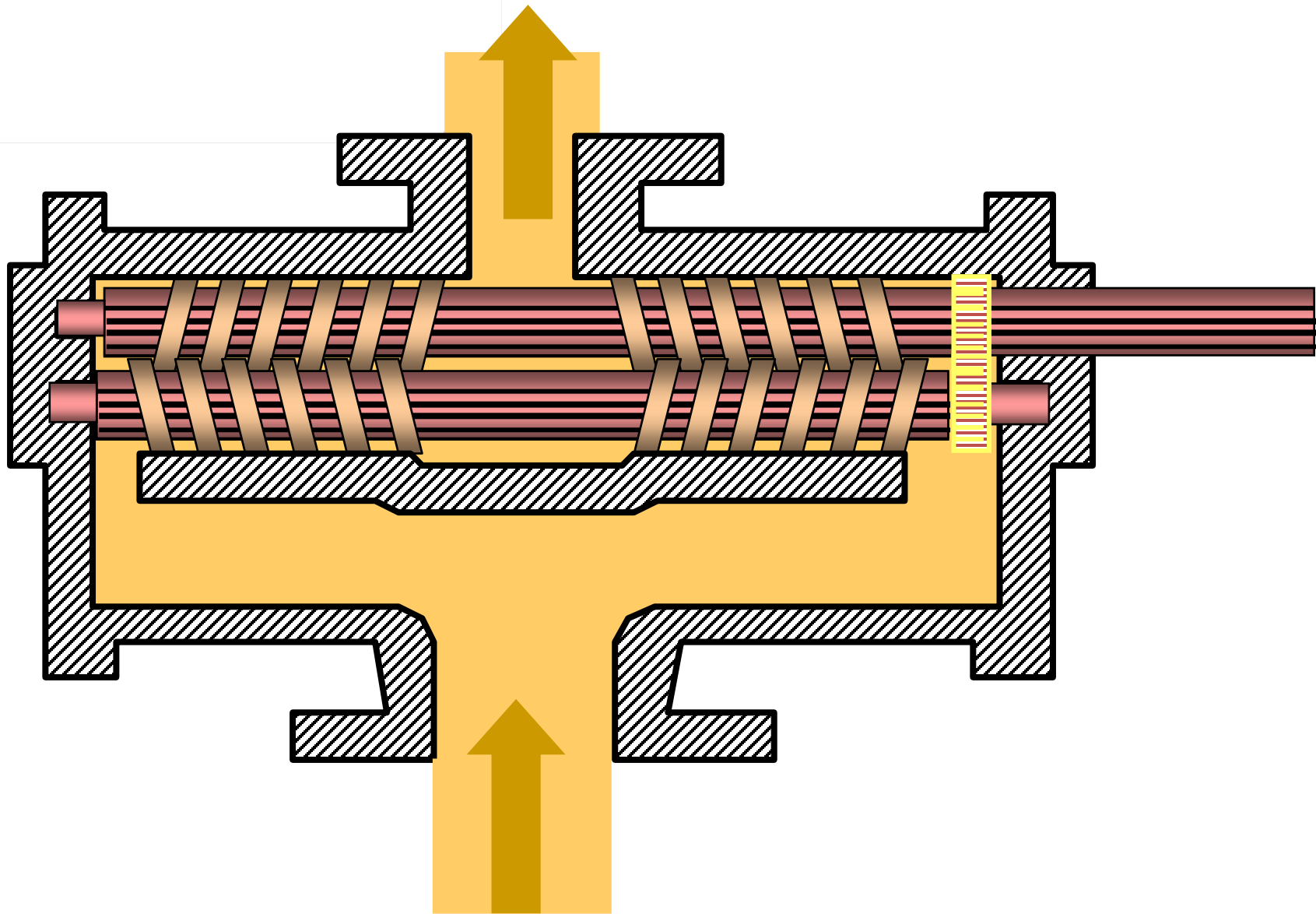


Rotor profile of the four -----lobe male and the six lobe female

Oil – flooding system for a rotary screw air compressor.



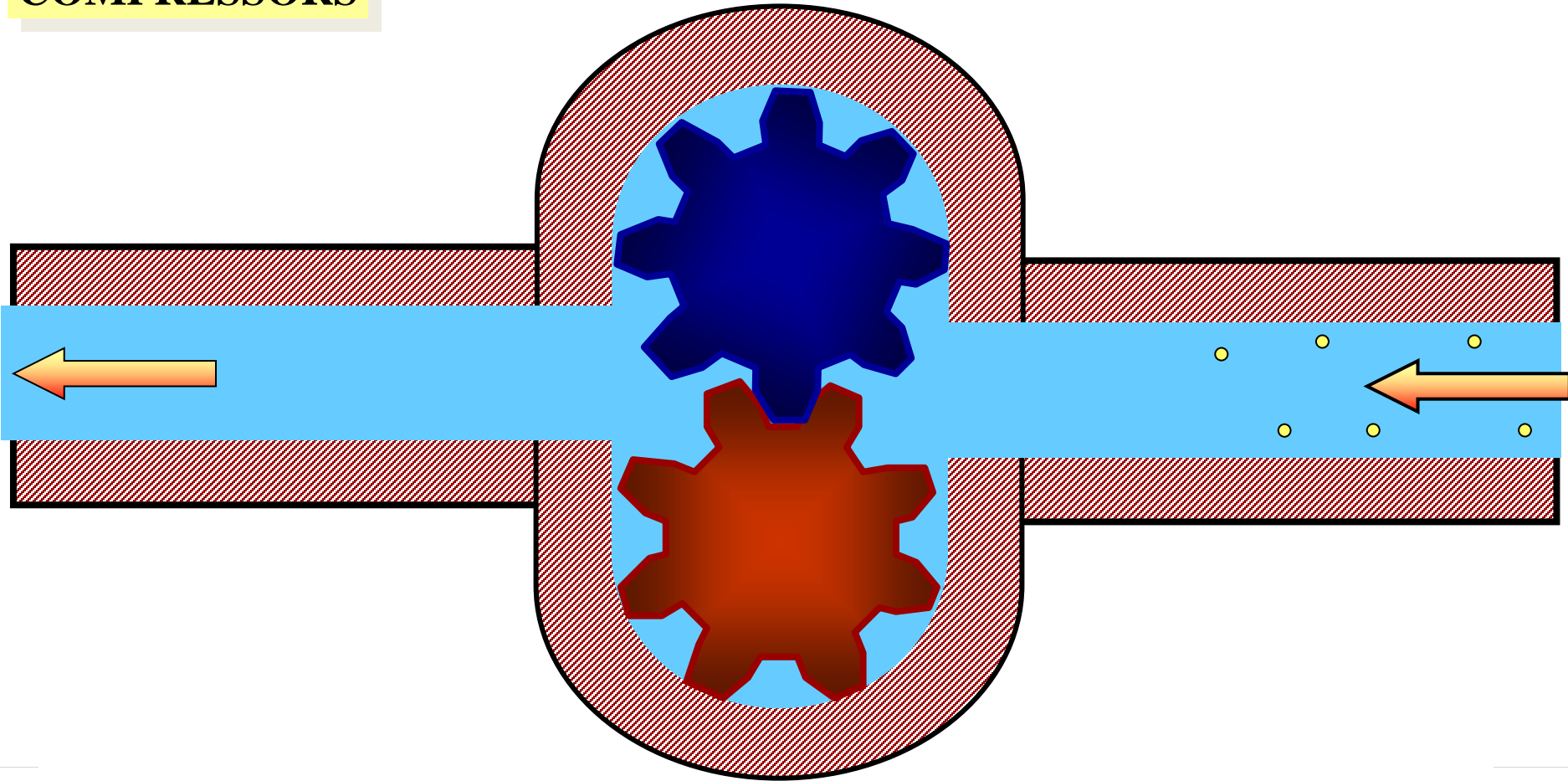
- 1 – air filter
- 2 – inlet throttle
- 3 – compressor element
- 4 – non-return valve
- 5 – oil separator
- 6 – oil sump
- 7 – oil separating filter
- 8 – non-return valve
- 9 – cooling fan
- 10 – oil cooler
- 11 – oil filter
- 12 – throttling
- 13 – thermostatic valve
- 14 – non-return valve



Multiple-screw double-end arrangement

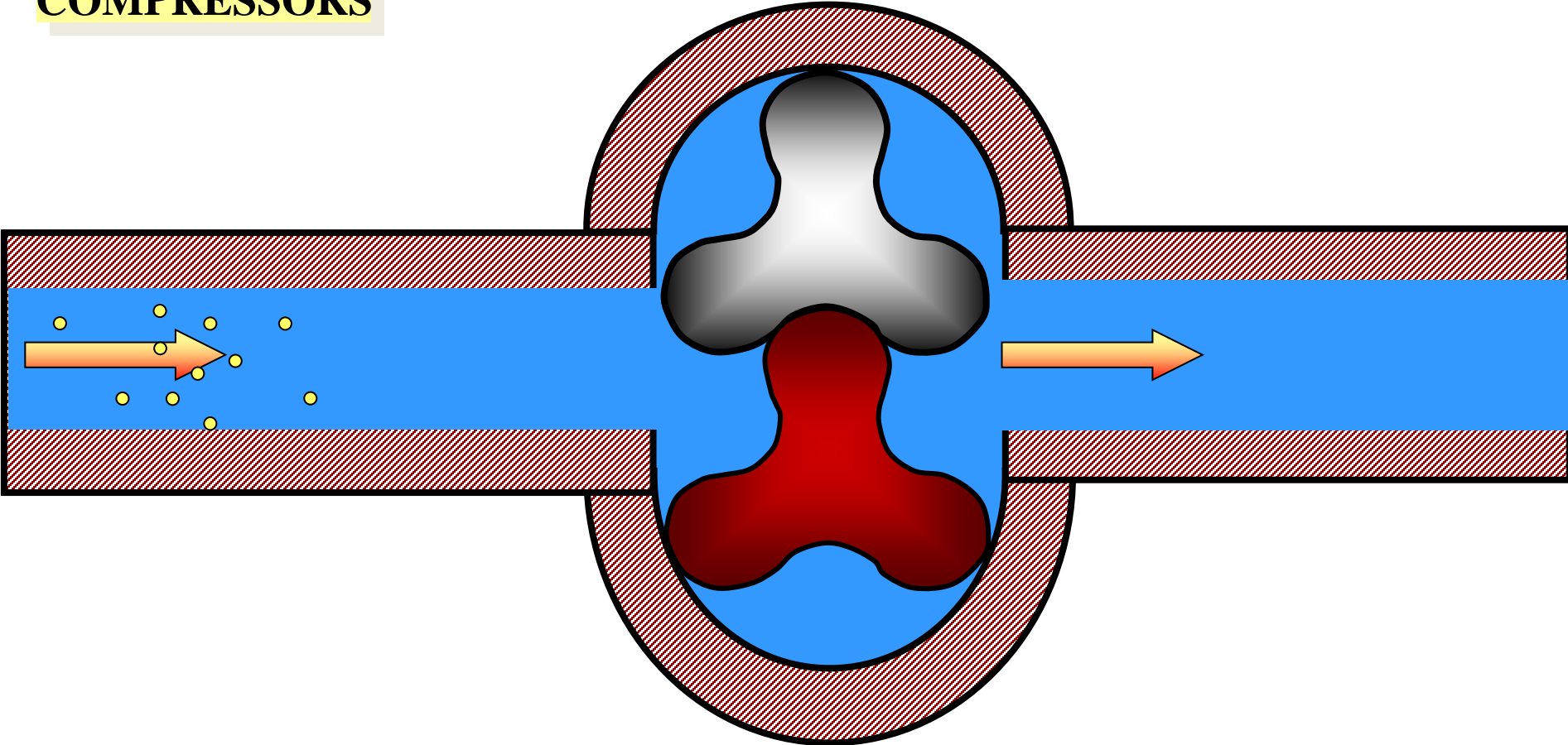
**ROTARY
COMPRESSORS**

External Gear

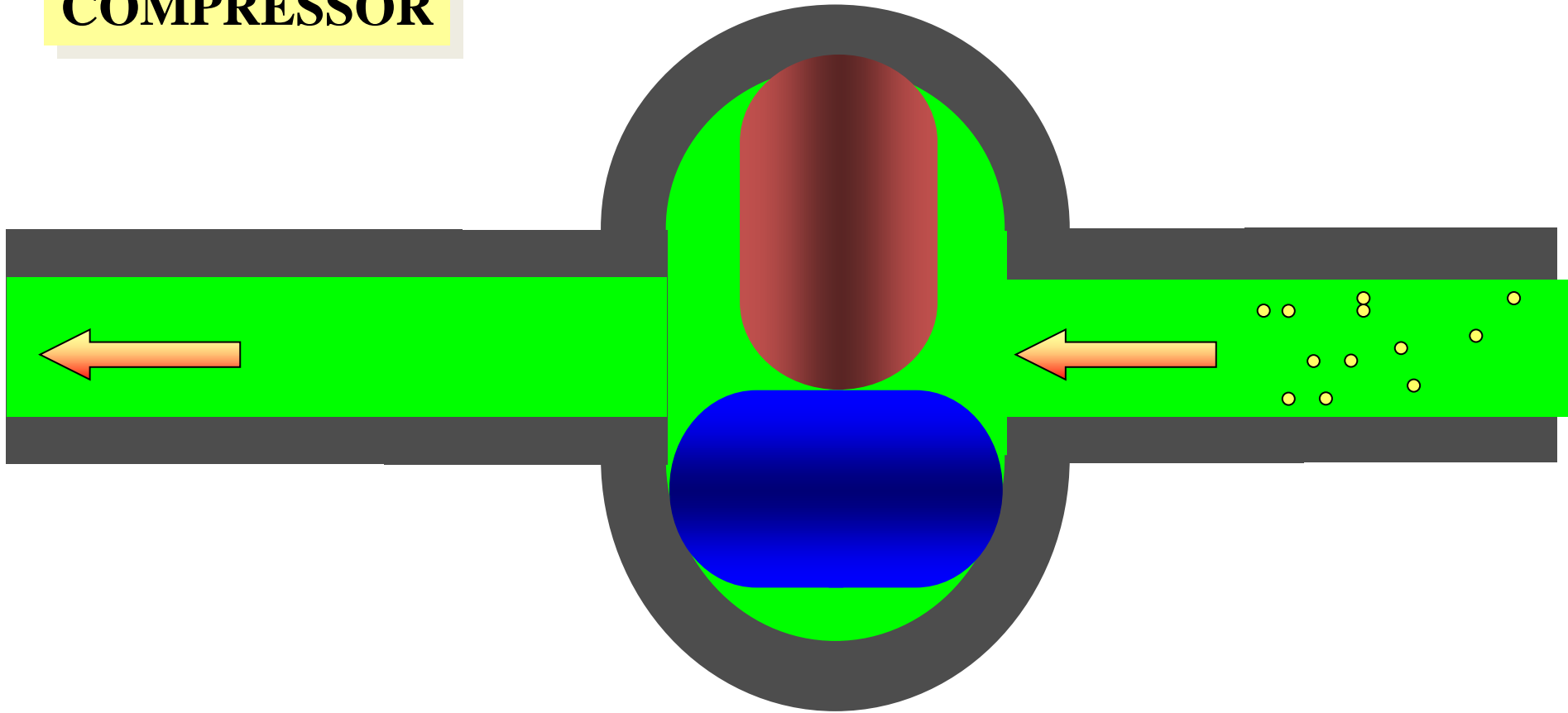


THE FLUID IS TRAPPED BETWEEN ROTOR AND CASING

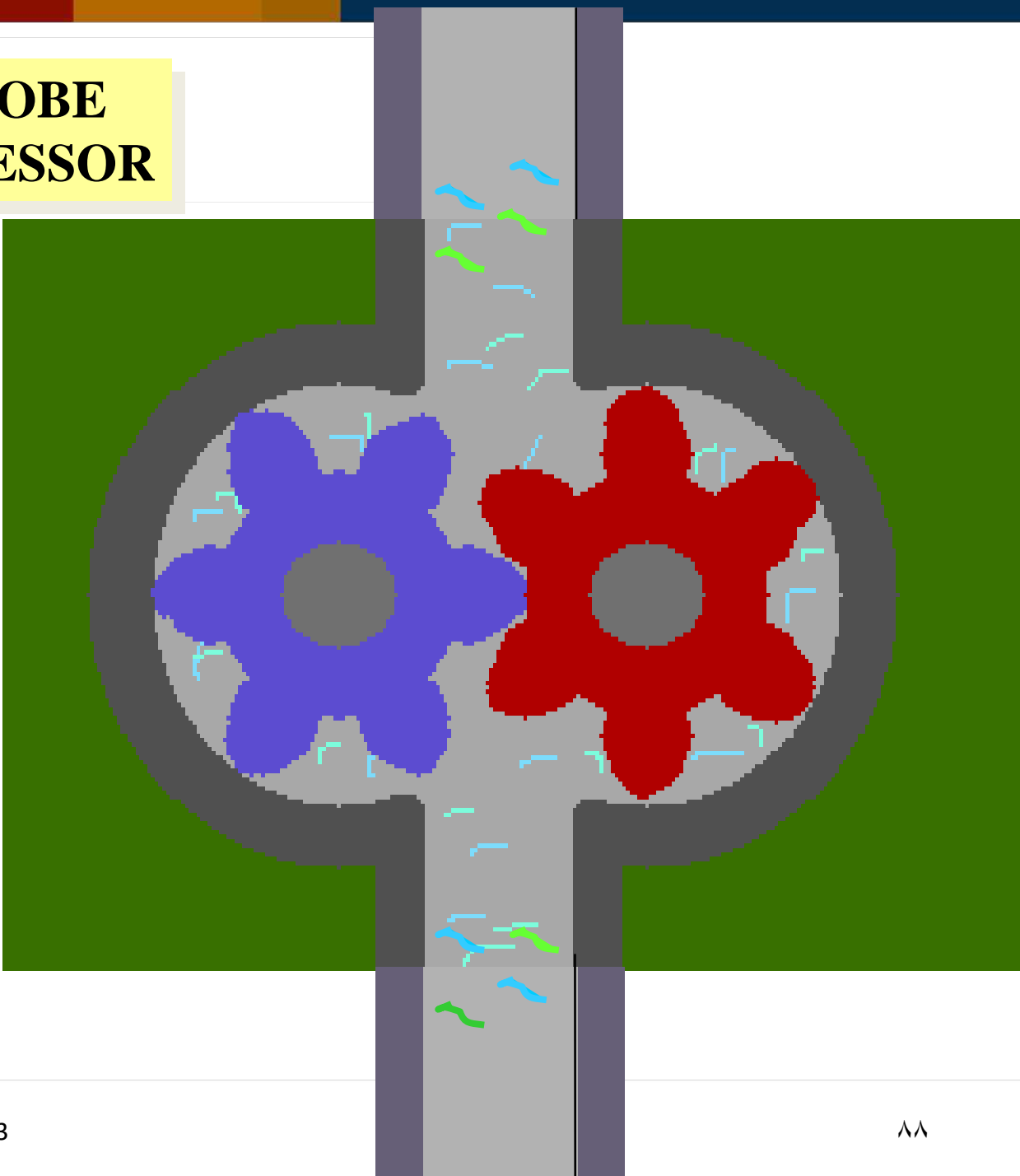
THREE LOBE COMPRESSORS



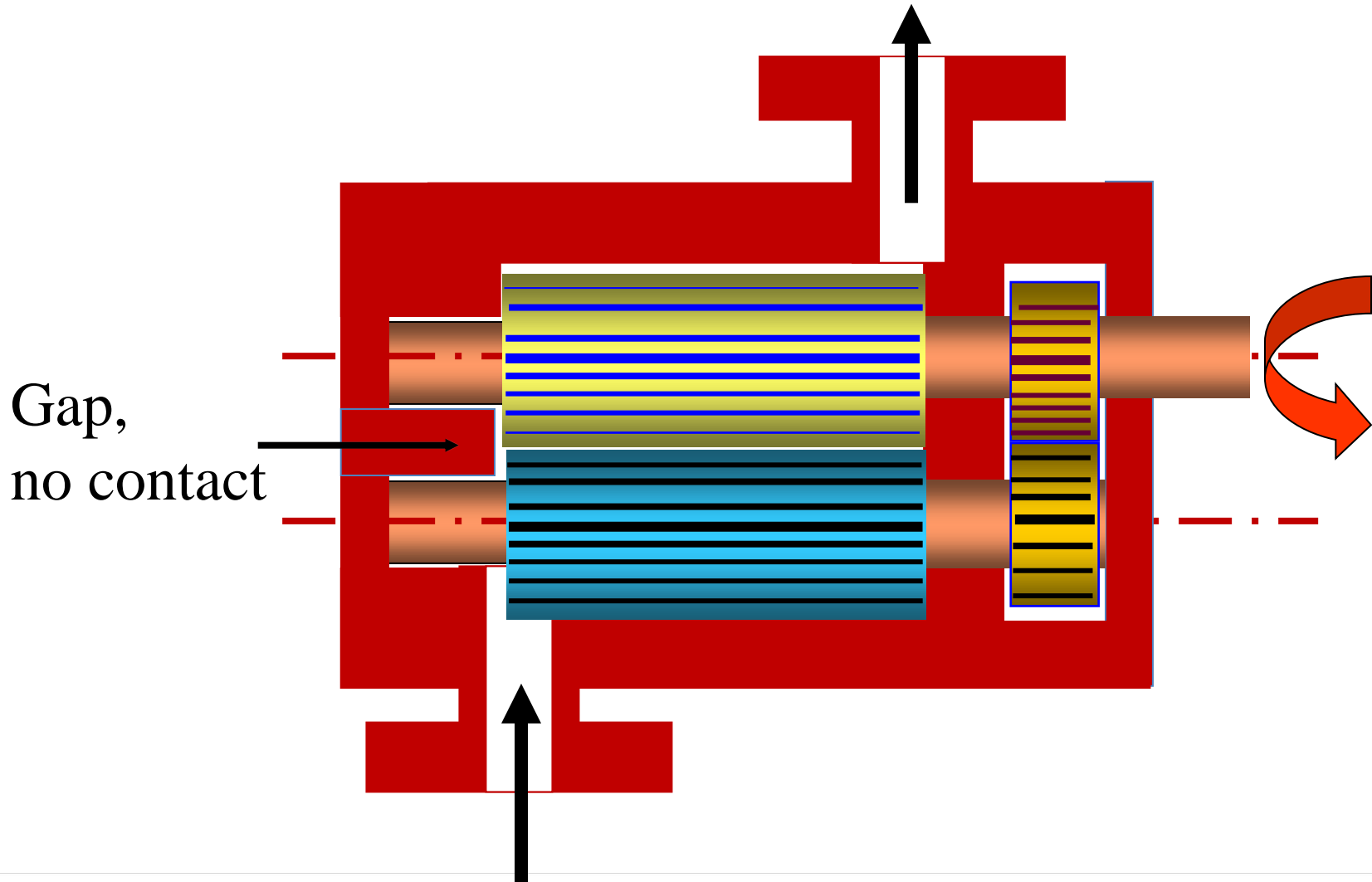
TWO LOBE COMPRESSOR



TWO LOBE COMPRESSOR



Dry helical screw compressors. (driven by external timing gears).

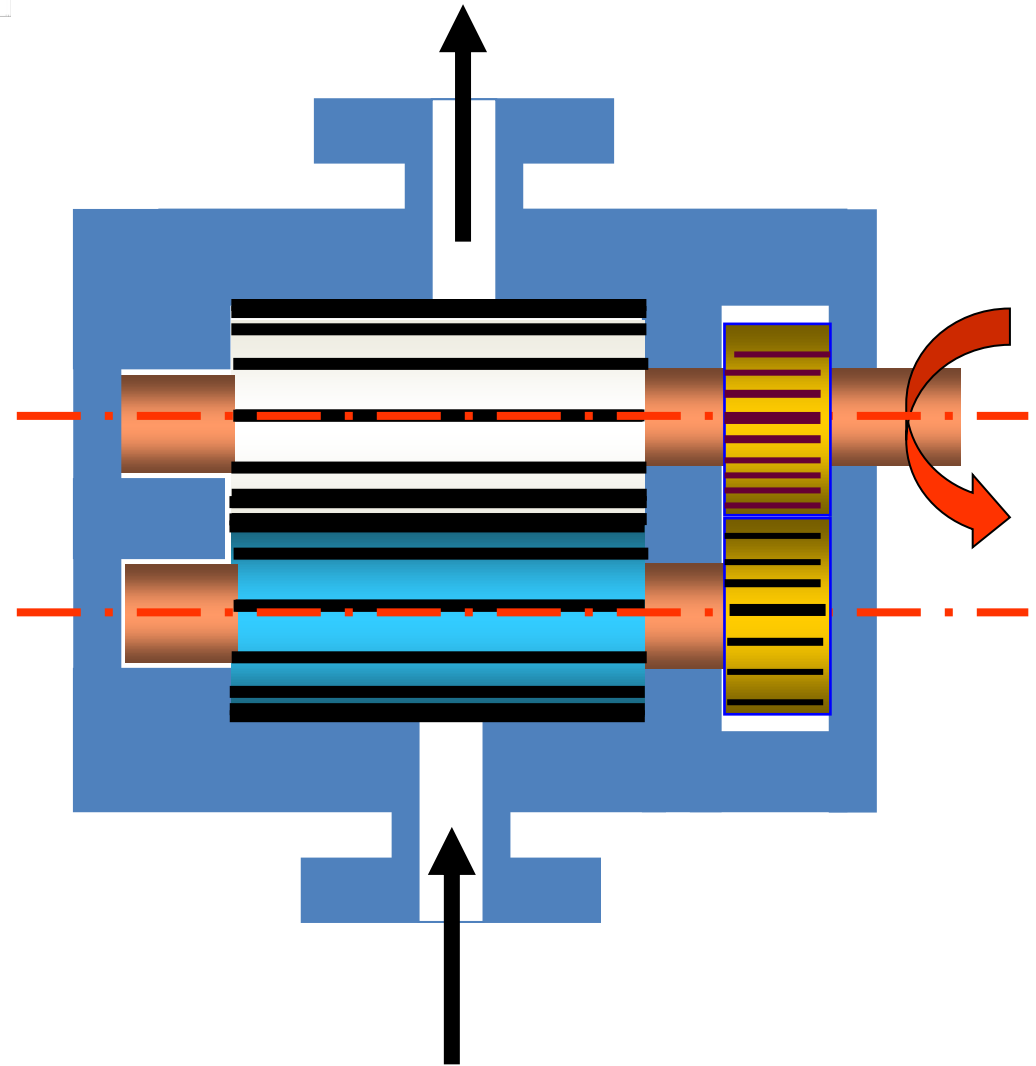


TIMING GEAR FUNCTION

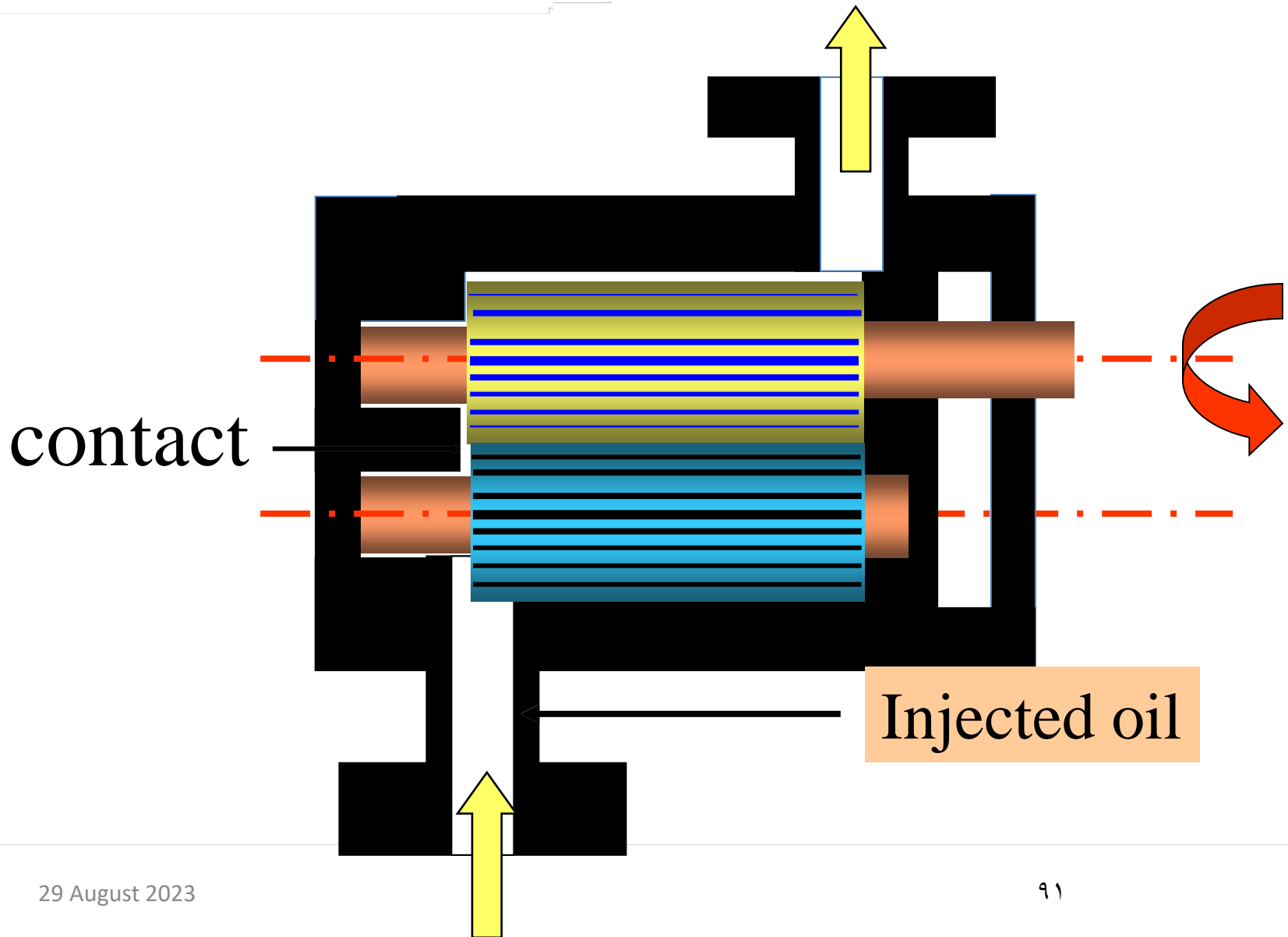
1- Transmit Motion To Other Rotor

2- Keeps No Contact Between Rotors

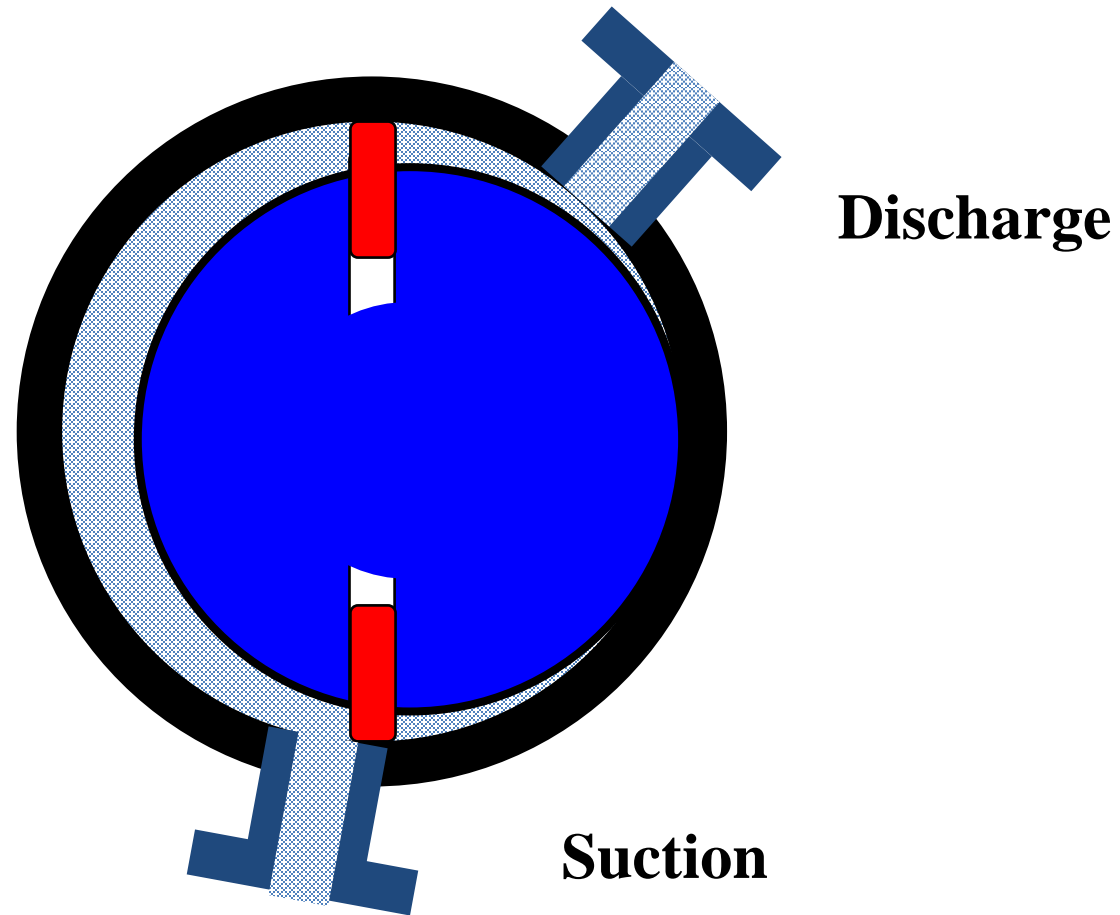
3- Prevent Wear Between Rotors



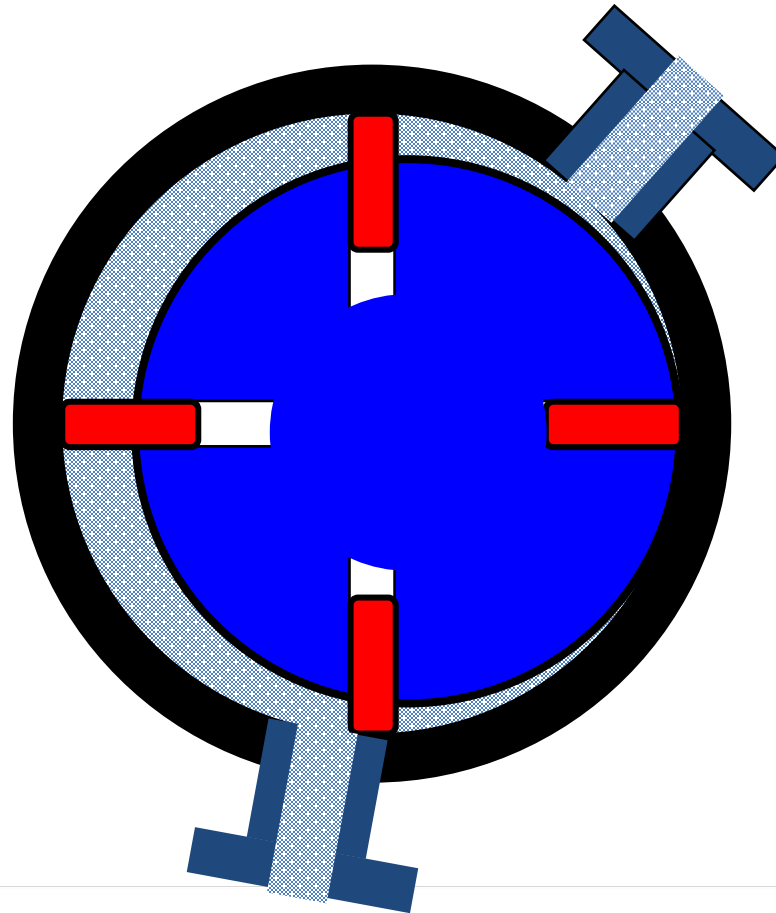
Rotary helical screw oil injected compressor



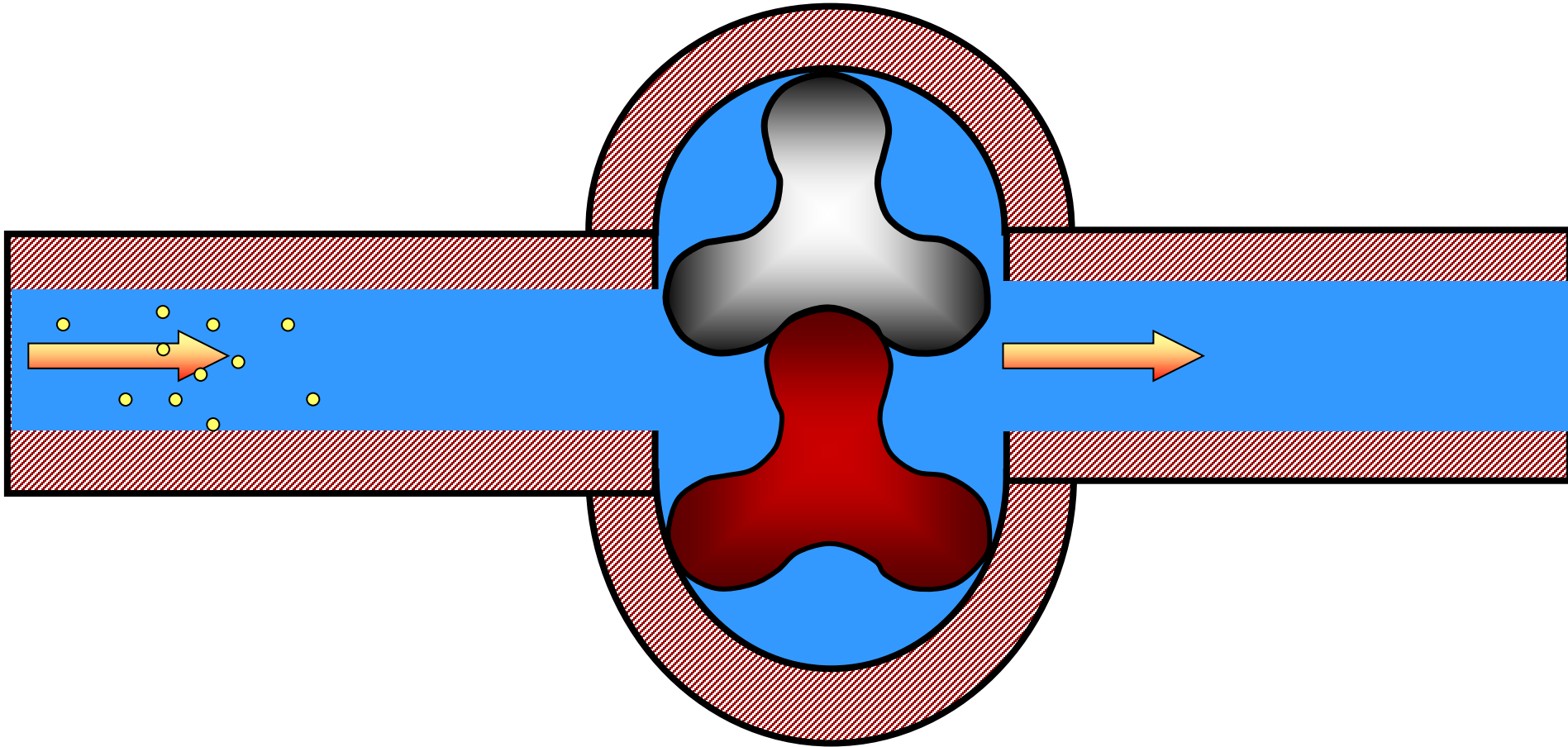
Rotary Vane Compressor



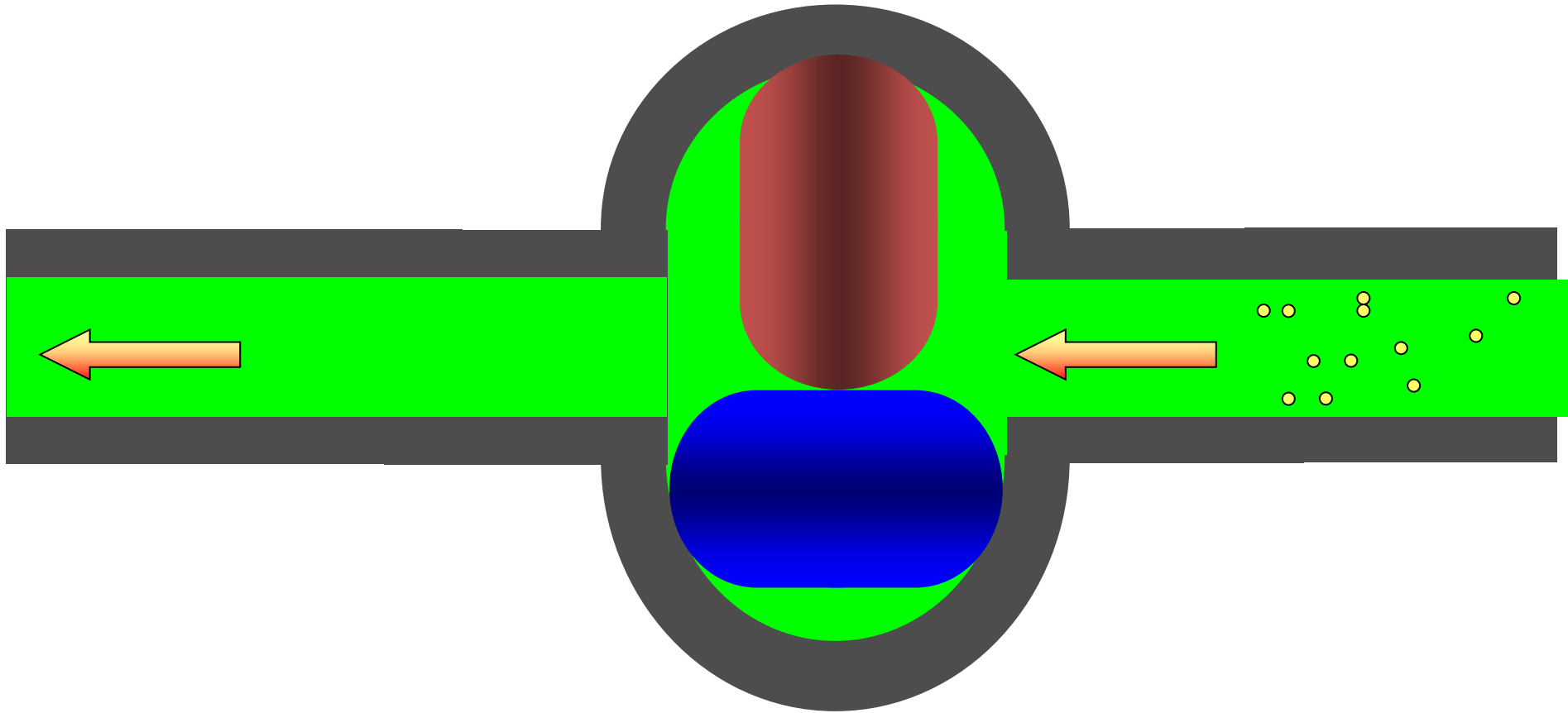
Rotary Vane Gas Compressors



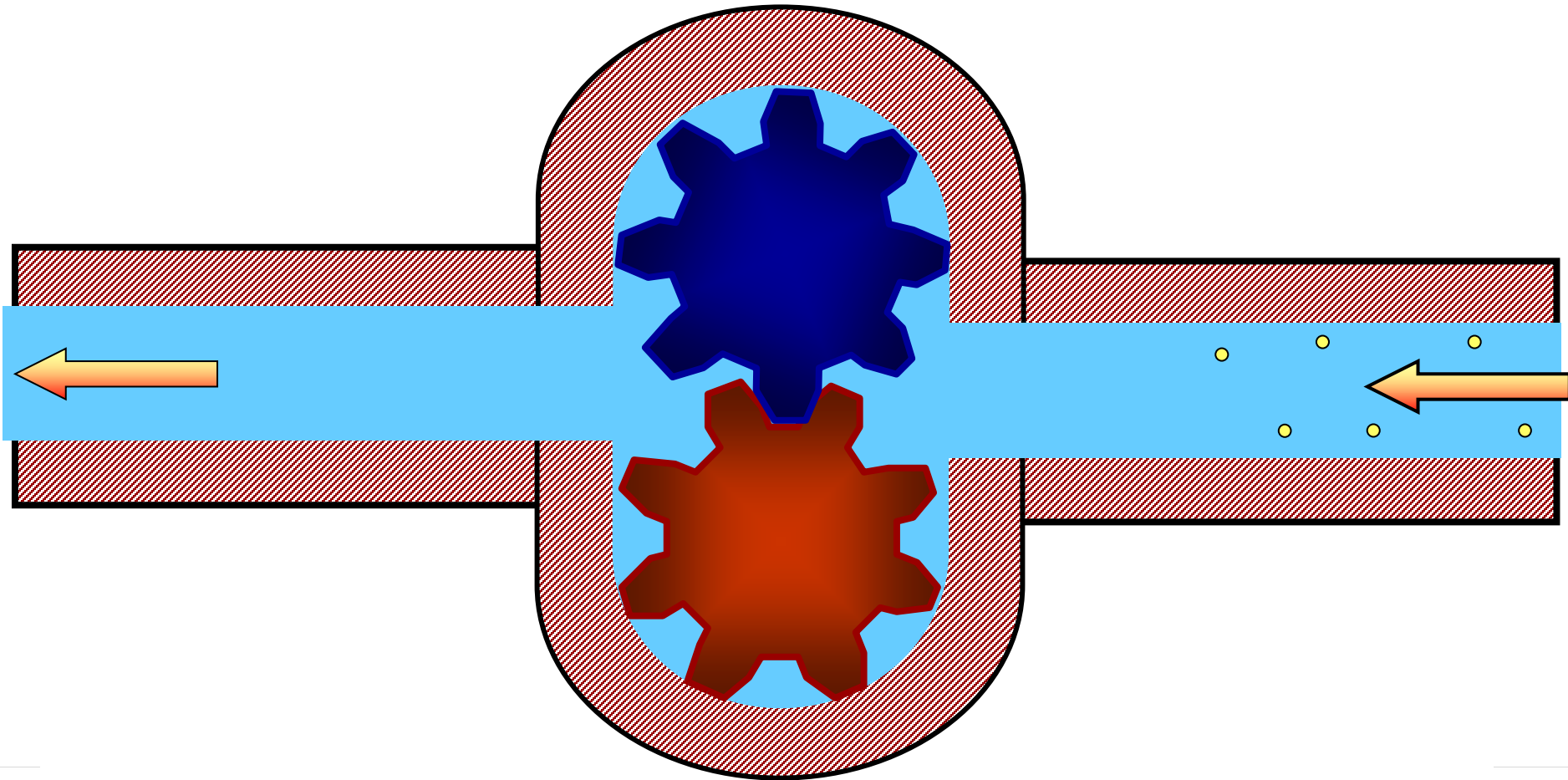
THREE LOBE COMPRESSORS



TWO LOBE COMPRESSOR



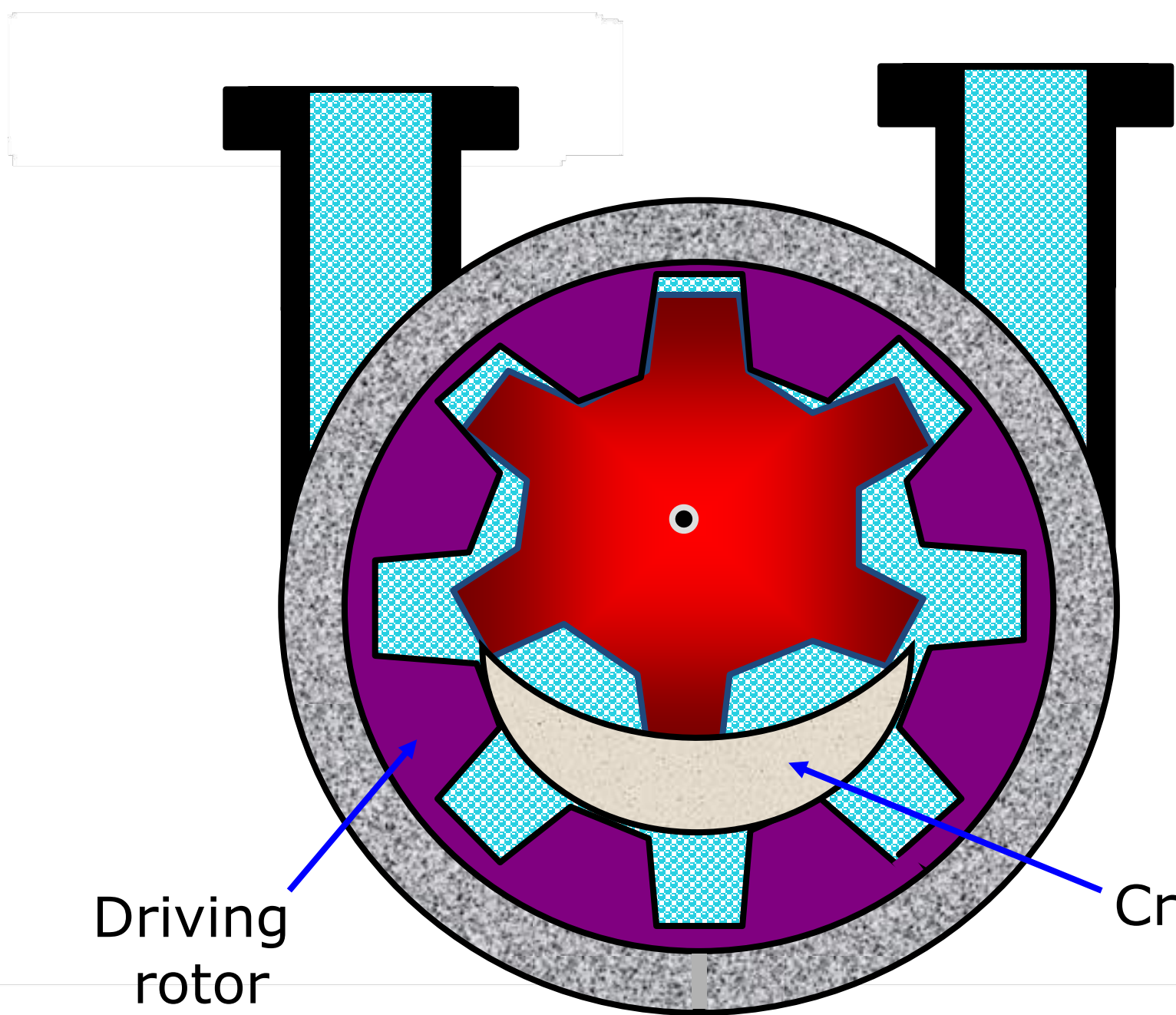
External Gear



THE FLUID IS TRAPPED BETWEEN ROTOR AND CASING

Internal Gear





Driving
rotor

Crest